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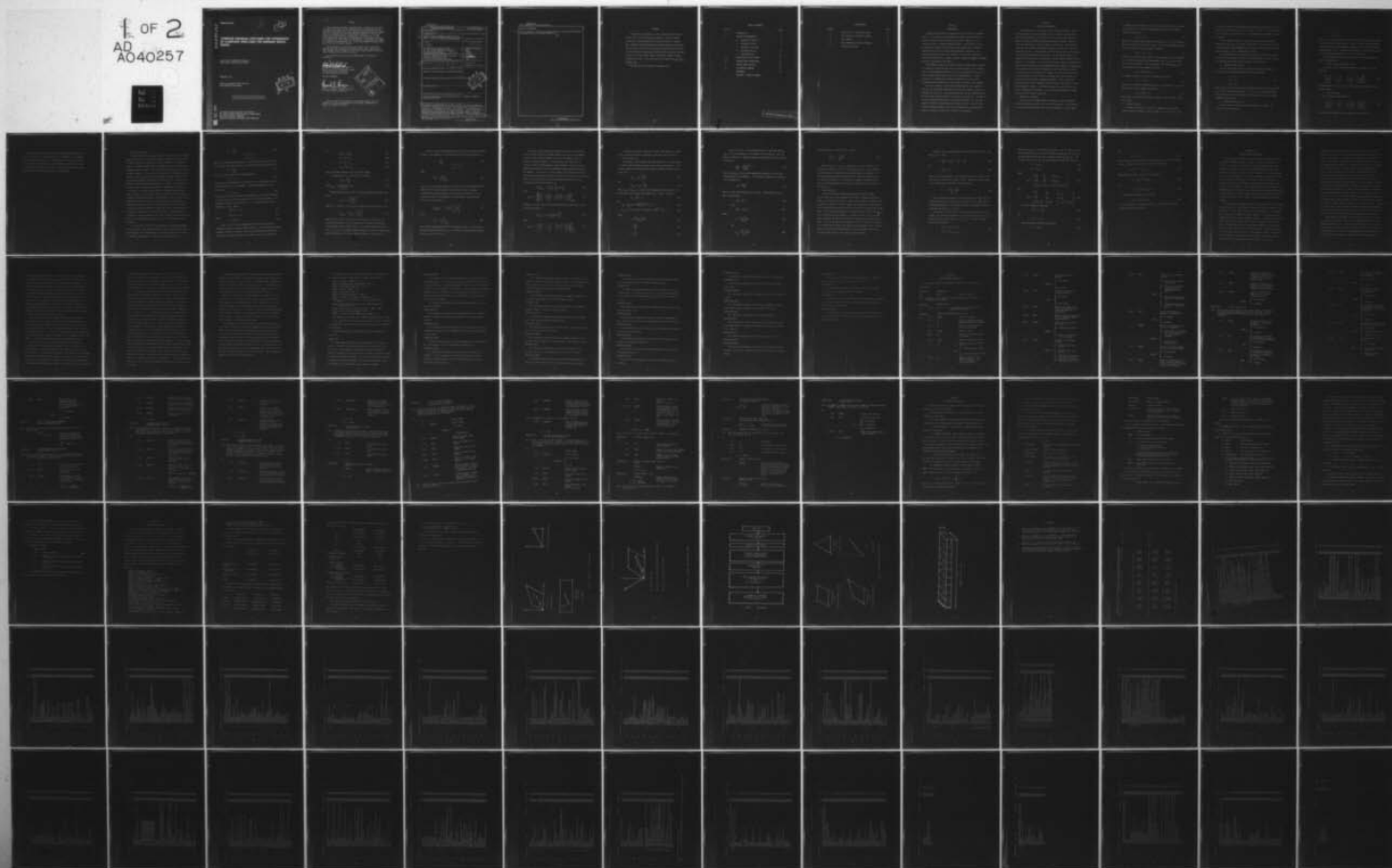
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COMPUTER PROGRAM (OPTCOMP) FOR OPTIMIZATION OF COMPOSITE STRUCTURES FOR MINIMUM WEIGHT DESIGN

*ANALYSIS & OPTIMIZATION BRANCH
STRUCTURAL MECHANICS DIVISION*

FEBRUARY 1977

TECHNICAL REPORT AFFDL-TR-76-149
FINAL REPORT FOR PERIOD

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This technical report has been reviewed and is approved for publication.

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sizes if desired. A mixture of composite and metal structure can be designed by suitable definition of material properties.



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FOREWORD

This report is prepared as a part of in-house effort under Project 1467, "Structural Analysis Methods", Task No. 146702, "Structural Analysis Methods for Aerospace Vehicles", and Work Unit 14670246, "Automated Design of Advanced Aerospace Structures". The work was carried out in the Design and Analysis Methods Group of the Analysis and Optimization Branch (FBR), Structural Mechanics Division, Air Force Flight Dynamics Laboratory (AFFDL), Wright-Patterson AFB, Ohio. The time period of the effort was June 1973 - October 1976.

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SECTION I

INTRODUCTION

Advanced fiber reinforced composites such as boron-epoxy, graphite-epoxy, etc. are being used increasingly in flight vehicle structures. In order to make the most efficient use of these composites, it is necessary to have a reliable design procedure. In References 1 and 2, an optimization method was presented for the minimum weight design of composite structures with stress and displacement constraints. The concepts developed in these two references for the stress constraint design form the basis of a computer program to Optimize Composite (OPTCOMP) structures discussed in this report.

The OPTCOMP program can be used to design a minimum weight structure or to analyze a given structure which is idealized with finite elements. This is an experimental program and as such it includes different recurrence relations and strength criteria for experimentation. A metal structure or a combination of metal and composite structure can be designed by proper definition of the elastic properties of the elements.

The basic relations of the finite element method, the optimality criteria, the recurrence relations and the various strength criteria are discussed in Section II. The buckling equations and the transformations for the elastic constant matrix are also given in Section II. Section III contains a general description of the computer program. The input and output instructions are included in Sections IV and V. Section VI discusses the sample problem. The Fortran listing of the program and the output of the sample problem are presented in an Appendix.

SECTION II

ANALYTICAL FORMULATION

The optimization of a structure is an iterative procedure. The two main steps involved in the algorithm are 1) to analyze the structure to determine the response of the structure to the applied loads and 2) to modify the design variables by using a recurrence relation. These two steps are repeated for a specific number of iterations. The recurrence relations used to modify the design variables are based on the selected optimality criteria. The optimality criteria used to derive the recurrence relation may be theoretical or intuitive. In the OPTCOMP program, the response of the structure is obtained by a finite element analysis of the structure. The geometry of the structure is fixed. Hence, the design variables are the thicknesses of the homogeneous plate elements, the thicknesses of the layers in the composite elements, and the cross-sectional areas of the bars. As used throughout this report, the word "layer" identifies the total quantity of all the plies or laminae which are oriented in a specified direction within a composite laminate. Since out-of-plane bending effects are not considered in this program, the concept of layers may be applied. However, in actual laminates, these plies will be dispersed throughout the thickness. In the composite elements, the fiber directions to be used are specified.

1. Equations of Analysis

Since the finite element method of analysis is used in OPTCOMP, the basic equations are given here for ready reference. In this method the structure is discretized into a group of elements. Let the number

of elements in the structure be m and the number of layers in the composite elements be n . The force displacement relation for the whole structure is given by

$$[K] \{r\} = \{R\} \quad (1)$$

where $[K]$ is the stiffness matrix of the structure, $\{r\}$ is the displacement vector and $\{R\}$ is the force vector. The force displacement relation for the j th layer of the i th element is given by

$$[k]_{ij} \{r\}_i = \{f\}_{ij} \quad (2)$$

where $[k]_{ij}$ is the stiffness matrix of the j th layer of the i th element, $\{r\}_i$ are the i th element nodal displacements and $\{f\}_{ij}$ are the nodal forces of the j th layer of the i th element. The strain energy e_{ij} of the j th layer of the i th element is given by

$$e_{ij} = \{r\}_i^t [k]_{ij} \{r\}_i \quad (3)$$

The element strains $\{\epsilon\}_i$ in the i th element are given by

$$\{\epsilon\}_i = [c] \{r\}_i \quad (4)$$

where $[c]$ is the strain-displacement matrix. The stresses $\{\sigma\}_{ij}$ in the j th layer of the i th element are given by

$$\{\sigma\}_{ij} = [A]_{ij} \{\epsilon\}_i \quad (5)$$

where $[A]_{ij}$ is the matrix of elastic constants of the j th layer of the i th element.

2. Strength Criteria

The state of stress in a layer of an element has to satisfy a certain specified strength criteria in order to avoid failure. There

are several strength criteria proposed by different authors (For a detailed discussion, see Reference 3.). Most of these are explicit or implicit functions of the stresses or strains and the corresponding maximum allowable stresses or strains.

For an orthotropic material, there are basically five allowables required for each layer of a plane stress problem. These are the tension and compression maximum allowables in the fiber direction and in the transverse direction, and the maximum shear allowable.

In the OPTCOMP computer program four strength criteria are included. The user of the program can select the one he prefers. The four criteria for the generalized plane stress condition are given below.

a. Maximum Stress Criteria

This criteria states that the failure of the material would occur if any one of the following conditions are not satisfied.

$$\begin{aligned}\sigma_1 &\leq \tilde{\sigma}_1 \\ \sigma_2 &\leq \tilde{\sigma}_2 \\ \sigma_{12} &\leq \tilde{\sigma}_{12}\end{aligned}\tag{6}$$

where σ_1 and σ_2 are the stresses in the fiber direction and the one normal to it and σ_{12} is the shear stress. $\tilde{\sigma}_1$ and $\tilde{\sigma}_2$ are the maximum allowable stresses in the fiber and transverse directions, and $\tilde{\sigma}_{12}$ is the maximum allowable shear stress.

b. Maximum Strain Criteria

In this criteria strains instead of stresses are checked. The criteria can be expressed as

$$\begin{aligned}
\epsilon_1 &\leq \bar{\epsilon}_1 \\
\epsilon_2 &\leq \bar{\epsilon}_2 \\
\epsilon_{12} &\leq \bar{\epsilon}_{12}
\end{aligned}
\tag{7}$$

where ϵ_1 and ϵ_2 are the strains in the fiber direction and the one normal to it and ϵ_{12} is the shear strain. $\bar{\epsilon}_1$ and $\bar{\epsilon}_2$ are the maximum allowable strains in the fiber and transverse directions, and $\bar{\epsilon}_{12}$ is the maximum allowable shear strain. The layer is considered to have failed if any one of the conditions specified in Eq. 7 is not satisfied.

In the maximum stress and strain criteria the individual failure modes are independent.

c. Hill's Criteria Modified by Tsai

According to this criteria the material is assumed not to have failed if

$$\left[\left(\frac{\sigma_1}{\bar{\sigma}_1} \right)^2 + \left(\frac{\sigma_2}{\bar{\sigma}_2} \right)^2 - \left(\frac{\sigma_1 \sigma_2}{\bar{\sigma}_1^2 \bar{\sigma}_2} \right) - \left(\frac{\sigma_{12}}{\bar{\sigma}_{12}} \right)^2 \right] \leq 1 \tag{8}$$

The definitions of $\bar{\sigma}_1$, $\bar{\sigma}_2$, etc. are the same as in the maximum stress criteria above.

d. Norris Criteria

This criteria can be written as

$$\left[\left(\frac{\sigma_1}{\bar{\sigma}_1} \right)^2 + \left(\frac{\sigma_2}{\bar{\sigma}_2} \right)^2 - \left(\frac{\sigma_1 \sigma_2}{\bar{\sigma}_1 \bar{\sigma}_2} \right) - \left(\frac{\sigma_{12}}{\bar{\sigma}_{12}} \right)^2 \right] \leq 1 \tag{9}$$

This criteria is similar to Eq. 8 except for the third term.

It should be pointed out that different versions of the above four criteria can be used by a proper choice of the allowables. For example, if the transverse stress is not to be considered critical as in the case of a fiber controlled laminate, $\bar{\sigma}_2$ can be assigned a high value so that σ_2 will not become critical. Any special type of criteria can be easily introduced into the program by writing a new subroutine.

3. Recurrence Relations

In this section the recurrence relations incorporated in the OPTCOMP program are given. For the stress constraint problem, when the maximum allowable stress is not the same in all the elements of a structure, there is no rigorous method based on optimality criteria for designing a minimum weight structure. This is particularly true when the structure has plate elements. If the structure consists of bar elements only, the problem can be correctly solved by replacing the stress constraints by equivalent displacement constraints and treating it as a multiple displacement constraint problem (Reference 2). However, this procedure of treating stress constraints by equivalent displacement constraints is not feasible in a structure where there are elements connecting more than two nodes. The recurrence relations proposed here for resizing the elements are based on optimality criteria which are not rigorous in the mathematical sense, but they are found to give near optimum weight designs for large structures in a very efficient way. The recurrence relations included in OPTCOMP are either based on the strain energy in an element or the state of stress in an element. Those depending on the state of stress correspond to the fully stressed iterative procedure. The user of the program may select any one of them depending on his preference.

The optimality criteria for the generalized stiffness requirement can be stated as, "the optimum structure is the one in which the ratio of the average strain energy density to the mass density is the same in all elements" (Reference 2). This criteria can be written as

$$1 = \lambda \frac{\bar{e}_{ij}}{\rho_{ij}} \quad (10)$$

$$i = 1, \dots, m$$

$$j = 1, \dots, n$$

where λ is the Lagrangian parameter, ρ_{ij} is the mass density of the j th layer of the i th element, and \bar{e}_{ij} is the strain energy density given by

$$\bar{e}_{ij} = \frac{e_{ij}}{V_{ij}} \quad (11)$$

Here V_{ij} is the volume of the element defined as

$$V_{ij} = t_{ij} l_i \quad (12)$$

where t_{ij} is the thickness of the j th layer of the i th element and l_i is the surface area of the i th element. The design variable t_{ij} can be written as

$$t_{ij} = \Lambda \alpha_{ij} \quad (13)$$

where α_{ij} is the relative thickness (normalized to the maximum thickness) of the j th layer of the i th element and Λ is the scaling parameter. For homogeneous elements in the structure $j = 1$. Introducing the scaling parameter Λ in Eq. 1 gives

$$\Lambda [K'] \{r\} = \{R\} \quad (14)$$

or

$$[K'] \{r'\} = \{R\} \quad (15)$$

where

$$\{r\} = \frac{1}{\Lambda} \{r'\} \quad (16)$$

In Eq. 15 $[K']$ is the stiffness matrix for the whole structure obtained by using the relative design vector α_{ij} . Introducing the scaling parameter into Eqs. 2 through 5, the relations between the actual quantities and the relative quantities at element level can be expressed

as

$$[k]_{ij} = \Lambda [k']_{ij} \quad (17)$$

$$\{r\}_i = \frac{1}{\Lambda} \{r'\}_i \quad (18)$$

$$\{\epsilon\}_i = \frac{1}{\Lambda} \{\epsilon'\}_i \quad (19)$$

$$\{\sigma\}_{ij} = \frac{1}{\Lambda} \{\sigma'\}_{ij} \quad (20)$$

where the primed quantities are the relative values.

Introducing Eqs. 11, 17 and 18 in Eq. 10 gives

$$1 = \frac{\lambda}{\Lambda^2} \frac{\bar{e}'_{ij}}{\rho_{ij}} \quad (21)$$

$$\text{where } \bar{e}'_{ij} = \frac{\{r'\}_i^t [k']_{ij} \{r'\}_i}{\alpha_{ij} l_i} \quad (22)$$

Multiplying both sides of Eq. 21 by α_{ij}^2 and taking the square root gives

$$\alpha_{ij} = B \alpha_{ij} \left(\frac{\bar{e}'_{ij}}{\rho_{ij}} \right)^{1/2} \quad (23)$$

where B is a constant. Eq. 23 can be rewritten in an iterative form as

$$(\alpha_{ij})_{v+1} = B (\alpha_{ij})_v \left[\frac{\bar{e}'_{ij}}{\rho_{ij}} \right]_v^{1/2} \quad (24)$$

where $v+1$ and v refer to the cycles of iteration. In Eq. 24, B is a constant and generally it is not essential to evaluate it, since only the normalized relative design vector α_{ij} is used to analyze the structure. In the case of multiple loading conditions, a sum of relative strain energy densities due to all the loading conditions is used in Eq. 24.

In order to take into consideration the effect of different allowable stresses, the optimality criteria defined by Eq. 10 may be modified as

$$l = \lambda \frac{\tilde{e}_{ij}}{\rho_{ij}} \quad (25)$$

$i = 1, \dots, m$
 $j = 1, \dots, n$

where

$$\tilde{e}_{ij} = \frac{e_{ij}}{(\tilde{\sigma}_1 \tilde{\epsilon}_1)_{ij}} \quad (26)$$

where $\tilde{\sigma}_1$ is the maximum allowable stress and $\tilde{\epsilon}_1$ is the maximum allowable strain in the fiber direction of the j th layer of the i th element.

Following the procedure similar to the one used to derive Eq. 24, the recurrence relation corresponding to the optimality criteria defined by Eq. 25 can be written as

$$(\alpha_{ij})_{v+1} = B(\alpha_{ij})_v \left[\frac{\tilde{e}'_{ij}}{\rho_{ij}} \right]_v^{1/2} \quad (27)$$

where

$$\tilde{e}'_{ij} = \frac{\tilde{e}_{ij}}{(\tilde{\sigma}_1 \tilde{\epsilon}_1)_{ij}} \quad (28)$$

In Eq. 28 the prime quantities are the relative values. In the case of multiple loading conditions \tilde{e}'_{ij} is calculated for each loading condition and the sum is used in Eq. 27.

The fully stressed iterative procedure is based on the intuition that a structure will have minimum weight if the stress in each member is equal to the maximum allowable stress for that member, under at least one loading condition. In this procedure the resizing of an element is done by multiplying the design variable by the ratio of the maximum actual stress to the corresponding maximum allowable stress for that member. In the case of a plate element since there will be three stresses (two normal and one shear), the effective stress ratio can be used. Hence, in the case of Hill's criteria modified by Tsai (Eq. 8) the recurrence relation can be written as

$$(\alpha_{ij})_{v+1} = (\alpha_{ij})_v \left[\sigma'_{\text{eff}}(\text{Ht}) \right]_v^{1/2} \quad (29)$$

where

$$\sigma'_{\text{eff}}(\text{Ht}) = \left[\left(\frac{\sigma'_1}{\sigma_1} \right)^2 + \left(\frac{\sigma'_2}{\sigma_2} \right)^2 - \left(\frac{\sigma'_1 \sigma'_2}{\sigma_1 \sigma_2} \right) + \left(\frac{\sigma_{12}}{\sigma_{12}} \right)^2 \right]_{\text{max}} \quad (30)$$

Similarly, to fully stress the design with Norris criteria (Eq. 9), the recurrence relation is

$$(\alpha_{ij})_{v+1} = (\alpha_{ij})_v \left[\sigma'_{\text{eff}}(\text{N}) \right]_v^{1/2} \quad (31)$$

where

$$\sigma'_{\text{eff}}(\text{N}) = \left[\left(\frac{\sigma'_1}{\sigma_1} \right)^2 + \left(\frac{\sigma'_2}{\sigma_2} \right)^2 - \left(\frac{\sigma'_1 \sigma'_2}{\sigma_1 \sigma_2} \right) + \left(\frac{\sigma_{12}}{\sigma_{12}} \right)^2 \right]_{\text{max}} \quad (32)$$

In using the recurrence relation to resize a bar element or a shear panel, only the appropriate component of the stress vector is used.

4. Buckling Equations

The equations used to modify the layer thickness of a plate element to prevent local buckling are given in this section (Reference 4). For a simply supported orthotropic plate where the length to width ratio is large ($>>1$),

$$N_{xcr} = K_x \frac{\pi^2 D_{11}}{w^2} \quad (33)$$

and

$$N_{xycr} = K_{xy} \frac{\pi^2 D_{11}}{w^2} \quad (34)$$

where N_{xcr} and N_{xycr} are the critical compressive load (parallel to 0° fiber) and critical shear load respectively. In Eqs. 33 and 34,

$$K_x = 2\sqrt{\beta} + \alpha \quad (35)$$

and

$$K_{xy} = \left(3.293 + \frac{2.047}{\tilde{\theta}} \right) \beta^{3/4} \text{ for } \tilde{\theta} \geq 1 \quad (36)$$

$$K_{xy} = (4.742 + 0.216 \tilde{\theta} + 0.380 \tilde{\theta}^2) \left(\frac{\alpha\beta}{2} \right)^{1/2} \text{ for } \tilde{\theta} < 1 \quad (37)$$

where

$$\alpha = \frac{2(D_{12} + 2D_{66})}{D_{11}} \quad (38)$$

$$\beta = \frac{D_{22}}{D_{11}} \quad (39)$$

$$\tilde{\theta} = \frac{2\beta^{1/2}}{\alpha} \quad (40)$$

In Eqs. 33 and 34, w is the effective width of the plate and D_{11} , D_{12} , etc. are the elements of the bending stiffness matrix. When the plate is subjected to combined compressive and shear load, the stability criterion is

$$\frac{n_x}{N_{xcr}} + \left(\frac{n_{xy}}{N_{xycr}} \right)^2 \leq 1 \quad (41)$$

where n_x and n_{xy} are the actual compressive and shear force per unit length applied to the laminate. If the elastic properties are smeared one can assume that

$$D_{ij} = \frac{A_{ij} t^3}{12} \quad (42)$$

where t is the total thickness of the plate. Substituting Eq. 42 in Eqs. 33 and 34 gives

$$N_{xcr} = \bar{K}_x t^3 \quad (43)$$

and

$$N_{xycr} = \bar{K}_{xy} t^3 \quad (44)$$

where

$$\bar{K}_x = \frac{K_x \pi^2 A_{11}}{12 w^2} \quad (45)$$

and

$$\bar{K}_{xy} = \frac{K_{xy} \pi^2 A_{11}}{12 w^2} \quad (46)$$

Substituting Eqs. 43 and 44 in Eq. 41 gives

$$\frac{n_x}{\bar{K}_X t^3} + \frac{n_{xy}}{\bar{K}_{XY} t^3}^2 \leq 1 \quad (47)$$

Hence, knowing n_x , n_{xy} , \bar{K}_X , and \bar{K}_{XY} , Eq. 47 can be solved for t to satisfy the stability criteria in the equality sense. In the OPTCOMP program during each iteration in the second stage, the required 't' is evaluated and the thickness of each layer is increased proportional to the specified fractions. The influence of boundary conditions can be taken into consideration by specifying the proper value of the width, w , of the plate.

5. Transformations

In this section the transformations used to define the fiber direction and the elastic constants in the local coordinate system are given. The fiber direction for 0° fibers can be defined in OPTCOMP either by giving the angle θ as shown in Figure 1(a) separately for each element or by specifying the projection of the 0° fibers on a defined plane (Figure 1(b)). If the latter is used, the projected direction will be the same for all 0° fibers in all the elements. In Figure 1(a) $(\bar{e}_1, \bar{e}_2, \bar{e}_n)$ is the local coordinate system used to define the position of the quadrilateral ABCD. Let \bar{AF} be the unit vector parallel to the 0° fibers in the element. On the plane PQRS in Figure 1(b), $\bar{A_1F_1}$ is the projection of \bar{AF} . The plane PQRS is defined by $\bar{A_1F_1}$ and $\bar{A_1G_1}$ which are perpendicular to each other. \bar{n} is the normal vector of unit length to the plane PQRS.

Consider a plane passing through the vectors $\overline{A_1F_1}$, \overline{n} and \overline{AF} . (See Figure 1(c)). Then

$$\overline{A_1F_1} + \overline{F_1F} = \overline{A_1F} = \overline{AF} \quad (48)$$

and

$$\overline{F_1F} = H \overline{n} \quad (49)$$

where H is a scalar quantity. The relation given in Eq. 49 is correct since the unit vector \overline{n} is parallel to $\overline{F_1F}$. Taking the scalar product of Eq. 48 with $\overline{e_n}$ and using Eq. 49, one can write

$$H = - \frac{\overline{e_n} \cdot \overline{A_1F_1}}{\overline{e_n} \cdot \overline{n}} \quad (50)$$

Thus, knowing the direction of the projected vector $\overline{A_1F_1}$, Eqs. 48 through 50 can be used to define the vector \overline{AF} . In the input data of the OPTCOMP program the vectors $\overline{A_1F_1}$ and $\overline{A_1G_1}$ are defined by giving the coordinates of points A_1 , F_1 and G_1 .

In Figure 2, $(\overline{e_1}, \overline{e_2}, \overline{e_n})$ is the local coordinate system of the element. The unit vector parallel to the fiber direction is $\overline{e_f}$. $\overline{e_g}$ is perpendicular to $\overline{e_f}$ such that $\overline{e_n} = \overline{e_f} \times \overline{e_g}$. The two coordinate systems are related by

$$\begin{aligned} \overline{e_f} &= l_{11} \overline{e_1} + l_{12} \overline{e_2} \\ \overline{e_g} &= l_{21} \overline{e_1} + l_{22} \overline{e_2} \end{aligned} \quad (51)$$

where ℓ_{11} , ℓ_{12} , etc. are the direction cosines. Let $\{\sigma\}$ and $\{\epsilon\}$ be the stresses and strains in the local coordinate system, and $\{\bar{\sigma}\}$ and $\{\bar{\epsilon}\}$ be the same quantities in the fiber coordinate system (\bar{e}_f , \bar{e}_g , \bar{e}_n). The stresses and the strains in the two coordinate systems are related by

$$\{\bar{\sigma}\} = [L_1] \{\sigma\} \quad (52)$$

$$\{\bar{\epsilon}\} = [L_2] \{\epsilon\} \quad (53)$$

where

$$[L_1] = \begin{bmatrix} \ell_{11}^2 & \ell_{12}^2 & 2\ell_{11}\ell_{12} \\ \ell_{21}^2 & \ell_{22}^2 & 2\ell_{22}\ell_{21} \\ \ell_{11}\ell_{21} & \ell_{12}\ell_{22} & \ell_{12}\ell_{21} + \ell_{11}\ell_{22} \end{bmatrix} \quad (54)$$

and

$$[L_2] = \begin{bmatrix} \ell_{11}^2 & \ell_{12}^2 & \ell_{11}\ell_{12} \\ \ell_{21}^2 & \ell_{22}^2 & \ell_{22}\ell_{21} \\ 2\ell_{11}\ell_{21} & 2\ell_{12}\ell_{22} & \ell_{12}\ell_{21} + \ell_{11}\ell_{22} \end{bmatrix} \quad (55)$$

It is of interest to note that

$$[L_1]^{-1} = [L_2]^t \quad (56)$$

and

$$[L_2]^{-1} = [L_1]^t \quad (57)$$

The stress-strain relations are given by

$$\{\sigma\} = [A] \{\epsilon\} \quad (58)$$

and

$$\{\bar{\sigma}\} = [\bar{A}] \{\bar{\epsilon}\} \quad (59)$$

where $[A]$ and $[\bar{A}]$ are the elastic constant matrices in the two coordinate systems. Using Eqs. 52 and 53, Eq. 59 can be written as

$$[L_1] \{\sigma\} = [\bar{A}] [L_2] \{\epsilon\} \quad (60)$$

Multiplying both sides of Eq. 60 by $[L_1]^{-1}$ gives

$$\{\sigma\} = [L_1]^{-1} [\bar{A}] [L_2] \{\epsilon\} \quad (61)$$

or

$$\{\sigma\} = [L_2]^t [\bar{A}] [L_2] \{\epsilon\} \quad (62)$$

Comparing Eqs. 58 and 62 we can write

$$[A] = [L_2]^t [\bar{A}] [L_2] \quad (63)$$

Eq. 63 can be used to transform the elastic constant matrix from one coordinate system to the other.

SECTION III

DESCRIPTION OF THE PROGRAM

The flow diagram showing the basic steps of the optimization procedure used in OPTCOMP is given in Figure 2. In the first stage, the structure is optimized to satisfy the stress constraints alone. The design at the end of this stage will have minimum weight, and it will satisfy the strength criteria in all the elements. In the second stage, the plate elements which are critical in buckling are increased in thickness to prevent local buckling. At the end of the second stage, all elements will satisfy the strength criteria and no plate elements will fail due to local buckling. The initial thicknesses of the elements for optimization can be included in the input data or the program will assign 0.1-inch thickness for plate elements with equal percentage of laminae in all fiber directions, and 1.0 square inch cross-sectional area for bar elements.

The four types of elements included in the program are: 1) a constant strain triangle, 2) a quadrilateral constructed from four constant strain triangles, 3) a shear panel, and 4) a bar. The sequence of node numbers used to define the elements is given in Figure 4. The triangle and the quadrilateral may be layered composite elements. In defining the material axis, either the direction of the 0° fiber orientation is specified by angle θ (Figure 1(a)) defined by the variable ZANGLE in the input Card Set 11 for each element, or the projection of the 0° fibers is given on a certain plane (See Figure 1(b)). This projection will be the same for all the elements. The projection and the plane is defined by the coordinates of the three points in the input Card Set 9. The fiber

directions in other layers are given in the input Card Set 10, with respect to the 0° direction. The material properties are given for all layers in an element as a set, which is designated by one material number. For example, an element with all layers of graphite epoxy may be assigned material number 1, an element with one layer of boron-epoxy and the remaining layers of graphite epoxy may be assigned material number 2, and an element with two layers of graphite-epoxy and the remaining layers of glass-epoxy may be assigned material number 3. If there are any metal elements with isotropic properties they can be assigned yet a different material number. The total number of materials to be used for quadrilaterals and triangles is defined by the variable NMAT2 in the input Card Set 3, and the elastic properties, thickness of lamina, minimum number of lamina in each layer are read in the input Card Set 7. In the case of shear panels and bar elements the number of materials used is defined by NMAT1 in Card Set 3 and the elastic properties and the maximum allowable stresses are read in Card Set 6. It should be pointed out here that the material numbers assigned for the quadrilaterals and the triangles may be the same as the one used for the shear panels and bars, but their meanings are different for the two cases.

The program has provision for four widely used strength criteria for composite structures. These are: 1) maximum allowable stress, 2) maximum allowable strain, 3) Hill's criteria modified by Tsai, and 4) Norris criteria. These criteria are given in Section II(2). The user can select any one of these criteria depending on his preference by specifying the proper number for NCRTIA in the input Card Set 3, and

for NEF in the input Card Set 4. All these criteria are functions of the maximum normal allowable stress or strain in the fiber direction and in the transverse direction, and the maximum shear allowable. The maximum normal allowable stresses may have different magnitudes depending on whether they are tensile or compressive stresses. As such the total number of maximum allowable stresses specified is five. In case the user wants to relax the restriction on any of these allowables, he can specify a very high value. Selection of these allowables is very important since the weight of the structure is directly proportional to the scaling parameter which is determined on the basis of satisfying the strength criteria for each element. This data is included in the input Card Set 8 for each layer of the material set.

There are four recurrence relations included in the program (See Section II(3)). Two relations are based on the strain energy density in an element and the remaining are based on the state of stress. The user may select any one of these by assigning the proper value to NENG in the input Card Set 3. If the user is not familiar with the implications of using the different recurrence relations, it is recommended that he use $NENG = 1$. This will produce good results for a general structure.

In the input Card Set 3, LSTCCL designates the total number of iterations for the stress constraint design. Out of this total, for LLSTCL iterations, the design variables are modified by using the recurrence relation. In the remaining iterations, the thickness of each element is modified to make the scaling parameter Λ (See Eqs. 13, 19, 20) nearly the same for all the elements. During this second phase of optimization, the percentage of laminae in the different fiber directions in the composite

elements remains unchanged. This process of equalizing the scaling parameter sometimes reduces the weight of the structure depending upon the type of structure, the variation in the magnitude of the different maximum allowables, and the selected strength criteria. At the end of LSTCCL iterations, the details of the minimum weight design are printed out. In the second stage for LNSB iterations, the thicknesses of the quadrilateral and triangular elements are modified to prevent local buckling of the panels (See Section II(4)). These equations are based on the approximation of smeared orthotropic properties. The effect of boundary conditions can be taken into consideration by specifying the proper value of the effective width of the plate (AWIDE(I), BWIDE(I) in the input Card Set 11). It should be noted that the effective width is not the width of the element, but the distance between the supports multiplied by the proper fraction to account for boundary conditions other than simply supported. The user has the option to indicate which layer thicknesses to increase and in what proportion to satisfy the stability criteria. This is achieved by giving proper values to PECT in the input Card Set 5. In the case of a 0° , 90° , $\pm 45^\circ$ laminate, it has been found that it is more beneficial to increase the $\pm 45^\circ$ layers than to increase the thickness of the whole laminate. The initial design for the second stage of optimization is the minimum weight design obtained in the first stage. In the second stage, since the thicknesses of the elements are increased, the weight of the structure goes up with each additional iteration. When all the plate elements satisfy the stability criteria, the weight of the structure stabilizes. It has been found that about five to six iterations are generally sufficient to reach the stable conditions.

The program has a provision for specifying the minimum and maximum thickness of each element by specifying the proper values of BMIN and BMAX in the input Card Sets 11 and 12. The minimum number of laminae in a layer is defined by THMIN in the input Card Set 7. The elements can be linked to have the same sizes by defining the number of sets and the element numbers in each set in the input Card Sets 13 and 14.

When the structure is to be analyzed only, set LSTCCL = 0 in the input Card Set 3. The elements which violate the specified criteria can be separated by assigning LCHEK = 1 in the input Card Set 4. This control will print out all the elements which violate the strength criteria and also the ratio of the actual stress to the maximum allowable stress for those elements. If the specified criteria is the maximum strain criteria, it will be the ratio of strains and for the Hill and Norris criteria, the effective stress ratio.

In the output the user has the option to print the stresses in the local element coordinate or the fiber coordinate system in a layer (See Figure 2), the strains in the fiber coordinate system, the effective stress ratios in each layer as defined by the Hill or Norris criteria, the average stresses in the 0° fiber coordinate system, and the nodal forces in the local or global coordinate system. The control parameters for this data are defined in the input Card Set 4. The reactions at the boundaries and the sum of the forces and moments about the three coordinate axes through the origin are printed at the end. This information is generally useful for an equilibrium check.

The program listed in the report is limited to solving problems with a maximum value of the different variables as given below

Number of members - MEMBS - 120

Number of quadrilaterals and triangles - NC - 50

Number of shear panels and posts - NI - 70

Number of nodes - JOINTS - 50

Number of boundaries - NB - 50

Number of loading conditions - LOADS - 2

Number of layers in the composite elements - NZ - 4

Number of sets of the materials for shear panels and bars -
NMAT1 - 4

Number of different materials used for quadrilaterals and
triangles - NMAT2 - 4

Number of elements in the stiffness matrix - NMAX - 5000

Number of sets of linked elements - LNK - 5

Number of elements in each linked sets - LMAX - 10

Core required on the Cyber-7400 - NOS/BE operating system - 114₈K

On the Cyber-7400 a much larger problem can be solved by increasing the dimensions of the different variables as indicated by the comments in the program.

A short description of the subroutines used in the program is given below.

Program MAIN

The MAIN program reads the input data, generates the stiffness matrix of the whole structure and determines the displacement vector. It then uses the information received from the MEMB subroutine to calculate the weight of the structure and to modify the design variables by using the recurrence relations. At the end it prints out the number of laminas in the layered elements, the cross-sectional areas of the bar elements and the thicknesses of the remaining elements.

SUBROUTINE MEMB

This subroutine is the longest subroutine and is called twice from the MAIN program. It calculates the stresses, strains, and strain energies in all the elements. The scaling parameter Λ used to satisfy the strength criteria in all the elements is determined in this subroutine. The membrane elements which are susceptible to local buckling are modified here. The stresses, strains, nodal forces, etc. are printed out by this subroutine after the last iteration.

The following subroutines are called by MAIN and MEMB to carry out various calculations.

SUBROUTINE SUGD

This subroutine calculates the scaling parameter for the membrane element to satisfy the maximum stress criteria or the maximum strain criteria.

SUBROUTINE AVECT

This subroutine defines the direction of the fibers in the layer of a membrane element by a set of components in the local coordinate system of the element.

SUBROUTINE SURFACE

This subroutine calculates the scaling parameter for a shear panel and a bar element to satisfy the maximum allowable stress.

SUBROUTINE COORD

This subroutine establishes the local coordinate system of the elements. It also determines the transformation matrices needed to transform the stiffness matrix and the displacement vector from one coordinate system to another.

SUBROUTINE ELAS

This subroutine determines the elastic matrix of the layer in the membrane element in the local coordinate system. It also calculates the transformation matrix to get the stresses and strains in the fiber coordinate system from the local coordinate system.

SUBROUTINE STRAIN

This subroutine evaluates the strains in a membrane element from the nodal displacements in the local coordinate system.

SUBROUTINE COMP

This subroutine calculates the stiffness matrix of a triangular membrane element in the local coordinate system.

SUBROUTINE ELSTIF

This subroutine determines the stiffness matrix of a bar element in the local and the global coordinate system.

SUBROUTINE ASEMBL

This subroutine assembles the stiffness matrix for the whole structure in the global coordinate system.

SUBROUTINE CONDNS

This subroutine eliminates the internal dummy node point of the quadrilateral and condenses the stiffness matrix from (10x10) to (8x8).

SUBROUTINE SUM

This subroutine assembles the stiffness matrix (10x10) of the quadrilateral from the stiffness matrices of the four triangles.

SUBROUTINE CHANGE

This subroutine which is called from the CONDNS subroutine rearranges the stiffness matrix of the quadrilateral and the shear panel.

SUBROUTINE TRNSF

This subroutine is used to transform the stiffness matrix from the local coordinate system to the global coordinate system.

SUBROUTINE POP

This subroutine determines the exact core required to store the stiffness matrix of the whole structure. It also defines the location of the elements on the main diagonal and the first nonzero element of each row.

SUBROUTINE GAUSS

This subroutine solves the linear algebraic equations and determines the displacement vector.

SUBROUTINE BOUND

This subroutine eliminates the rows and columns corresponding to the boundary conditions from the stiffness matrix of the whole structure.

SUBROUTINE PRNTDR

This subroutine is called to print out the node numbers, coordinates, forces, and the displacements.

SUBROUTINE REDUCE

This subroutine eliminates the rows from the force matrix corresponding to the boundary conditions.

SUBROUTINE RESTOR

This subroutine restores the boundary conditions in the force and the displacement matrices.

SUBROUTINE ELSTRS

This subroutine determines the stress and the strain energy in a bar element.

SUBROUTINE SCOMP

This subroutine computes the stiffness matrix of a shear panel.

SUBROUTINE SSRS

This subroutine determines the stress and the strain energy in a shear panel.

SUBROUTINE BFORCE

This subroutine calculates the average stress on the the membrane element.

SUBROUTINE SBUCKL

This subroutine determines the increase in thickness of the membrane element needed to prevent local buckling of the plate.

SUBROUTINE ANORM

This subroutine normalizes the relative design vector.

SUBROUTINE MULT

This subroutine is used to multiply the stiffness matrix and the displacement matrix to determine the applied forces and the reactions at the supports.

SUBROUTINE ENGS

This subroutine determines the strain energy in a layer of a membrane element.

SUBROUTINE STRESS

This subroutine determines the stresses in the local coordinate system and the material coordinate system and the strains in a membrane element.

SUBROUTINE SHILL

This subroutine determines the scaling parameter in a membrane element for the strength criteria of Hill or Norris.

SUBROUTINE AVG

This subroutine determines the average stresses for a quadrilateral from the stresses in the four triangles.

SUBROUTINE AEQ

The sum of all the external forces and the moments about the three axes passing through the origin are calculated in this subroutine to check the overall equilibrium.

SUBROUTINE ALINK

This subroutine calculates the average strain energy density for all the elements which are specified to have the same thickness for linking of variables.

SECTION IV

PROGRAM INPUT INSTRUCTIONS

This section describes the input data required for optimization of a structure.

Card Set 1 PROBLEMS (15)

Col	1-5	NSTAR	Number of problems
-----	-----	-------	--------------------

Note: Subsequent set of cards are repeated depending on the number of problems to be solved.

Card Set 2 HEADING (8A10)

Col	1-80	HED	An alphanumeric description of the problem to be solved.
-----	------	-----	--

Card Set 3 GENERAL PROBLEM RELATED DATA (16I5)

Col	1-5	MEMBS	Number of elements
	6-10	NC	Number of membrane elements (quadrilaterals and triangles)
	11-15	NI	Number of shear panels and posts (bar elements)
	16-20	JOINTS	Number of nodes
	21-25	NB	Number of restrained degrees of freedom
	26-30	LOADS	Number of loading conditions
	31-35	MM	Number of degrees of freedom per node

MM =

$$\begin{cases} 2 - \text{two-dimensional problem} \\ 3 - \text{three-dimensional problem} \end{cases}$$

36-40	NZ		Number of layers in the membrane elements. Each layer corresponds to a fiber orientation.
-------	----	--	---

41-45 INCHES

Control for units of distance

INCHES =

- 1 - for inches
- 0 - for feet

46-50 KIPS

Control for the units of the applied forces

KIPS =

- 1 - for kips
- 0 - for pounds

51-55 NMAT1

Number of different sets of materials used for shear panels and bars (See Section III)

56-60 NMAT2

Number of different materials used for membrane elements (See Section III)

61-65 NFIBER

Control for the definition of 0° fibers

NFIBER =

- 0 - Projection specified
- 1 - Angle θ is specified for each element

66-70 NCRTIA

Control for the strength criteria

NCRTIA =

- 1 - Maximum stress (See Eq. 6)
- 2 - Maximum strain (See Eq. 7)
- 3 - Hill (See Eq. 8) (NEF in Card Set 4 must be 1)
- 4 - Norris (See Eq. 9) (NEF in Card Set 4 must be 2)

71-75 NENG

Control for the recurrence relation

NENG =

- 1 - Strain energy density (See Eq. 24)
- 2 - Ratio of strain energy to maximum strain energy density (See Eq. 27)
- 3 - Effective stress ratio (See Eq. 29) NCRTIA must be 3
- 4 - Effective stress ratio (See Eq. 31) NCRTIA must be 4

76-80 LINK

Control for linkage of elements (See Section III)

LINK =

- 0 - No linkage
- 1 - linkage

1-5 MCONST

Control for specified sizes for a set of elements

MCONST =

- 0 - Small number of specified sizes (less than 20% of the total number of elements)
- 1 - Large number of specified sizes

6-10 NSTART

Control to read the element sizes for the initial design

NSTART =

- 0 - Not specified
- 1 - Specified

11-15 LSTCCL

Number of iterations for the stress constraint design. If LSTCCL = 0 the given structure is to be analyzed only.

16-20	LLSTCL	Number of iterations to modify the design variables by using the recurrence relation (See Section III) LLSTCL < LSTCCL
21-25	LNSB	Number of iterations for modification of the membrane elements to avoid local buckling (See Section III).
26-30	NBLNCE	Control to equalize the thickness of $\pm \theta$ layers (last two layers).

$$\text{NBLNCE} = \begin{cases} 0 & \text{- Not required} \\ 1 & \text{- Required} \end{cases}$$

Card Set 4

OUTPUT CONTROL (1615)

Note: This set controls the nature of the output desired. The program will write only that information which the user asks. Most of these controls are for the stresses in the quadrilaterals and triangles.

Col

1-5	NAREA	The relative sizes of the elements for intermediate iterations
	NAREA =	$\begin{cases} 0 & \text{- Not required} \\ 1 & \text{- Required} \end{cases}$
6-10	NLO	The stresses in the layers of the membrane elements in the local element coordinates.
	NLO =	$\begin{cases} 0 & \text{- Not required} \\ 1 & \text{- Required} \end{cases}$
11-15	NZEO	The stresses in the layers of the membrane elements in the material coordinates of the 0° layer
	NZEO =	$\begin{cases} 0 & \text{- Not required} \\ 1 & \text{- Required} \end{cases}$

16-20 NSTRN

The strains in the layers
of the membrane elements in
fiber direction

NSTRN =

[0 - Not required
1 - Required

21-25 NAVG

The average stresses for
the laminate in the 0°
fiber direction

NAVG =

[0 - Not required
1 - Required

26-30 NFI

The stresses in the layers
of the membrane elements in
the individual layer material
coordinates

NFI =

[0 - Not required
1 - Required

31-35 NEF

The effective stresses in the
layers of membrane elements

NEF =

[0 - Not Required
1 - Required (See Eq. 8)
2 - Required (See Eq. 9)

36-40 NFOR

The nodal forces in the
elements

NFOR =

[0 - Not required
1 - Required (local element
coordinates)
2 - Required (global
coordinates)

41-45

LCHEK

Identification of the elements which do not satisfy the strength criteria, when the structure is to be analyzed only (LSTCCL = 0).

0 - Not required

LCHEK =

1 - Required

Card Set 5

DATA TO PREVENT MEMBRANE ELEMENTS
FROM LOCAL BUCKLING (8E10.4)

Note: Card set 5 is needed only if LNSB defined in card set 3 is not equal to zero.

PECT(I)

I = 1,...NZ

Fractions to control the increase in thickness of each layer to prevent local buckling (See Section III)

$$\sum_{I=1}^{NZ} \text{PECT}(I) = 1.$$

Card Set 6

MATERIAL PROPERTY DATA FOR SHEAR
PANELS AND BARS (4F20.4)

Note: Units of the allowable stresses in this set should be consistent with that of the loads. There will be two cards for each value of I. I is the material property number.

Col

1-20

E(I)

Elastic modulus in $\text{psi}/10^6$
for the Ith material

21-40

PMU(I)

Poisson's ratio for the
Ith material

41-60

GSHE(I)

Shear modulus in $\text{psi}/10^6$
for shear panels for the
Ith material. If this input
is zero, then

$$\text{GSHE}(I) = \frac{E(I)}{2*(1+\text{PMU}(I))}$$

61-80	SSTRT(I)	Allowable stress in tension for bars for the Ith material
1-20	SSTRC(I)	Allowable stress in compression for bars for the Ith material
21-40	SSTRS(I)	Allowable stress in the shear panels for the Ith material
41-60	SPWTT(I)	Density in lbs/in ³ for the Ith material

$I = 1, \dots, \text{NMAT1}$

Card Set 7

MATERIAL PROPERTY DATA FOR
MEMBRANE ELEMENTS (8E10.4)

Note: The total number of data cards in this set is equal to $NZ * \text{NMAT2}$. There will be NMAT2 subset of cards. The Jth subset contains NZ cards giving the properties of each layer in sequence. J defines the material property number for the laminate.

Col

1-10	E11(I,J)	Elastic modulus parallel to the fiber direction for the Ith layer of the Jth material in psi/10 ⁶ .
11-20	E22(I,J)	Elastic modulus transverse to the fiber direction for the Ith layer of the Jth material in psi/10 ⁶ .
21-30	ANU1(I,J)	Poisson's ratio ν_{12} for the Ith layer of the Jth material.
31-40	ANU2(I,J)	Poisson's ratio ν_{21} for the Ith layer of the Jth material. If this input is zero, then $\text{ANU2(I,J)} = \text{ANU1(I,J)} \frac{\text{E22(I,J)}}{\text{E11(I,J)}}$
41-50	GSH(I,J)	Shear modulus for the Ith layer of the Jth material in psi/10 ⁶ . If this input is zero, then $\text{GSH(I,J)} = \frac{\text{E11(I,J)}}{(2(1+\text{ANU1(I,J)}))}$

51-60	SPWTC(I,J)	Density in lbs/in ³ for the Ith layer of the Jth material.
61-70	THEK(I,J)	Thickness of a lamina of the Ith layer of the Jth material. This value is used to calculate the number of laminas in a layer for the final design.
71-80	THMIN(I,J)	Minimum thickness of the Ith layer of the Jth material. This value is equal to the minimum number of laminas multiplied by THEK(I,J).

I = 1,...,NZ

J = 1,...,NMAT2

Card Set 8 ALLOWABLE STRENGTHS FOR THE
MEMBRANE ELEMENTS (5F15.2)

Note: The number of cards in this set is equal to NZ * NMAT2. Thus there will be NMAT2 subset of cards. The Jth subset contains NZ cards giving the allowables of each layer in sequence. J is the material property number for the laminate. Units of the allowable stresses in this set should be consistent with that of the loads. If NCRTIA = 2 (i.e., Maximum strain criteria) this set will contain allowable strains instead of allowable stresses.

Col

1-15	SSMAX(1,K,J)	Tension allowable parallel to the fiber direction in the Kth layer of the Jth material.
16-30	SSMAX(2,K,J)	Compression allowable parallel to the fiber direction in the Kth layer of the Jth material.
31-45	SSMAX(3,K,J)	Tension allowable transverse to the fiber direction in the Kth layer of the Jth material.

46-60 SSMAX(4,K,J)

Compression allowable
transverse to the fiber
direction in the Kth layer
of the Jth material.

61-75 SSMAX(5,K,J)

Shear allowable transverse
to the fiber direction in
the Kth layer of the Jth
material.

$K = 1, \dots, NZ$

$J = 1, \dots, N\text{MAT}2$

Card Set 9

DATA FOR DEFINITION OF THE 0°
FIBER (3E10.4)

Note: Coordinates of three points are given. This defines the plane of projection and the projection of the 0° fibers on the plane. (See Section II(5)). If $MM=2$, only $XD(I)$ and $YD(I)$ are input. If $NIFBER=1$ in card set 1, this card set is not read.

Col

1-10 XD(I)

X coordinate of the Ith
point.

11-20 YD(I)

Y coordinate of the Ith
point.

21-30 ZD(I)

Z coordinate of the Ith
point.

$I = 1, \dots, 3$

Card Set 10

FIBER DIRECTION ANGLES IN LAYERS
(8E10.4)

ANGLE(I)

Angle in degrees for the NZ
layers. The first angle is
 0° .

$I = 1, \dots, NZ$

Card Set 11

DATA FOR MEMBRANE ELEMENTS
(QUADRILATERALS OR TRIANGLES)
(I5, I3, I2, 4I5, 5E10.4)

Note: There is one card for each membrane element (See Figure 4). The elements are numbered in sequence but can be arranged in any order within this card set. All membrane elements are first numbered, then shear panels and bars are numbered.

Col

1-5	I	Element number
6-8	KTYPE(I)	Type of element
		3 - triangle
	KTYPE(I) =	
		4 - quadrilateral
9-10	NMAT(I)	Material property number of the element
11-15	MA(I)	First node number of the element
16-20	MB(I)	Second node number of the element
21-25	MC(I)	Third node number of the element
26-30	MD(I)	Fourth node number of the element. For a triangle MD(I) = 0
31-40	BMAX(I)	Maximum thickness allowed for the element. If there is no limit on size input a large number say 100.
41-50	BMIN(I)	Minimum thickness allowed for the element. This should be consistent with THMIN(I,J) in card set 7.

Note: AWIDE and BWIDE should be consistent with the element supports for local buckling.

51-60	AWIDE(I)	Effective width of the Ith element transverse to the 0° fibers for local buckling. N_x (compression) critical.
61-70	BWIDE(I)	Effective width of the Ith element parallel to the 0° fibers for local buckling. N_y (compression) critical.

Note: If NFIBER = 0 in card set 3, then ZANGLE(I) is zero.

71-80	ZANGLE(I)	Angle in degrees which the 0° fibers make with a line joining MA(I) and MB(I). See Fig. 1(a).
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$I = 1, \dots, NC$

Card Set 12

DATA FOR SHEAR PANELS AND BARS
(I5, I3, I2, 4I5, 2E10.4)

Note: There is one card for each element. The element numbering is sequential but they can be arranged in any order within this card set. These elements are numbered after all the membrane elements are numbered.

Col

1-5	I	Element number
6-8	KTYPE(I)	Type of element
		$KTYPE(I) = \begin{cases} 5 & \text{- shear panel} \\ 2 & \text{- bar} \end{cases}$
9-10	NMAT(I)	Material property number of the element
11-15	MA(I)	First node number of the element
16-20	MB(I)	Second node number of the element
21-25	MC(I)	Third node number of the element. MC(I) = 0 for bar.

26-30	MD(I)	Fourth node number of the element. MD(I) = 0 for bar.
31-40	BMAX(I)	Maximum thickness allowed for the element. For a bar element BMAX(I) is the cross-sectional area. If there is no limit on size, input a large number, say 100.
41-50	BMIN(I)	Minimum thickness allowed for the element. For a bar element BMIN(I) is the cross-sectional area.

$$I = (NC + 1), \dots, MEMBS$$

Note: Card sets 13, 14, and 15 are read only if LINK = 1 in card set 3.

Card Set 13 DATA FOR LINKAGE (3I5)

Col

1-5	LNK	Total number of sets of linked elements.
6-10	NSKIN	Number of sets of linked elements amongst the membrane elements.
11-15	NINT	Number of sets of linked elements amongst the shear panels and bars.

Card Set 14 NUMBER OF ELEMENTS LINKED PER SET (16I5)

NLINK(I) Number of elements in each linked set.
I = 1, ..., LNK

Card Set 15 LINKED ELEMENTS (16I5)

NELEM(K,J) Element numbers in each linked set. Each subset is started on a new card.
K = 1, ..., LNK
J = 1, ..., NLINK(K)

Note: Card sets 16 and 17 are read only if LSTCCL = 0 or NSTART = 1 in card set 3.

Card Set 16

THICKNESSES OF THE QUADRILATERALS
AND TRIANGLES (8E10.4)

AL(I,L)
I = 1,...,NC
L = 1,...,NZ

Thickness of membrane elements.
Each subset contains one thickness of the layer for all elements. The number of subsets is equal to the number of layers. Each subset is started on a new card.

Card Set 17

THICKNESSES OF SHEAR PANELS AND
CROSS-SECTIONAL AREAS OF BARS (8E10.4)

A(I)
I = (NC + 1),...,MEMBS

Thickness of shear panels and cross-sectional areas of bars.

Card Set 18

COORDINATES OF NODES (15, 3E10.4)

Note: The node numbering must be sequential but may be arranged in any order within this card set.

Col

1-5	I	Node number
6-15	X(I)	X coordinate of the Ith node
16-25	Y(I)	Y coordinate of the Ith node
26-35	Z(I)	Z coordinate of the Ith node

I = 1,...,JOINTS

Card Set 19

BOUNDARY CONDITIONS (1615)

IBND(I)
I = 1,...,NB

Degree of freedom numbers of those nodes which are restrained. For node k the degree of freedom numbers are 3*K-2, 3*K-1, and 3*K (X, Y and Z respectively) for MM = 3 and 2*K-1 and 2*K (X and Y respectively) for MM = 2.

Card Set 20

NUMBER OF FORCES IN EACH LOADING
CONDITION (1615)

NJLODS(I)
I = 1,...,LOADS

Number of load components in the Ith loading condition.

Card Set 21

LOCATION AND APPLIED LOADS
3(E10.4, 215)

Note: The number of subsets is equal to the number of loading conditions (LOADS). Each subset is started on a new card.

Col

1-10	TFR(J)	Value of the Jth load.
11-15	IM(J)	Direction of the load
	IM(J) =	$\begin{cases} 1 - X \text{ direction} \\ 2 - Y \text{ direction} \\ 3 - Z \text{ direction} \end{cases}$
16-20	JM(J)	Number of the node where the load is applied.

$J = 1, \dots, \text{NJLOADS}(I)$

SECTION V

PROGRAM OUTPUT DETAILS

This section describes the output from the program. The items are discussed in the order they appear.

1) The input data is printed out in the same format as the READ statements in the program.

2) The output from the POP subroutine concerning the distribution of elements in the stiffness matrix is printed. This information is

a) GROSS POPULATION - total number of elements in the upper triangle of the matrix.

b) APPARENT POPULATION - actual number of elements considered as non-zero by a given solution scheme. The apparent population represents the number of storage locations required for the stiffness matrix. If the apparent population is greater than NMAX, the program will write the message INSUFFICIENT CORE TO STORE STIFFNESS MATRIX, DIMENSION OF SK SHOULD BE (APPARENT POPULATION NUMBER).

c) STARTING ROW NUMBER FOR EACH COLUMN - The number of the row where the first non-zero element occurs in each column. The variable defined in the program is IC(I), I=1, NN(total degrees of freedom).

d) NUMBERS OF DIAGONAL ELEMENTS IN SINGLE ARRAY STIFFNESS MATRIX - For each column I the actual number of elements, ID(I) in the upper triangular matrix up to and including that column, i.e.,

$$ID(I) = \frac{I(I+3)}{2} - \sum_{j=1}^I b_j$$

where b_j is the row number given for Column I in (c). Thus for the last column $ID(LAST) = APPARENT POPULATION$.

3) TIME USED IN SECONDS - Time the program has spent in execution up to that point. This item is repeated after each iteration.

4) RELATIVE AREAS OF MEMBERS - These are printed only if NAREA in the Input Card Set 4 is 1.

a) The first set includes the relative total areas of all members. For composite elements this will be the relative total thickness of all layers; for shear panels, the total thickness, and for bars, cross-sectional areas. The variable defined in the program is $A(I)$, $I = 1, \dots, \text{MEMBS}$.

b) The remaining sets include the relative thickness of each layer in the quadrilateral and triangular elements. The variable in the program is $AL(I,J)$, $I = 1, \dots, \text{NC}$

$J = 1, \dots, \text{NZ}$

- | | | |
|-----------------|---|---|
| 5) WEIGHT-SKIN | - | Weight of all quadrilateral and triangular elements |
| WEIGHT-S-PANELS | - | Weight of all shear panels |
| WEIGHT-POSTS | - | Weight of all bar elements |
| TOTAL-WEIGHT | - | Total weight of the structure. This is the sum of above three weights. |
| ELE SKIN | - | Element amongst the quadrilaterals and triangles requiring the largest value of the scaling parameter to satisfy the strength criteria. |
| ABASE | - | Scaling parameter for the element ELE SKIN. |
| ELE STRUCT | - | Element amongst shear panels and bars requiring largest value of the scaling parameter to satisfy the maximum allowable stress. |
| CBASE | - | Scaling parameter for the element ELE STRUCT. |

STRUCTURE NO	-	Problem number
NO OF LOADS	-	Number of loading conditions
CYCLE NO	-	Iteration number
EQ CYCLE NO	-	Number of iterations used to equalize the scaling parameter in the elements. (See Section III)
NSTBLTY	-	Number of iterations used to modify the elements critical in local buckling.

The data in item 5 is repeated after each iteration.

6) The following information is given for the minimum weight design after completing the specified number of iterations.

L	=	element number
BASEA	=	scaling parameter
L-C	=	critical loading condition for the element
CRI	=	the critical stress in that element
		1 - stress in fiber direction
		2 - stress in transverse direction
		3 - shear stress

In the case of the maximum strain criteria this number will correspond to the critical strain. For Hill and Norris criteria CRI = 1.

LAR	=	layer number where the stress is critical
PRCNT	=	percentage of laminae in the different fiber directions.

In the case of shear panels and bars only L, BASEA, L-C are printed.

7) The following information is given if the structure is to be analyzed only. The output is optional. It is printed only if LCHEK = 1 in the input Card Set 4.

L	=	element number not satisfying the strength criteria.
---	---	--

BASEA = ratio of actual state of stress to the maximum allowable stress. In the case of maximum strain criteria, BASEA is the ratio of strains and for the Hill and Norris criteria BASEA is the ratio of effective stress.

L-C = same as in Item 6

CRI = same as in Item 6

LAR = same as in Item 6

In the case of shear panels and bars only L, BASEA, L-C are printed.

8) MINIMUM WEIGHT CYCLE - the iteration number which gave the lowest weight design.

9) STRESSES - The following information is given under this heading for the minimum weight design

- a) MEMB - element number
- b) NODES - node numbers defining the element
- c) AREA - area of the element (in.²)
- d) THICK - total thickness of the element (in.)
- e) LAYER(THICK) - individual layer thickness (in.)
- f) The following output are optional. They are printed only if proper control numbers are input in Card Set 4.
 - 1) STRESSES IN INDIVIDUAL LAYERS (LOCAL COORDINATES)
 - 2) STRESSES IN INDIVIDUAL LAYERS (ZERO FIBER DIRECTION)
 - 3) STRAINS IN INDIVIDUAL LAYERS (FIBER DIRECTION)
 - 4) AVERAGE STRESSES (ZERO FIBER DIRECTION)
 - 5) STRESSES IN INDIVIDUAL LAYERS (FIBER DIRECTION)
 - 6) EFFECTIVE STRESS
 - 7) NODAL FORCES

For items 1 through 6 the stresses or strains are given for each loading condition on one line. The first number is the loading condition, then the layer numbers, and the stresses σ_1 , σ_2 , σ_{12} are printed out for each layer. The format can accommodate a maximum of four layers. In the case of average stresses only three stresses are given. The nodal forces are printed for each loading condition on one line. When NFOR = 2, Fx, Fy, Fz are given in the global coordinates and when NFOR = 1, Fx and Fy are given in the local coordinates. The sequence of nodal forces is the same as the sequence of numbers used to define the element.

In the case of shear panels MEMB, NODES, AREA, and THICK are given and for bar elements MEMB, NODES, LENGTH, and AREA are printed out. In these two cases since there is only one pertinent stress in the member, the loading conditions and stresses are printed on one line under the heading of stresses.

10) DISPLACEMENTS - The following information is given for each node under this heading for the minimum weight design

- a) JOINT - Node number
- b) X, Y, Z - x, y, z coordinates of the node
- c) FORCE-X, FORCE-Y, FORCE-Z - Applied forces in the x, y, z directions.
- d) DISPL-X, DISPL-Y, DISPL-Z - Displacement in the x, y, and z directions.

The quantities (c) and (d) are given for each loading condition.

11) NUMBER OF LAMINAE IN COMPOSITE ELEMENTS - The number of laminae in all composite elements are given. The element number is given in the parenthesis before the laminae numbers.

12) TOTAL THICKNESS OF ELEMENTS - The actual thicknesses of all the elements corresponding to the minimum weight design are listed.

13) If $LNSB > 0$ items 3 to 12 are repeated for the iterations used to modify the quadrilaterals and triangles to prevent local buckling.

14) REACTIONS - The format for this item is the same as Item 10, except that under the heading FORCE-X, FORCE-Y, FORCE-Z, the reactions are given corresponding to the fixed boundary conditions.

15) SUMMATION OF FORCES

a) Applied loads

LOAD - loading condition

MX - moment of applied forces about x axis through the origin

MY - moment of applied forces about y axis through the origin

MZ - moment of applied forces about z axis through the origin

b) REACTIONS - Same as (a) above except that the quantities are calculated for the reactions at the boundaries.

SECTION VI

ILLUSTRATIVE PROBLEM

A cantilever box-beam shown in Figure 5 is designed to illustrate the use of the computer program. The top and bottom skins consist of four layers with fibers in 0° , 90° , $+45^\circ$, and -45° . The coordinates of the node numbers shown in Figure 5 are given in the computer output in the Appendix. There are 40 nodes and the nodes 37 through 40 are fixed. The total number of elements are 54. There are 18 quadrilaterals, 18 shear panels, and 18 posts. The 6 quadrilaterals at the tip of the box-beam consist of four layers of graphite epoxy. The 6 quadrilaterals in the middle consist of one layer of boron epoxy and three layers of graphite epoxy, and the remaining quadrilaterals consist of four layers of boron epoxy. The shear panels and posts have different elastic properties.

Number of elements - MEMBS = 54
Number of quadrilaterals - NC = 18
Number of shear panels and bars - NI = 36
Number of nodes - JOINTS = 40
Number of restrained degrees of freedom - NB = 12
Number of loading conditions - LOADS = 2
Number of degrees of freedom per node - MM = 3
Number of layers - NZ = 4
Coordinates of nodes given in feet - INCHES = 0
Applied loads given in kips - KIPS = 1
Number of materials used in shear panels and bars - NMAT1 = 2
Number of sets of materials used in quadrilaterals - NMAT2 = 3
Projection of 0° fibers is specified - NFIBER = 0
Maximum stress criteria - NCRTIA = 1
Recurrence relation used (Eq. 24) - NENG = 1
No linkage of elements - LINK = 0
No specified sizes of elements - MCONST = 0
Initial sizes of the elements not specified - NSTART = 0
Total number of iterations - LTSCCL = 15
Number of iterations to modify the design variables by using
the recurrence relation - LLSTCL = 10
Number of iterations to modify the element sizes to prevent
local buckling - LNSB = 8
Equal thickness of layers in $+45^\circ$ and -45° required - NBLNCE = 1

Relative sizes of elements required - NAREA = 1
 Stresses in fiber direction needed - NFI = 1
 Nodal forces in global coordinates required - NFOR = 2

Only the thickness of +45° and -45° layers are to be increased to prevent local buckling. Hence PECT(1) = 0.0; PECT(2) = 0.0; PECT(3) = 0.5; PECT(4) = 0.5.

In Card Set 12, material property number NMAT1 for all bars is 1 and for all shear panels is 2. The elastic properties for the two materials are as follows

	Material 1	Material 2
E	10.5×10^6 psi	30.0×10^6 psi
ν	0.3	0.3
G	-	-
Allowable Stress in Tension	25.0 kips/in ²	10.0 kips/in ²
Allowable Stress in Compression	20.0 kips/in ²	10.0 kips/in ²
Allowable Stress in Shear	15.0 kips/in ²	8.0 kips/in ²
Density	0.1 lbs/in ³	0.28 lbs/in ²

In Card Set 11 material property number NMAT2 for quadrilaterals 13 through 18 is 1; quadrilaterals 7 through 12 is 2 and quadrilaterals 1 through 6 is 3

	Material 1	Material 2	Material 1
0° layer	graphite-epoxy	boron-epoxy	boron-epoxy
90° layer	graphite-epoxy	graphite-epoxy	boron-epoxy
+45° layer	graphite-epoxy	graphite-epoxy	boron-epoxy
-45° layer	graphite-epoxy	graphite-epoxy	boron-epoxy

Elastic properties and allowable stresses for graphite epoxy and boron epoxy are as follows:

	graphite-epoxy	boron-epoxy
E_{11}	18.5×10^6 psi	32.0×10^6 psi
E_{22}	1.6×10^6 psi	3.5×10^6 psi
ν_{12}	0.208	0.25
ν_{21}	0.0203	-
G	0.65×10^6 psi	0.93×10^6 psi
density	.55 lbs/in ³	.0725 lbs/in ³
thickness of laminae	.0052	.0052
Minimum thickness of a layer	.0104	.0104
Allowable stress fiber direction (tension)	139.0 kips/in ²	166.0 kips/in ²
fiber direction (compression)	92.4 kips/in ²	86.0 kips/in ²
Allowable stress transverse fiber direction (tension)	4.95 kips/in ²	6.0 kips/in ²
fiber direction (compression)	29.7 kips/in ²	11.86 kips/in ²
Allowable stress shear	4.68 kips/in ²	3.95 kips/in ²

The projection of 0° fibers is defined by two points (0.0, 0.0, 0.0), (1.0, 0.0, 0.0). The third point defining the plane is (0.0, 1.0, 0.0). The line joining the first and second points is perpendicular to the line joining the first and the third points.

The four fiber orientations are 0°, 90°, +45°, and -45°.

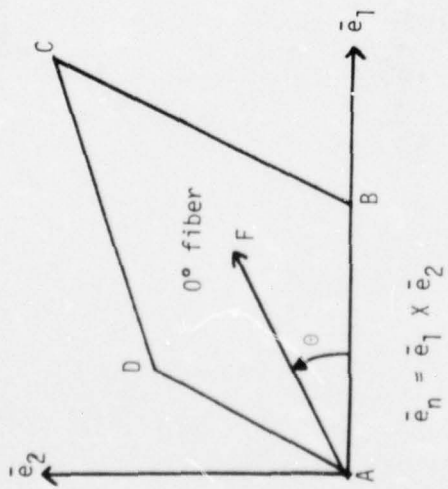
The element connections and coordinates of the nodes are given in the output of the program. The sequence of defining the elements is as shown in Figure 4.

The minimum thickness of all elements is .01 inch.

For all quadrilaterals $AWIDE = 9.0$ in
For all quadrilaterals $BWIDE = 12.0$ in

Nodes 37 through 40 are fixed. Hence, the restrained degrees of freedom are 109 through 120.

The first loading condition consists of a 1 kip load at nodes 1, 2, 3 and 4 acting in the negative z direction. The second loading condition consists of a 0.5 kip load at nodes 2 and 4 acting in the negative z direction.



(c) fiber direction and projection

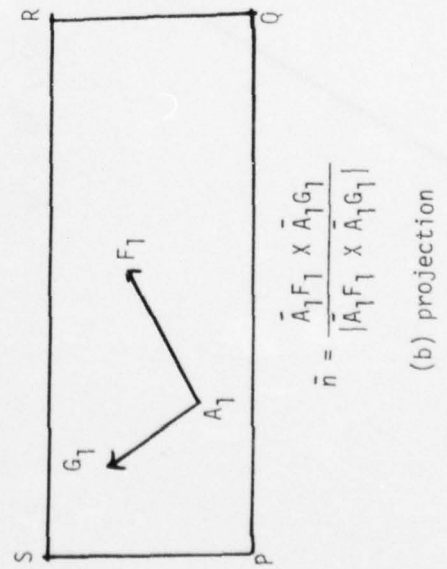
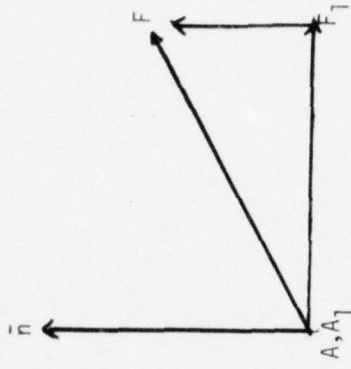
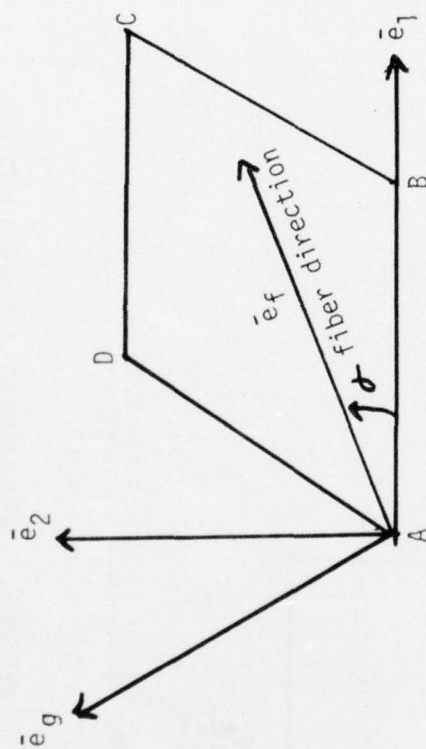


FIGURE 1. DEFINITION OF 0° FIBER DIRECTIONS



$$\bar{e}_n = \bar{e}_1 \times \bar{e}_2 = \bar{e}_f \times \bar{e}_g$$

$(\bar{e}_1, \bar{e}_2, \bar{e}_n)$ local coordinate system of the element ABCD

$(\bar{e}_f, \bar{e}_g, \bar{e}_n)$ material coordinate system

FIGURE 2. LOCAL AND FIBER COORDINATE SYSTEMS

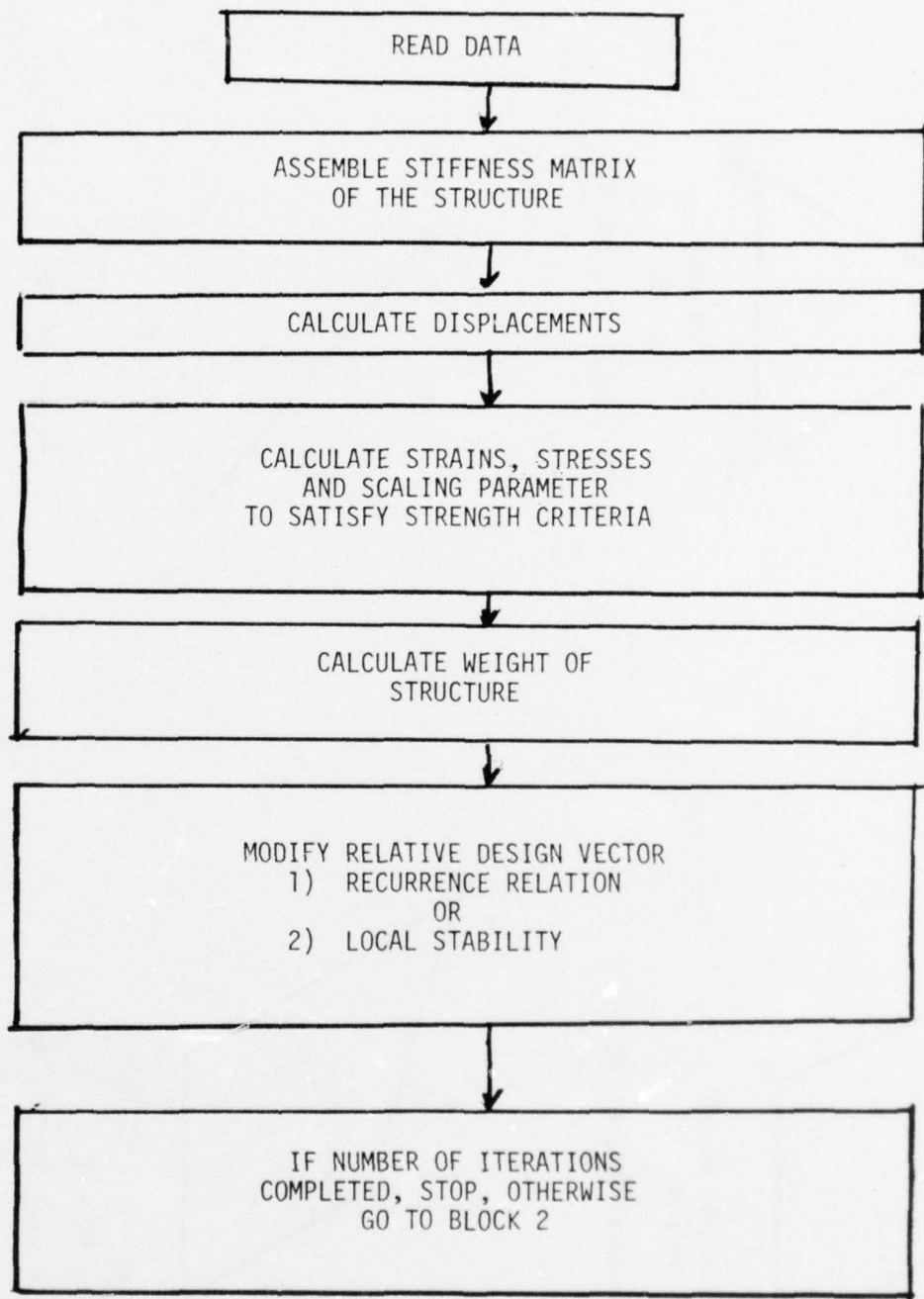
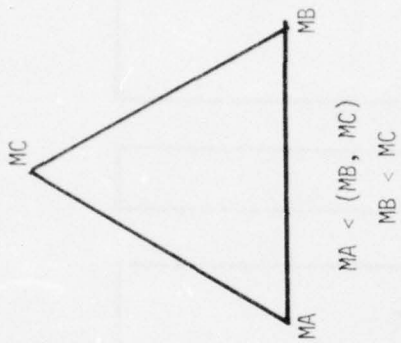
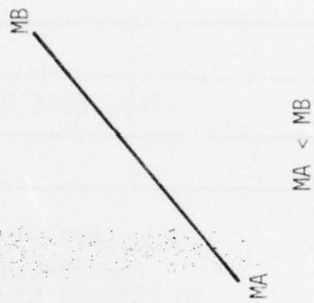


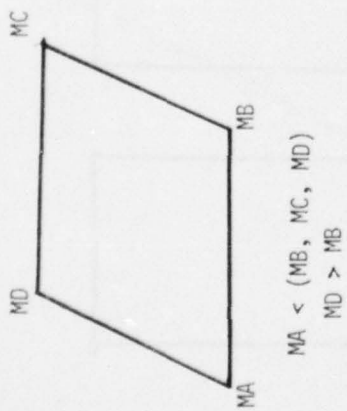
FIGURE 3. FLOW DIAGRAM



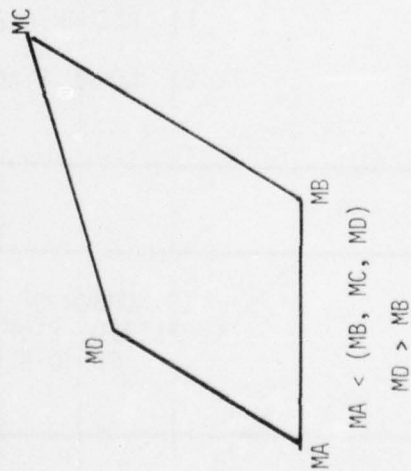
2) triangle (MA, MB, MC)



4) bar (MA, MB)



1) quadrilateral (MA, MB, MC, MD)



3) shear panel (MA, MB, MC, MD)

FIGURE 4. NODAL NUMBERING SYSTEM FOR ELEMENTS

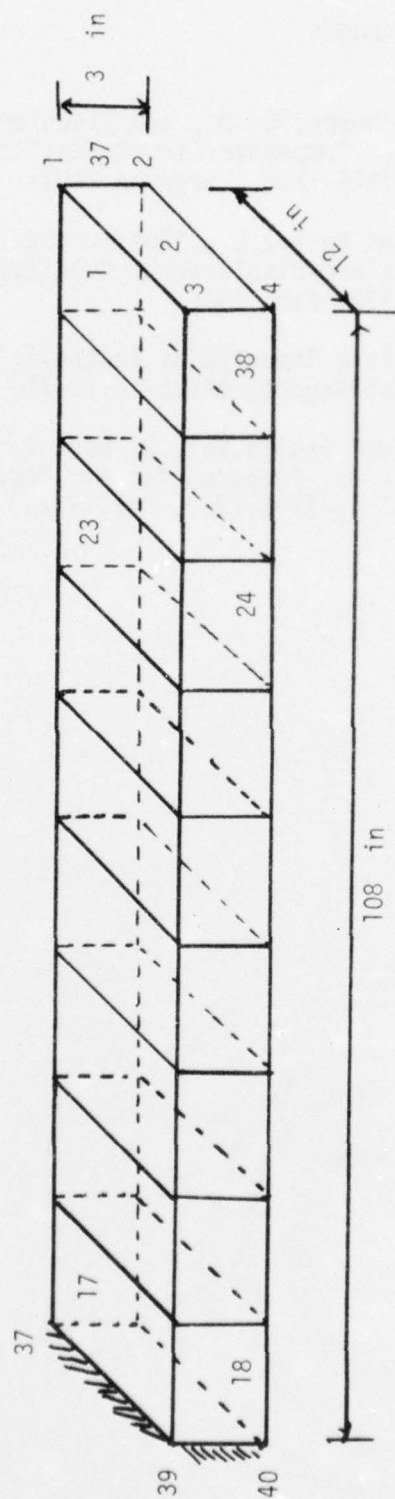


FIGURE 5. Cantilever Box-Beam

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2. Khot, N. S., Venkayya, V. B., and Berke, L., "Optimum Design of Composite Structures with Stress and Displacement Constraints," AIAA Journal, Vol. 14, pp. 131-132, Feb. 1976.
3. Sandhu, R. S., "A Survey of Failure Theories of Isotropic and Anisotropic Materials," Technical Report, AFFDL-TR-72-71, Sept. 72.
4. "Advanced Composites Design Guide" Design Vol. 2, Book 2, Division of the North American Rockwell Corp. Prepared for Air Force Materials Lab. (AFML/LC), Contract No. F33615-71-C-1362, Nov. 1971.

UNLABELED OLDPL

MODIFICATIONS / CONTROL CARDS

10/29/76 12.19.53.

PAGE 1

UPDATE 1.2-76038.

CORRECTION IDENTIS ARE LISTED IN CHRONOLOGICAL ORDER OF INSERTION

MAIN	MEMB	SUGJ	AVECT	SURF	COORD	ELAS	STRAIN
COMP	ESTIF	ASE4BL	CONDNS	SUM	CHANG	TRNSF	POP
GAUSS	BOUND	PRNTD	REDUCE	RESTOR	ELSTRS	SCOMP	SSRS
BFORCE	SBUCK	ANORM	MULT	ENGS	STRESS	SHILL	AVG
AEO	ALINK						

DECKS ARE LISTED IN THE ORDER OF THEIR OCCURRENCE ON A NEW PROGRAM LIBRARY IF ONE IS CREATED BY THIS UPDATE

YANKISS	MAIN	MEM3	SUGD	AVECT	SURF	COORD	ELAS
STRAIN	COMP	ESTIF	ASEMUL	CONDNS	SUM	CHANG	TRNSF
POP	GAUSS	BOUND	PRNTD	REDUCE	RESTOR	ELSTRS	SCOMP
SSRS	BFORCE	SBUCK	ANORM	MULT	ENGS	STRESS	SHILL
AVG	AEO	ALINK					

DECKS WRITTEN TO COMPILE FILE

MAIN	MEMB	SUGJ	AVECT	SURF	COORD	ELAS	STRAIN
COMP	ESTIF	ASE4BL	CONDNS	SUM	CHANG	TRNSF	POP
GAUSS	BOUND	PRNTD	REDUCE	RESTOR	ELSTRS	SCOMP	SSRS
BFORCE	SBUCK	ANORM	MULT	ENGS	STRESS	SHILL	AVG
AEO	ALINK						

THIS UPDATE REQUIRED 341003 WORDS OF CORE.


```

60      WRITE (6,108)
        READ (5,125) NSTR
        WRITE (6,125) NSTR
        KSTR=1
        CONTINUE
1      IF (KSTR.GT.1) WRITE (6,108)
        READ (5,109) HED
        WRITE (6,110) KSTR,HED
        ATIME=SECOND(AAZ)
        WRITE (6,146) ATIME
        READ (5,125) MEMBS,NC,N1,JOINTS,NB,LOADS,MM,NZ,INCHES,KIPS,NMAT1,N
1MAT2,NFIBER,NORTIA,NENG,LINK,MCONST,NSTART,LSTOCL,LLSTOCL,LNSB,NBLN
70      2CE
        WRITE (6,111) MEMBS,NC,N1,JOINTS,NB,LOADS,MM,NZ,INCHES,KIPS,NMAT1,
1MAT2,NFIBER,NORTIA,NENG,LINK,MCONST,NSTART,LSTOCL,LLSTOCL,LNSB,NBL
2NCE
        READ (5,125) NAREA,NLO,NZEO,NSTRN,NAVG,NFI,NEF,NFOR,LCHCK
75      NN=MM*JOINTS
        NM=NN-N3
        IF (LNSB.EQ.0) GO TO 2
        READ (5,126) (PECT(I),I=1,NZ)
        WRITE (6,112) (PECT(I),I=1,NZ)
2      CONTINUE
        IF (NENG.EQ.4) LLSTOCL=LSTOCL+1
        LNSS=LLSTOCL-1
        DO 3 I=1,NMAT1
        READ (5,113) E(I),PMU(I),GSHE(I),SSTRT(I),SSTRC(I),SSTRS(I),SPWT(
85      1)
        WRITE (6,114) E(I),PMU(I),GSHE(I),SSTRT(I),SSTRC(I),SSTRS(I),SPWT
1(I)
        CONTINUE
3      IF (NC.EQ.0) GO TO 8
        READ (5,115) (E1(I),J),E2(I,J),ANU1(I,J),ANU2(I,J),GSH(I,J),SPWT
1(I),J),THEK(I,J),THMIN(I,J),I=1,NZ),J=1,NMAT2)
        WRITE (6,116) ((E1(I),J),E2(I,J),ANU1(I,J),ANU2(I,J),GSH(I,J),SPW
1(I),J),THEK(I,J),THMIN(I,J),I=1,NZ),J=1,NMAT2)
        DO 4 J=1,NMAT2
        DO 4 K=1,NZ
        READ (5,117) (SSMAX(I,K,J),I=1,5)
        WRITE (6,118) (SSMAX(I,K,J),I=1,5)
        CONTINUE
4      IF (NFIBER.EQ.1) GO TO 6
        DO 5 I=1,NTH
        READ (5,119) XO(I),YO(I),ZO(I)
        WRITE (6,120) XO(I),YO(I),ZO(I)
        CONTINUE
5      CONTINUE
6      READ (5,126) (ANGLE(I),I=1,NZ)
        WRITE (6,112) (ANGLE(I),I=1,NZ)
        DO 7 J=1,NC
        READ (5,121) I,KTYPE(I),NMAT(I),MB(I),MC(I),MD(I),BMAX(I),BM
110      1IN(I),AMIDE(I),BWIIDE(I),ZANGLE(I)
        WRITE (6,122) I,KTYPE(I),NMAT(I),MA(I),MB(I),MC(I),MD(I),BMAX(I),B
1MIN(I),AMIDE(I),BWIIDE(I),ZANGLE(I)
        ZANGLE(I)=ZANGLE(I)+3.1415926/180.
        CONTINUE
7      CONTINUE

```

```

115      8      CONTINUE
          NK=NC+1
          DO 9 J=1, MEMBS
            READ (5,121) I,KTYPE(I),NMAT(I),MA(I),MB(I),MC(I),MD(I),BMAX(I),BM
            11N(I)
            WRITE (6,122) I,KTYPE(I),NMAT(I),MA(I),MB(I),MC(I),MD(I),BMAX(I),B
            11MIN(I)
            9      CONTINUE
            IF (LINK.EQ.0) GO TO 11
            READ (5,125) LNK,NSKIN,NINT
            WRITE (6,111) LNK,NSKIN,NINT
            READ (5,125) (NLINK(I),I=1,LNK)
            WRITE (6,111) (NLINK(I),I=1,LNK)
            DO 10 K=1,LNK
              KX=NLINK(K)
              READ (5,125) (NELEM(K,J),J=1,KX)
              WRITE (6,111) (NELEM(K,J),J=1,KX)
            10      CONTINUE
            11      CONTINUE
            IF (INSTART.EQ.1) GO TO 12
            IF (LSTCCL.GT.0) GO TO 15
            12      CONTINUE
            IF (NC.EQ.0) GO TO 13
            DO 14 L=1,NZ
              READ (5,126) (AL(I,L),I=1,NC)
              WRITE (6,112) (AL(I,L),I=1,NC)
            14      CONTINUE
            13      CONTINUE
            K1=NC+1
            READ (5,126) (A(I),I=K1,MEMBS)
            WRITE (6,112) (A(I),I=K1,MEMBS)
            15      CONTINUE
            DO 16 J=1,JOINTS
              READ (5,124) I,X(I),Y(I),Z(I)
              WRITE (6,123) I,X(I),Y(I),Z(I)
            16      CONTINUE
            READ (5,125) (IBND(I),I=1,NB)
            WRITE (6,111) (IBND(I),I=1,NB)
            DO 17 I=1,NB
              DO 17 J=1,LOADS
                DR(I,J)=0
                FR(I,J)=0
            17      CONTINUE
            READ (5,125) (NJLOADS(I),I=1,LOADS)
            WRITE (6,111) (NJLOADS(I),I=1,LOADS)
            DO 23 J=1,LOADS
              KX=NJLOADS(J)
              IF (KX-3) 19,19,20
            19      KX=KX
              GO TO 21
            23      KX=3
            21      READ (5,126) (TFK(I),IM(I),JM(I),I=1,KX)
              WRITE (6,127) (TFK(I),IM(I),JM(I),I=1,KX)
              DO 22 I=1,KX
                KY=MM+JM(I)-MM+IM(I)
                FR(KY,J)=FR(KY,J)+TFK(I)
                KX=KX-KX
            22      CONTINUE
              IF (KX) 23,23,10

```



```

23 CONTINUE
   DO 24 I=1,NMAT1
      IF (GSH(I),EQ,0.) GSH(I)=E(I)/(2.*(1+PMU(I)))
      E(I)=E(I)*10.**6
      GSH(I)=GSH(I)*10.**6
   CONTINUE
24  DO 25 I=1,NZ
      ANGLE(I)=3.1415926536*ANGLE(I)/180.
      THE(1,I)=COS(ANGLE(I))
      THE(1,2I)=SIN(ANGLE(I))
      THE(1,3I)=0.0
      S1=E(I)
      IF (NM1,EQ,0) E1=E11(1,I)*10.**6
      IF (MNC,EQ,0) GO TO 27
      DO 26 J=1,NMAT2
         DC 26 I=1,NZ
         E11(I,J)=E11(I,J)*10.**6
         E22(I,J)=E22(I,J)*10.**6
         GSH(I,J)=GSH(I,J)*10.**6
         IF (GSH(I,J),EQ,0.) GSH(I,J)=E11(I,J)/(2.*(1+ANU1(I,J)))
         IF (ANU2(I,J),EQ,0.) ANU2(I,J)=ANU1(I,J)/E11(I,J)
         CNU(I,J)=1.-ANU1(I,J)+ANU2(I,J)
         C1=E11(I,J)/CNU(I,J)
         C2=E22(I,J)/CNU(I,J)
         C12=C22*ANU1(I,J)
         C66=GSH(I,J)
         C16=0.0
         C26=0.0
         AEE(1,I,J)=C11/E1
         AEE(5,I,J)=C22/E1
         AEE(2,I,J)=AEE(5,I,J)*ANU1(I,J)
         AEE(9,I,J)=GSH(I,J)/E1
         AEE(4,I,J)=AEE(2,I,J)
         AEE(6,I,J)=0.0
         AEE(8,I,J)=0.0
         AEE(3,I,J)=0.0
         AEE(7,I,J)=0.0
         AM(I)=COS(ANGLE(I))
         AN(I)=SIN(ANGLE(I))
         S1(I)=AM(I)**4
         S2(I)=AM(I)**2*AN(I)**2
         S3(I)=AM(I)**3*AN(I)
         S4(I)=AM(I)**4
         S5(I)=AM(I)*AN(I)**3
         S6(I)=AM(I)**2
         S7(I)=AM(I)**2
         S8(I)=AM(I)*AN(I)
         L=1
         CC(1,L,J)=S1(L)*C11+2.*S2(L)*C12+4.*S3(L)*C16+S4(L)*C22+4.*S5(L)*C
            126+4.*S2(L)*C66
         CC(2,L,J)=S2(L)*C11+S4(L)*C12+2.*(S5(L)-S3(L))*C16+S2(L)*C
            122+2.*C3(I)-S5(L))*C26-4.*S2(L)*C66
         CC(3,L,J)=CC(2,L,J)
         CC(4,L,J)=-S3(L)*C11+S3(L)-S5(L)*C12+(S1(L)-3.*S2(L))*C16+S5(L)*
            1022+3.*S2(L)-S4(L))*C26+2.*(S3(L)-S5(L))*C66
         CC(7,L,J)=CC(3,L,J)
         CC(8,L,J)=S6(L)+C11+2.*S2(L)*C12+4.*S3(L)*C16+S4(L)*C22+4.*S5(L)*C

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230      126+4.*S2(L)*C66
      CC(6,L,J)=-S5(L)*C11+(S5(L)-S3(L))*C12+(3.*S2(L)-S4(L))*C16+S3(L)*
      1C22+(S1(L)-3.*S2(L))*C26+(S5(L)-S3(L))*2.*C66
      CC(8,L,J)=CC(6,L,J)
      CC(9,L,J)=S2(L)*C11-2.*S2(L)*C12+2.*(S5(L)-S3(L))*C16+S2(L)*C22+2.
      1*(S3(L)-S5(L))*C26+(S6(L)-S7(L))*2.*C66
235      CONTINUE
27      CONTINUE
27      IF (NSTART.EQ.1) GO TO 28
      IF (LSTCOL.GT.0) GO TO 29
28      CONTINUE
      CALL ANORM (A,AL,MEMBS,AMAX,NC,NZ,NCC)
      BASEAE=BASEA*E1
29      CONTINUE
      IF (INCHES.EQ.1) GO TO 31
      DO 30 I=1,JOINTS
240      X(I)=X(I)*12.0
      Z(I)=Z(I)*12.0
      Y(I)=Y(I)*12.0
30      CONTINUE
31      IF (KIPS.NE.1) GO TO 35
      DO 32 I=1,NN
      DO 32 J=1,LOADS
      FR(I,J)=1000.0*FR(I,J)
32      DO 33 I=1,NHAT1
      SSTR(I)=SSTR(I)*1000.
      SSTRC(I)=SSTRC(I)*1000.
      SSTRS(I)=SSTRS(I)*1000.
33      CONTINUE
      IF (NCRTIA.EQ.2) GO TO 35
      DO 34 I=1,NHAT2
      DO 34 K=1,NZ
      DO 34 J=1,5
      SSMAX(J,K,I)=SSMAX(J,K,I)*1000.
34      CONTINUE
35      CONTINUE
      SMAX(I)=SMAX(I)*E1
      BMIN(I)=BMIN(I)*E1
36      CALL POS (MEMBS,JOINTS,MV,MA,MB,MC,MD,KTYPE,ICOL,IDIAG,NONZRO)
      IF (NONZRO.LT.NMAX) GO TO 37
      WRITE (6,129) NONZRO
      GO TO 435
37      CONTINUE
      DO 38 I=1,NN
      ICOLS(I)=ICOL(I)
      IDIAGS(I)=IDIAG(I)
38      IF (LSTCOL.EQ.0) GO TO 41
      ZN=FLOAT(NZ)
      AZN=0.1/ZN
      DO 39 I=1,NC
      A(I)=0.1
      DO 39 J=1,NZ
      AL(I,J)=AZN
      KI=NC+1
      DO 40 I=K1,MEMBS
285      286

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10/29/76 12.20.38

FIN 4.5+414

PROGRAM OPTCONF 74/74 OPT=1

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40 A(I)=0.1
41 IF (KTYPE(I).EQ.2) A(I)=1.0
    CONTINUE
290 DO 42 I=1,NN
    DO 42 J=1,LOADS
42 DR(I,J)=0.0
    WLAST=0.0
    LAST=0
    KCOUNT=1
    NREQ=0
    WNLIS=1
    NSTBLTY=0
    MFORCE=0
    RAZSEA=0.0
    NPAGE=0
    WMIN=10.**20
    DO 43 I=1,4
    ZI(I)=0.
    MBR(I)=1+1
    MCC(I)=5
    ZI(5)=0.
    MAA(4)=1
    MBR(4)=4
    NZERO=0
    L=1
    NDU=0
    DO 45 I=1,NN
    ICOL(I)=ICOLS(I)
    IDIAG(I)=IDIAGS(I)
315 DO 46 I=1,NONZRO
    SK(I)=0
    AT=MESECOND(AAZ)
    WRITE (6,146) ATIME
    IF (NAREA.EQ.0) GO TO 48
    WRITE (6,144)
    WRITE (6,143) (A(I),I=1,MEMBS)
    WRITE (6,130)
    DO 47 J=1,NZ
325 WRITE (6,143) (AL(I,J),I=1,NC)
    WRITE (6,130)
    CONTINUE
47 CONTINUE
48 IF (NSTBLTY.GT.1) WRITE=1
    KNODE=4
    KNMAT(L)
    IF (KTYPE(L).LE.4) KNODE=KTYPE(L)
    CALL COORD (MA(L),MB(L),MC(L),MD(L),X,Y,Z,AA,XI,ETA,PL,KNODE,NZERO
    1,AR)
    IF (L.GT.NC) GO TO 49
    IF (NFIKER.EQ.1) FANGLE=ZANGLE(L)
    CALL SUBRT (X,Y,Z,ETA,AA,XI,ETA,PL,KNODE,NZERO
    1,AR)
335

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FTW 4.5+414

```

345      DO 51 I=1,12
      DO 51 J=1,12
      C(I,J)=0.0
      EK(I,J)=0.
      QUAD=0.
      51      DO 57 I=1,4
      CALL COORD (MAA(I),MBB(I),MCC(I),MD(I),XI,ETA,ZI,AAA,XXI,YII,PL,NT
      1H,NZERO,AB)
      DO 52 JK=1,12
      DO 52 IK=1,12
      52      EKK(IK,JK)=0.0
      DO 56 IL=1,NZ
      THK=AL(L,IL)
      DO 53 KL=1,3
      53      EO(KL)=THETA(IL,KL)
      JK=0
      DO 54 J1=1,3
      DO 54 K1=1,3
      JK=JK+1
      54      AE(J1,K1)=AEE(JK,I,IL,KM)
      CALL ELAS (AAA,AB,RE,EO,DA,EE)
      CALL COMP (EKL,THK,TRIANG,XXI,YII,EE)
      DO 55 JK=1,6
      55      EKK(IK,JK)=EKK(IK,JK)+EKL(IK,JK)
      56      CONTINUE
      QUAD=QUAD+TRIANG
      CALL TRNSFM (EKK,AAA,B,C,MTO,INT,NT4)
      CALL SUM (EK,C,MAA(I),MBB(I),MCC(I))
      57      CONTINUE
      NZERO=0
      CALL CONDONS (EK,EKK,MA(L),MB(L),MC(L),MD(L),NZERO)
      350      GO TO 66
      365      CONTINUE
      370      DO 59 I=1,3
      DO 59 J=1,3
      AC(I,J)=0.0
      AC(1,1)=1.0
      AC(2,2)=1.0
      DO 60 I=1,6
      DO 60 J=1,6
      EK(I,J)=0.0
      DO 64 I=1,NZ
      THK=AL(L,I)
      DO 61 K=1,3
      61      EO(KL)=THETA(I,K)
      JK=0
      DO 62 J1=1,3
      DO 62 K1=1,3
      JK=JK+1
      62      AE(J1,K1)=AEE(JK,I,KM)
      CALL ELAS (AC,AB,RE,EO,DA,EE)
      CALL COMP (EKL,THK,QUAD,XXI,ETA,EE)
      DO 63 KI=1,6
      DO 63 KJ=1,6
      63      EKK(KI,KJ)=EKK(KI,KJ)+EKL(KI,KJ)
      64      CONTINUE

```



```

400      GO TO 66
        CONTINUE
        CALL SCOMP (EX,A(L),QUAD,XI,ETA,GSHE(KM),E1)
        CALL CONDONS (EX,EKK,MA(L),MB(L),MC(L),MO(L),NTO)
        CONTINUE
        CALL TRNSFM (EX,AA,B,C,MM,KNODE,NTW)
        GO TO 63
    405      CONTINUE
        EBASE=KM)
        CALL ELSTIF (AA,B,C,A(L),MM,PL,EBAR,E1)
        QUAD=PL
    410      CONTINUE
        IF (WRITE.EQ.1) WRITE (6,131) ((C(I),J),J=1,8),I=1,8)
        CALL ASEHBL (SK,C,MA(L),MB(L),MC(L),MO(L),MM,IDIAG,KNODE,NTW)
        IF (L.GE.MEMB3) GO TO 69
        L=L+1
        GO TO 43
    415      CONTINUE
        IF (FORCE.EQ.0) GO TO 71
        DO 70 I=1,NONZRO
        70      SK(I)=SK(I)*BASEAE
        CALL MULT (SK,FR,DR,ICOL,IDIAG,MM,LOADS,NZK)
        CALL PRATOR (FR,DR,X,Y,Z,IBND,MM,MALLOADS,JOINTS,NPAGE,NB,NON,NZK)
        CALL AED (X,Y,Z,FR,IBND,NZK,LOADS,JOINTS,NB,MM)
        GO TO 135
    425      CONTINUE
        CALL BOUND (SK,IBND,MM,NB,IDIAG,ICOL)
        CALL REDUCE (FR,IBND,MM,NB,LOADS,NZK)
        CALL GAUSS (SK,FR,DR,ICOL,IDIAG,LOADS,MM,NZK,NZERO)
        ATIME=SECOND(AAZ)
        IF (LSTCOL.EQ.0) GO TO 72
        BASEAE=100.
        BASEAE=BASEAE/E1
    430      CONTINUE
        CALL RESTOR (FR,IBND,MM,NB,LOADS,NZK)
        CALL RESTOR (DR,IBND,MM,NB,LOADS,NZK)
        NFULL=0
        IF (KCOUNT.GE.LLSTCOL.AND.NSTBLTY.EQ.0.AND.LAST.EQ.0) NFULL=1
        IF (INENG.EQ.4) NFULL=0
        IF (KCOUNT.GT.LLSTCOL.AND.NFULL.EQ.1) NNEQ=NNEQ+1
        NCHK=0
        NOLIS=0
        CALL ME3 (X,Y,Z,MA,MB,MC,MD,KTYPE,A,DR,STPENG,AI,ELENTM,SSTHK,ZAN
    435      1GLE,STNG,ENGL,MAT,SSTRT,SSTRC,SSTKS,GSHE,AWIDE,AWIDE,ANEQ,MANEG,P
        2ECT,NC,IZ,LOADS,NCC,NZK,NON)
        IF (MCOUNT.EQ.0) GO TO 73
        IF (LAST.EQ.1) GO TO 73
        WRITE (6,132) NALLIS
        PBASE=0.001*BASEAE
        OBASE=ABS(BAZSER-BASEAE)
    440      WRITE (6,133) BAZSER,OBASE
        IF (OBASE.LE.PBASE) GO TO 73
        IF (NALLIS.GT.2) GO TO 73
        BAZSER=1*BASEAE

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460      73  CONTINUE
          RAZSEA=0
          NANLIS=1
          MSKIN=0.0
          IF (NO.EQ.0) GO TO 75
          DO 74 L=1,NC
            KM=NMAT(L)
            DO 74 J=1,NZ
              SPWT=SPWT+(J,KM)
              MSKIN=MSKIN+AL(L,J)*BASEA*ELENTHT(L)*SPWT
            CONTINUE
          MSHEAR=0.0
          MPOST=0.0
          KI=NC+1
          DO 76 L=X1,MEMBS
            KM=NMAT(L)
            SPWT=SPWT+(J,KM)
            WELE=A(L)*BASEA*ELENTHT(L)*SPWT+(J,KM)
            IF (KTYPE(L).EQ.5) MSHEAR=MSHEAR+WELE
            IF (KTYPE(L).EQ.2) MPOST=MPOST+WELE
          CONTINUE
          WEIGHT=MSKIN+MSHEAR+MPOST
          WRITE (6,145) MSKIN,MSHEAR,MPOST,WEIGHT
          WRITE (6,134) LSKIN,ABAS,LINT,CBAS
          ABAS=ABAS
          WRITE (6,142) KSTR,LOADS,KCOUNT,NNEQ,NSTBLTY
          IF (LAST.EQ.1) WRITE (6,135) NNCYCL
          DO 77 I=1,NN
            DO 77 J=1,LOADS
              DR(I,J)=DR(I,J)/BASEA
              IF (KCOUNT.EQ.LISS) GO TO 78
              IF (INSTALTY.GE.1) GO TO 95
              IF (LSTOCL.EQ.0) GO TO 95
              IF (KCOUNT.EQ.1) GO TO 85
              IF (LAST.GE.1) GO TO 95
              IF (KCOUNT.GT.LSTOCL) GO TO 76
              GO TO 85
            CONTINUE
          IF (WEIGHT.GT.WMIN) GO TO 81
          NNCYCL=KCOUNT
          WMIN=WEIGHT
          9LAST=BASEA
          DO 79 I=1,NC
            DO 79 J=1,NZ
              ADEL(I,J)=AL(I,J)
            DO 80 L=1,NI
              KI=L+NC
              ADEL(L)=A(KI)
            GO TO 84
          500
          80
          505

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83 CONTINUE
84 CALL ANORM (A,AL,MEMBS,AMAX,NC,NZ,NCC)
   KCOUNT=KCOUNT+1
   IF (KCOUNT.EQ.ULLSTCL) GO TO 44
   LAST=LAST+1
   BASEAE=BLAST
   BASEA=BLAST/E1
   GO TO 84

85 CONTINUE
   IF (WEIGHT.LT.WMIN) NNCYCL=KCOUNT
   IF (LINK.EQ.1) CALL ALINK (A,AL,STRENG,STNG,ELENTN,NELEM,NLINK,NZ,
1INC,NSKIN,NINT,NSTBLTY,NCC)
   IF (NC.EQ.0) GO TO 87
   DO 86 L=1,NC
     KM=NMAT(L)
     SPMT=SPATC(J,KM)
     IF (WEIGHT.LT.WMIN) AAEL(L,J)=AL(L,J)
     IF (WEIGHT.LT.WMIN) BLAST=BASEAE
     IF (NFULL.EQ.1) AL(L,J)=STRENG(L,J)
     IF (NFULL.EQ.1) GO TO 86
     STRENG(L,J)=STRENG(L,J)/SPMT
     AL(L,J)=AL(L,J)*SORT(STRENG(L,J))
   CONTINUE
   DO 88 L=1,N1
     K1=L+NC
     IF (WEIGHT.LT.WMIN) AAEL(L)=A(K1)
     KM=NMAT(K1)
     SPMT=SPATC(KM)
     IF (NFULL.EQ.1) A(K1)=STNG(L)
     IF (NFULL.EQ.1) GO TO 88
     STNG(L)=STNG(L)/SPMT
     A(K1)=A(K1)*SORT(STNG(L))
   CONTINUE
   IF (WEIGHT.LT.WMIN) WMIN=WEIGHT
   NDUM=0
   CONTINUE
   DO 94 K=1,2
     CALL ANORM (A,AL,MEMBS,AMAX,NC,NZ,NCC)
     IF (NC.EQ.0) GO TO 91
     DO 90 I=1,NC
       AMIN=A(I)
       KM=NMAT(I)
       IF (AMIN*BASEAE.GT.BMAX(I)) A(I)=BMAX(I)/BASEAE
       IF (AMIN*BASEAE.LT.BMIN(I)) A(I)=BMIN(I)/BASEAE
       DO 90 J=1,NZ
         AL(I,J)=AL(I,J)*A(I)/AMIN
         THK1=THMIN(J,KM)
         IF (AL(I,J)*BASEA.LT.THK1) AL(I,J)=THK1/BASEA
       CONTINUE
       KI=NC+1
       DO 93 I=K1,MEMBS

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92      IF (LAST.GE.1) GO TO 92
        IF (NANEG(KK).GT.0) A(I)=ANEG(KK)/(AABAS*E1)
        CONTINUE
575      IF (A(I)*BASEAE.LT.BMIN(I)) A(I)=BMIN(I)/BASEAE
        IF (A(I)*BASEAE.GT.BMAX(I)) A(I)=BMAX(I)/BASEAE
        CONTINUE
93      CALL ANJRM (A,AL,HEMBS,AMAX,NC,NZ,NCC)
        CONTINUE
94      IF (LAST.GE.1) GO TO 44
        IF (NOLIS.EQ.1) GO TO 44
        IF (NOUT.EQ.1) GO TO 97
        WLAST=HEIGHT
        KCOUNT=KCOUNT+1
        GO TO 44
95      CONTINUE
        LAST=0
        IF (NSTBLTY.NE.0.AND.NSTBLTY.NE.LNSB) NZERO=1
        NCHK=1
        NFULL=0
96      CALL MEMB (X,Y,Z,MA,MB,MC,MD,KTYPE,A,DR,STRENG,AL,ELENTN,STHK,ZAN
        1GLE,STNG,ENGL,NMAT,SSRT,SSTRC,SSTRS,GSHE,AMIDE,3WIDE,ANEG,NANEG,P
        2ECT,NC,NZ,LOADS,NCC,NZX,NZERO)
        IF (NSTBLTY.GE.LNSB) GO TO 98
        IF (NSTBLTY.EQ.0) GO TO 98
        IF (LINK.EQ.1) CALL ALINK (A,AL,STRENG,STNG,ELENTN,NELEM,NLINK,NZ,
        1NC,MSKIN,NINT,NSTELTY,NCC)
        K1=NC+1
        NOLIS=0
        NDUH=1
        NHZ=NC+1
        DO 96 L=NMZ,HEMBS
        KK=L-NC
        IF (NANEG(KK).GT.0) A(L)=ANEG(KK)/(AABAS*E1)
        IF (A(L)*BASEAE.LT.BMIN(L)) A(L)=BMIN(L)/BASEAE
        IF (A(L)*BASEAE.GT.BMAX(L)) A(L)=BMAX(L)/BASEAE
        CONTINUE
        GO TO 83
97      CONTINUE
        NSTBLTY=NSTBLTY+1
        IF (NSTBLTY.EQ.LNSB) LAST=1
        IF (NSTBLTY.LE.LNSB) GO TO 44
        CONTINUE
98      CALL PRTOR (FR,DR,X,Y,Z,IBND,NN,MM,LOADS,JOINTS,NPAGE,NB,NZERO,NZ
        1K)
        IF (NC.EQ.0) GO TO 101
        DO 99 I=1,NC
        KM=NMAT(I)
        DO 99 J=1,NZ
        THKK=THEK(J,K4)
        ANLRS=AL(I,J)*BASEAE/THKK
        NLYRS(I,J)=IFIX(ANLRS)+1
        CONTINUE
        WRITE (5,136)
        K1=1
        K2=5
        K3=NC/5+1
        DO 100 JKK=1,K3

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630      IF (K1,GT,NC) GO TO 100
        IF (K2,GT,NC) K2=NC
        IF (N2,EQ,4) WRITE (6,137) ((I,(NLYRS(I,J),J=1,NZ)),I=K1,K2)
        IF (N2,EQ,3) WRITE (6,138) ((I,(NLYRS(I,J),J=1,NZ)),I=K1,K2)
        K1=K1+5
        K2=K2+5
100      CONTINUE
101      CONTINUE
        DO 102 I=1,MEMBS
          A(I)=A(I)/BASEA
102      WRITE (6,139)
          WRITE (6,140)
          K1=1
          K2=10
          K3=MEMBS/10+1
          DO 103 J=1,K3
            IF (K1,GT,MEMBS) GO TO 103
            IF (K2,GT,MEMBS) K2=MEMBS
            K4=J-1
            WRITE (6,141) K4, (A(I),I=K1,K2),K2
            K1=K1+10
            K2=K2+10
            CONTINUE
103      DO 104 I=1,MEMBS
          A(I)=A(I)/BASEA
          LAST=0
          NSTBLTY=NSTBLTY+1
          IF (NSTBLTY,EQ,1) GO TO 44
          IF (NSTBLTY,GT,4) NSTBLTY=1
          IF (NFORCE,EQ,1) GO TO 44
105      CONTINUE
          IF (KSTR,EQ,NSTR) GO TO 106
          KSTR=KSTR+1
          GO TO 1
106      CONTINUE
          ATIME=SECOND(AZT)
          WRITE (6,146) ATIME
          STOP
C
107      FORMAT (1HT)
108      FORMAT (1HT)
109      FORMAT (8A10)
110      FORMAT (5X,11HPROBLEM4 NO=,I5,5X,8A10)
111      FORMAT (5X,16I5)
112      FORMAT (5X,8E10.4)
113      FORMAT (4F20.4)
114      FORMAT (5X,4F20.4)
115      FORMAT (8E10.4)
116      FORMAT (5X,8E10.4)
117      FORMAT (5F15.4)
118      FORMAT (5X,5F15.4)
119      FORMAT (3E10.4)
120      FORMAT (5X,3E10.4)
121      FORMAT (15,I3,I2,4I5,5E10.4)
122      FORMAT (5X,I5,I3,I2,4I5,5E10.4)
123      FORMAT (5X,I5,3E10.4)
124      FORMAT (15,3E10.4)

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685      FORMAT (1615)
686      FORMAT (8E10.4)
687      FORMAT (5X,3(E10.4,215))
688      FORMAT (3(E10.4,215))
689      FORMAT (//10X,44HINSUFFICIENT CONE TO STORE STIFFNESS MATRIX,.25H
690      1JIMENSION OF SK SHOULD BE,115)
691      FORMAT (3X,/)
692      FORMAT (2X,4E12.4,5X,2HCC)
693      FORMAT (1H,10X,12HANALYSIS NO=,15)
694      FORMAT (1H,6HBASE=,1PE15.5,11HDIFFERENCE=,1PE15.5)
695      FORMAT (10X,9HELE SKIN=,15,5X,6HBASE=,1E15.4,10X,11HELE STRUCT=,1
696      15,10X,6HBASE=,1E15.4)
697      FORMAT (30X,21HMINIMUM WEIGHT CYCLE=,15)
698      FORMAT (//5X,39HNUMBER OF LAMINAE IN COMPOSITE ELEMENTS)
699      FORMAT (2X,(5(1H,13,1H),415,2X)))
700      FORMAT (2X,(5(1H,13,1H),315,2X)))
701      FORMAT (//5X,27HTOTAL THICKNESS OF ELEMENTS)
702      FORMAT (1H,18X,1H,13X,1H,2,9X,1H,3,9X,1H,4,9X,1H,5,9X,1H,6,9X,1H,7,9X,1
703      1H,8,9X,1H,9,9X,1H,10,/)
704      FORMAT (3X,15,4X,10E10.3,3X,15)
705      FORMAT (5X,13HSTRUCTURE NO=,15,9X,12HNO OF LOADS=,15,10X,9HCYCLE N
706      10,15,10X,12HEQ CYCLE NO=,15,10X,8HSTBLTY=,15)
707      FORMAT (1H,10F12.6)
708      FORMAT (//5X,25HRELATIVE AREAS OF MEMBERS)
709      FORMAT (//,5X,12HWEIGHT-SKIN=,1PE14.6,5X,16HWEIGHT-S-PANELS=,1PE14
710      1.6,5X,13HWEIGHT-POSTS=,1PE14.6,5X,13HTOTAL-WEIGHT=,1PE14.6)
711      FORMAT (//2X,22HTIME USED IN SECONDS =,F10.4)
712      END
713

```

```

1 SUBROUTINE MEMB (X,Y,Z,MA,MB,MC,MD,KTYPE,TH,OR,STRENG,AL,ELENGH,SS
  1THK,ZANGLE,STNG,ENGL,NMAT,SSIRT,SSIRC,SSIRS,GSHE,AWIDE,BWIDE,ANEG,
  2ANEG,PECT,NC,NZ,LO,NCC,NN,NOPRNT)
  3COMMON /AA/ NEF,NAV,NFUL,NCHK,NCRIT,ANSTBLTY,NLO,NFI,NZEO,NSTRN,
  4MEMB
  51NFOR,NEIG,NBLNCE,LAST,NLSD,ABAS,CBAS,LSKIN,ALINT,MEMBS,MM,E1,BASEAE
  6MEMB
  72BASEA,NFIBER,LSTCCL,LCHK
  8COMMON /BB/ MAA,MBB,MCC,MEK,EKK,B,C,XI,ETA,ZI,XI,YII,XD,YD,ZD,SSMA
  9MEMB
  10IX,THE,THETA,EKL,E,AAE,CC,AA,AAA,AAE,AB,DA,FE,EO
  11MEMB
  12C DIMENSIONS OF THE VARIABLES NOT MENTIONED IN THE COMMENT CARDS
  13C ARE UNCHANGED
  14C EDR,EDOR,COR,--(12,LOADS) NEEDS CHANGE IF LOADS IS GREATER THAN 10
  15C AXI,AYI,AXY,--(1,LOADS)
  16C SX,EX,EY,ELENG,--(LOADS)
  17C SSX,THE,THETA,E,AAE,CC,-- DIMENSIONS CONSISTENT WITH THOSE IN MAIN
  18C SSX,SSY,SSXY,EFFSTR,ASX,ASY,ASXY,PA,PP,XY,BSX,BSY,BSXY,EXFI,
  19C EYFI,EXYFI,--(4,LOADS,NZ)--NEEDS CHANGE ONLY IF LOADS IS GREATER
  20C THAN 10 AND NZ GREATER THAN 4
  21C ENGSTR,--(LOADS,NZ)$$BL,--(1,NZ)$$SSTHK,PROT,--(NZ)$$S
  22DIMENSION X(1),Y(1),Z(1),MA(1),MB(1),MC(1),MD(1),KTYPE(1),
  23TH(1),OR(NN,LO),STRENG(NC,NZ),AL(NCC,NZ),ELENGH(1),SSIRK(NCC
  24MEMB
  253HE(1),AWIDE(1),BWIDE(1),PECT(1),NMAT(1),SSIRT(1),SSIRS(1),GS
  26MEMB
  27DIMENSION AA(3,3),AB(3),EK(12,12),XI(5),ETA(5),C(12,12),B(12
  281,12),XX(3),YY(3),XXI(5),YII(5),ZI(5),EKK(12,12),AAA(3,3),M
  29MEMB
  302AA(4),MBB(4),MCC(4),AE(3,3),EE(3,3),EO(3),EKL(12,12),DA(3,3)
  31MEMB
  323),XD(3),YD(3),ZD(3),NCON(4)
  33MEMB
  34DIMENSION EDR(12,10),EDOR(12,10),COR(12,10),AX(1,10),AY(1,10),
  351AXY(1,10),SX(10),EX(10),EY(10),EXY(10),ELENG(10)
  36MEMB
  37DIMENSION SSX(5,4,4),THE(4,3),THETA(4,3),E(4),AAE(9,4,4),CC
  381(9,4,4)
  39MEMB
  40DIMENSION SSX(4,10,4),SSY(4,10,4),SSXY(4,10,4),EFFSTR(4,10,4),
  41ENGSTR(1,4),ASX(4,10,4),ASY(4,10,4),ASXY(4,10,4),PX(4,10,4),
  422PY(4,10,4),PXY(4,10,4),BSX(4,10,4),BSY(4,10,4),BSXY(4,10,4),E
  433XFI(4,10,4),EYFI(4,10,4),EXYFI(4,10,4),BL(1,4),STHK(4),PECT(4
  444),TRANG(4),AD(3,3),A(3,3)
  45NZERO=0
  46LLDS=LOADS
  47IF (NENG.EQ.3.OR.NENG.EQ.4) LLDS=1
  48NI=MEMBS-NC
  49IL1=NZ-1
  50TL2=NZ
  51LOADS=LO
  52NON=1
  53NTO=2
  54NTH=3
  55CBASE=0.0
  56CBAS=0.0
  57TBAS=BASEA
  58TBASE=BASEAE
  59ABASE=0.0
  60ABAS=0.0
  61NTM=12
  62DO 1 I=1,NC
  63DO 1 J=1,NZ
  64STPENG(I,J)=0.0
  65DO 2 I=1,NI
  66STNG(I)=0.0
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60      DO 106 L=1, MEMBS
          KM=NMAT(L)
          BASEA=TJAS
          BASEAE=IASE
          KNODE=4
          IF (KTYPE(L).LE.4) KNODE=KTYPE(L)
          IF (NDPNT.EQ.1) GO TO 3
          IF (L.EQ.1) WRITE (6,126)
          CONTINUE
          NCON(1)=MM*(MA(L)-1)+1
          NCON(2)=MM*(MB(L)-1)+1
          IF (KNODE.EQ.3) NCON(3)=MM*(MC(L)-1)+1
          IF (KNODE.EQ.4) NCON(4)=MM*(MD(L)-1)+1
          NDSP=1
          IF (KNODE.EQ.2) NDSP=2
          CALL GORD (MA(L),MB(L),MC(L),MD(L),X,Y,Z,AA,XI,ETA,PL,KNODE,NZERO
1,AR)
          IF (L.GT.NC) GO TO 4
          IF (NFIBER.EQ.1) PANGLE=ZANGLE(L)
          CALL AVECT (XD,YD,ZD,THE,AA,THETA,NZ,PANGLE,NFIBER)
          CONTINUE
          DO 5 J=1,NZ
            PROT(J)=0.0
          DO 6 J=1,LOADS
            DO 6 JL=1,NZ
              ENGST(J,JL)=0.0
            DO 8 K=1,LOADS
              KH=1
              DO 8 KK=1,KNODE
                DO 8 I=1,NDSP
                  KX=NCON(KK)
                  FDR(KH,K)=0
                  DO 7 J=1,MM
                    EDF(KH,K)=EDR(KH,K)*AA(I,J)*DR(KX,K)
                  KX=KX+1
                  KH=KH+1
                DO 10 K=1,LOADS
                  LK=1
                  DO 10 KK=1,KNODE
                    KX=NCON(KK)
                    DO 9 J=1,MM
                      CDR(LK,K)=DR(KX,K)
                      LK=LK+1
                      KX=KX+1
                    CONTINUE
                  IF (KTYPE(L).EQ.2) GO TO 81
                  IF (KTYPE(L).EQ.5) GO TO 71
                  DO 11 I=1,12
                    DO 11 J=1,12
                      EX(I,J)=0.
                    QUAD=0.
                  NT=4
                  IF (KTYPE(L).EQ.3) NT=1
                  NT=NT
                  DO 20 I=1,NT
                    IF (NT.EQ.4) GO TO 13
                    DO 12 J=1,3

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SUBROUTINE MEMB

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115      XXI(J1)=XI(J1)
      YII(J1)=ETA(J1)
      DO 12 I1=1,3
      AAA(I1,J1)=0.0
12      CONTINUE
      AAA(I1,J1)=1.0
      AAA(2,2)=1.0
      GO TO 14
13      CONTINUE
      CALL COORD (AAA(I),MBB(I),MCC(I),XI,ETA,ZI,AAA,XXI,YII,PL,NT
      14      14,NZERO,AB)
      CONTINUE
      DO 15 JK=1,6
      DO 15 IK=1,6
      FKK(IK,JK)=0.0
15      FKK(IK,JK)=0.0
      THK=AL(L,IL)
      DO 16 KL=1,3
      EOKL)=THETA(IL,KL)
      JK=0
      DO 17 J1=1,3
      DO 17 K1=1,3
      JK=JK+1
      AE(J1,K1)=AEE(JK,IL,KM)
      CALL ELAS (AAA,AB,AE,EO,DA,EE)
      CALL COMP (EKL,THK,TRIANG,XXI,YII,EE)
      DO 18 IK=1,6
      DO 18 JK=1,6
      EKK(IK,JK)=EKK(IK,JK)+EKL(IK,JK)
18      CONTINUE
      TRIANG=TRIANG
      QUAD=QUAD+TRIANG
      IF (NT.EQ.1) GO TO 20
      CALL TRAFM (EKK,AAA,B,C,NTQ,NTH,NTM)
      CALL SU1 (EK,C,MAA(I),MBB(I),MCC(I))
20      CONTINUE
      ELENTH(L)=QUAD
      IF (NT.EQ.1) GO TO 25
      DO 21 I1=1,10
      DO 21 J1=1,10
      EKL(I1,J1)=EKL(I1,J1)
      NON=1
      CALL COMONS (EK,EKK,MA(L),MB(L),MC(L),MD(L),NON)
      KX=0
      DO 23 I=9,10
      KX=KX+1
      EDR(I,K)=0.
      DO 22 J=1,8
      EDR(I,K)=EDR(I,K)+EKK(KX,J)*EDR(J,K)
      EDR(I,K)=EDR(I,K)
22      CONTINUE
      DO 24 K=1,LOADS
      EDR(5,K)=EDR(9,K)
      EDR(6,K)=EDR(10,K)
      XX(3)=XI(5)
      YY(3)=ETA(5)
24      YY(3)=ETA(5)

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230      DO 39 J=1,3
231      ADIK(J)=DA(IK,J)
232      CONTINUE
233      CALL STRES (EE,EI,EX,EY,ESSX,SSY,SSXY,ASX,ASY,ASXY,EXFI,EYFI,EX
234      YFI,BSX,BSY,BSXY,QA,AD,LOADS,I,IL)
235      CONTINUE
236      IF (NT.EQ.1) GO TO 43
237      CALL AVSTRS (EXFI,TRANG,QUAD,LOADS,NZ)
238      CALL AVSTRS (EYFI,TRANG,QUAD,LOADS,NZ)
239      CALL AVSTRS (ESSX,TRANG,QUAD,LOADS,NZ)
240      CALL AVSTRS (SSY,TRANG,QUAD,LOADS,NZ)
241      CALL AVSTRS (SSXY,TRANG,QUAD,LOADS,NZ)
242      CALL AVSTRS (ASX,TRANG,QUAD,LOADS,NZ)
243      CALL AVSTRS (ASY,TRANG,QUAD,LOADS,NZ)
244      CALL AVSTRS (BSX,TRANG,QUAD,LOADS,NZ)
245      CALL AVSTRS (BSY,TRANG,QUAD,LOADS,NZ)
246      CALL AVSTRS (BSXY,TRANG,QUAD,LOADS,NZ)
247      NT=1
248      TRI=QUAD
249      TRANG(1)=QUAD
250      CONTINUE
251      IF (NCRTIA.NE.2) CALL ENGS (SSMAX,BSX,BSY,BSXY,PX,PY,PXY,EXFI,EYFI
252      1,EXYFI,ENGSTR,EFFSTR,LOADS,NZ,KM,NCRTIA,NENG,NEF)
253      IF (NCRTIA.EQ.2) CALL ENGS (SSMAX,EXFI,EYFI,EXYFI,PX,PY,PXY,BSX,BS
254      1,Y,BSXY,ENGSTR,EFFSTR,LOADS,NZ,KM,NCRTIA,NENG,NEF)
255      DO 44 J=1,NZ
256      PRCI(J)=AL(L,J)/TH(L)
257      NTRI=NT
258      IF (LCHECK.EQ.0.AND.LSTCCL.EQ.0) GO TO 54
259      IF (NCRTIA.EQ.1.OR.NCRTIA.EQ.2) CALL SUGD (PX,PY,PXY,LOADS,BASEAE,
260      1,LL00,LCXI,LLYR,NZ,STHK,SSMAX)
261      IF (NCRTIA.EQ.3.OR.NCRTIA.EQ.4) CALL SHILL (EFFSTR,LOADS,BASEAE,LL
262      100,LCXI,LLYR,NZ,STHK)
263      BASEAE=BASEAE/E1
264      RBASEAE=BASEAE/TBAS
265      IF (LCHECK.GT.0.AND.RBASEAE.GT.1) WRITE (6,107) L,RBASEAE,LL00,LCXI,L
266      1LYR
267      IF (LSTCCL.EQ.0) GO TO 54
268      DO 45 J=1,NZ
269      SSTHK(L,J)=STHK(J)*AL(L,J)
270      IF (NENG.EQ.3) GO TO 46
271      IF (NENG.EQ.4) GO TO 49
272      GO TO 51
273      DO 48 II=1,NZ
274      AMAX=0.0
275      DO 47 KK=1,LOADS
276      IF (AMAX.LT.ENGSTR(KK,II)) AMAX=ENGSTR(KK,II)
277      CONTINUE
278      ENGSTR(1,II)=AMAX
279      GO TO 51
280      GO TO 51
281      DO 50 J=1,NZ
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      IF (NBLDE.EQ.0) GO TO 53
      IF (IL1.EQ.0) GO TO 53
      DO 52 J=1,LOS
      ENGSTR(J,IL1)=(ENGSTR(J,IL1)+ENGSTR(J,IL2))/2.
      ENGSTR(J,IL2)=ENGSTR(J,IL1)
      CONTINUE
      CONTINUE
      IF (LAST.EQ.0) GO TO 54
      WRITE (6,107) L,BASEA,LLD,ALCH1,LLYR,(PROT(J),J=1,NZ)
      CONTINUE
      IF (NORNT.EQ.1) GO TO 50
      CALL RFORCE (PROT,AX,AY,ASX,ASY,ASXY,NZ,LOADS)
      IF (KTYPE(L).EQ.3) GO TO 55
      NTH=3
      CALL COJONS (EKL,EKK,MA(L),MB(L),MC(L),MD(L),NTH)
      CONTINUE
      NT1=2*KTYPE(L)
      DO 56 I=1,NT1
      DO 56 J=1,NT1
      IF (KTYPE(L).EQ.3) EK(I,J)=EK(I,J)*BASEAE
      IF (KTYPE(L).EQ.4) EK(I,J)=EK(L,I,J)*BASEAE
      CONTINUE
      IF (NFOR.EQ.1) GO TO 59
      NTH=12
      CALL TRNSFM (EK,AA,AB,C,MP,KNODE,NTW)
      NT1=MM*KNODE
      DO 57 I=1,NT1
      DO 57 J=1,NT1
      EK(I,J)=C(I,J)
      CONTINUE
      DO 58 K=1,LOADS
      DO 58 I=1,NT1
      EDR(I,K)=CDR(I,K)
      CONTINUE
      CONTINUE
      DO 61 K=1,LOADS
      DO 60 I=1,NT1
      EDR(I,K)=0.0
      DO 60 J=1,NT1
      EDR(I,K)=EDR(I,K)+EK(I,J)*EDR(J,K)
      CONTINUE
      CONTINUE
      WRITE (6,110)
      9L(1,1)=AL(L,I)*BASEA
      THKK=TH(L)*BASEA
      WRITE (6,111) L,MA(L),MB(L),MC(L),MD(L),TRI,THKK,(BL(I),I=1,NZ)
      IF (NANV.EQ.0) GO TO 63
      WRITE (6,112)
      WRITE (6,113) ((J,AX(I,J),AY(I,J),ASX(I,J),J=1,LOADS)
      CONTINUE
      IF (NLC.EQ.0) GO TO 64
      WRITE (6,114)
      WRITE (6,115) (J,IL,SSX(I,J,IL),SSY(I,J,IL),SSXY(I,J,IL),IL=1,NZ)
      1,J=1,LOADS)
  
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345      WRITE (6,116)
      WRITE (6,115) (J,(IL,BSX(1,J,IL),BSY(1,J,IL),BSXY(1,J,IL),IL=1,NZ)
      1,J=1,LOADS)
      CONTINUE
      IF (NZE/EQ.0) GO TO 66
      WRITE (6,117)
      WRITE (6,115) (J,(IL,BSX(1,J,IL),BSY(1,J,IL),BSXY(1,J,IL),IL=1,NZ)
      1,J=1,LOADS)
      CONTINUE
      IF (NSTN/EQ.0) GO TO 67
      WRITE (6,118)
      WRITE (6,115) (J,(IL,BSX(1,J,IL),BSY(1,J,IL),BSXY(1,J,IL),IL=1,
      1,NZ),J=1,LOADS)
      CONTINUE
      IF (NEF/EQ.0) GO TO 68
      WRITE (6,119)
      WRITE (6,115) (J,(IL,BSX(1,J,IL),BSY(1,J,IL),BSXY(1,J,IL),IL=1,LOADS)
      CONTINUE
      IF (INFOX/EQ.0) GO TO 70
      WRITE (6,120)
      DO 69 J=1,LOADS
      WRITE (6,109) J,(EODX(1,J),I=1,NT1)
      CONTINUE
      GO TO 90
      69      CONTINUE
      70      CALL SSYS (EQ,XI,ETA,TH(L),DET,SK,GSHE(KM),LOADS,ELEENG)
      ELENGH(L)=DET
      IF (LCHEK/EQ.0.AND.LSTCOL/EQ.0) GO TO 72
      CALL SURFACE (SK,PASEAF,SSTRS(KM),SSTRS(KM),LOADS,STHK,LLOD)
      BASEA=BASEAF/E1
      PBASEA=BASEA/THAS
      IF (LCHEK/GT.0.AND.PBASEA/GT.1) WRITE (6,125) L,XBASEA,LLOD
      IF (LSTCOL/EQ.0) GO TO 73
      IF (NCHS/EQ.1) GO TO 72
      K1=L-NC
      NANEG(K1)=0
      IF (BASEAF/GT.BASE) NANEG(K1)=1
      ANCO(K1)=STHK(1)*TH(L)
      CONTINUE
      IF (LAST/EQ.1) WRITE (6,125) L,BASEA,LLOD
      CONTINUE
      IF (NOPINT/EQ.1) GO TO 95
      THK=TH(L)*BASEA
      WRITE (6,121) L,MA(L),MB(L),MC(L),MD(L),DET,THK
      WRITE (6,122)
      WRITE (6,123) (I,SK(I),I=1,LOADS)
      IF (INFOX/EQ.0) GO TO 95
      CALL SCOMP (EK,TH(L),COAD,XI,ETA,GSHE(KM),E1)
      DO 74 I=1,8
      DO 74 J=1,8
      EK(I,J)=EK(I,J)*BASEA
      NTI=8
      IF (INFOX/EQ.1) GO TO 77
      NTH=12

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400      DO 75 I=1,NT1
          DO 75 J=1,NT1
              EK(I,J)=C(I,J)
              CONTINUE
          75      DO 76 K=1,LOADS
              DO 76 I=1,NT1
                  EDR(I,K)=COR(I,K)
                  CONTINUE
              76      DO 79 K=1,LOADS
                  DO 79 I=1,NT1
                      EDR(I,K)=0.0
                      DO 78 J=1,NT1
                          EDR(I,K)=EDR(I,K)+EK(I,J)*EDR(J,K)
                          CONTINUE
                      78      CONTINUE
                      WRITE (6,108)
                      DO 30 J=1,LOADS
                          WRITE (6,109) J,(EDR(I,J),I=1,NT1)
                          GO TO 95
                      80      CALL ELSTRS (EDR,AA,PL,SK,E(KM),LOADS,ELENG)
                          ELETH(L)=PL
                          IF (LCHEK.EQ.0.AND.LSTCCL.EQ.0) GO TO 83
                          CALL SURFACE (SX,BASEAE,SSIRT(KM),SSTRC(KM),LOADS,STNK,LLOD)
                          BASEAE=BASEAE/E1
                          RBASEAE=JBASEAE/13AS
                          IF (LCHEK.GT.0.AND.RBASEAE.GT.1) WRITE (6,125) L,RBASEAE,LLOD
                          IF (LSTCCL.EQ.0) GO TO 83
                          IF (NCHK.EQ.1) GO TO 82
                          K1=L-NC
                          NAMEG(K1)=0
                          IF (BASEAE.GT.ABASE) NAMEG(K1)=1
                          ANEG(K1)=STHK(I)*TH(L)
                          CONTINUE
                          82      CONTINUE
                          IF (LAST.EQ.1) WRITE (6,125) L,BASEAE,LLOD
                          IF (NOPRINT.EQ.1) GO TO 95
                          THKK=TH(L)*BASEAE
                          WRITE (6,124) L,MA(L),MB(L),PL,THKK
                          WRITE (6,122)
                          WRITE (6,123) (I,SK(I),I=1,LOADS)
                          IF (NFOR.EQ.0) GO TO 95
                          DO 84 J=1,LOADS
                              EDR(I,J)=(EDR(I,J)-EDR(2,J))*BASEAE/PL
                              EDR(I,J)=EDR(I,J)*TH(L)
                              EDR(I2,J)=-EDR(I,J)
                              NT1=2
                              IF (NFOR.EQ.1) GO TO 88
                              CALL ELSTIF (AA,B,C,TH(L),MM,PL,E(KM),E1)
                              NT1=MM*2
                              DO 85 I=1,NT1
                                  DO 85 J=1,NT1
                                      EK(I,J)=C(I,J)*BASEAE
                                      DO 87 K=1,LOADS
                                          DO 86 I=1,NT1
                                              EDR(I,K)=0.0

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87 CONTINUE
88 CONTINUE
   WRITE (6,120)
   DO 89 J=1,LOADS
89    WRITE (6,103) J, (E00R(I,J), I=1,NT1)
   GO TO 95
90 CONTINUE
   IF (ABASE.GT.BASEAE) GO TO 91
   LSKIN=L
   ABASE=BASEAE
   ABAS=BASEA
91 CONTINUE
   IF (NSTBLTY.EQ.0) GO TO 101
   IF (NSTBLTY.GE.LNSB) GO TO 106
   IF (NCHK.EQ.0) GO TO 101
   JK=0
   DO 93 I=1,3
   DO 93 J=1,3
   A(I,J)=0.0
   JK=JK+1
   THK=0.0
   DO 92 K=1,NZ
   A(I,J)=A(I,J)+CC(JK,K)*AL(L,K)*BASEA
   THK=AL(L,K)*BASEA+THK
92 CONTINUE
   A(I,J)=A(I,J)/THK
93 CONTINUE
   CALL 3FORCE (PRCT,AX,AY,AXY,ASX,ASY,ASXY,NZ,LOADS)
   CALL 3RUCKL (A,AX,AY,AXY,THK,LOADS,AWIDE(L),BWIDTH(L),TMAX)
   TOIFF=(TMAX-THK)/BASEA*0.7
   IF (TOIFF.LT.0.) GO TO 106
   DO 94 I=1,NZ
   AL(L,I)=AL(L,I)+PECT(I)*TOIFF
   GO TO 106
94 CONTINUE
95 IF (NSTBLTY.NE.0) GO TO 105
   DO 100 J=1,LDS
   GO TO (36,97,99,99), NENG
96 ENGST(J,1)=ELEENG(J)
   GO TO 100
97 CONTINUE
   IF (KTYPE(L).EQ.2) GO TO 98
   ENGST(J,1)=ELEENG(J)/(SSTRS(KK)**2/GSHE(KK))
   GO TO 100
98 SSTR1=SSTR(K*)
   IF (SX(IJ).LT.0.) SSTR1=SSTRC(KH)

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515      IF (NFULL.EQ.1) STRENG(L,K)=AL(L,K)*BASEA      MEMB 515
      CONTINUE      MEMB 516
516      GO TO 106      MEMB 517
      CONTINUE      MEMB 518
      K1=L-NC      MEMB 519
      DO 104 J=1,LLDS      MEMB 520
      STNG(K1)=STNG(K1)+KENGSTR(J,1)      MEMB 521
      IF (NFULL.EQ.1) STNG(K1)=TH(L)*BASEA      MEMB 522
      CONTINUE      MEMB 523
      CONTINUE      MEMB 524
      IF (CBASE.GT.BASEAE) GO TO 106      MEMB 525
      LINT=L      MEMB 526
      CBASE=BASEAE      MEMB 527
      CBAS=BASEA      MEMB 528
      CONTINUE      MEMB 529
      BASEAE=MAX1(CBASE,BASE)      MEMB 530
      BASEA=MAX1(CBAS,BAS)      MEMB 531
      IF (LSTCOL.EQ.0) BASEA=TBAS      MEMB 532
      IF (LSTCOL.EQ.0) BASEAE=TBASE      MEMB 533
      RETURN      MEMB 534
      C      MEMB 535
      FORMAT (2X,2HL=,15,2X,6HBASEA=,1E12,5,2X,4HL-C=,12,2X,4HCF1=,12,2X      MEMB 536
      1,4HLAR=,13,2X,6HPRNT=,9F8.3)      MEMB 537
      FORMAT (10X,12HMODAL FORCES)      MEMB 538
      106      FORMAT (1X,12,12E11.4)      MEMB 539
      110      FORMAT (1X,12,12E11.4)      MEMB 540
      111      FORMAT (2X,4HMEMB,15,2X,5HNODES,415,2X,4HAREA,1E10.4,2X,5HTHICK,1E      MEMB 541
      19.4,2X,12HAYER(THICK),4E10.4)      MEMB 542
      112      FORMAT (2X,3HFAVERAGE STRESSES (ZERO FIBER DIRECTION))      MEMB 543
      113      FORMAT (1X,12,2X,3E10.4)      MEMB 544
      114      FORMAT (2X,4HSTRESSES IN INDIVIDUAL LAYERS (LOCAL COORDINATES))      MEMB 545
      115      FORMAT (11, 12,4(12,1X,3E10.4)))      MEMB 546
      116      FORMAT (2X,4HSTRESSES IN INDIVIDUAL LAYERS(FIBER DIRECTIONS))      MEMB 547
      117      FORMAT (2X,5HSTRESSES IN INDIVIDUAL LAYERS(ZERO FIBER DIRECTION))      MEMB 548
      118      FORMAT (2X,4HSTRAINS IN INDIVIDUAL LAYERS(FIBER DIRECTION))      MEMB 549
      119      FORMAT (2X,16HEFFECTIVE STRESS)      MEMB 550
      120      FORMAT (2X,12HMODAL FORCES)      MEMB 551
      121      FORMAT (2X,4HMEMB,15,2X,5HNODES,415,2X,4HAREA,1E10.4,2X,5HTHICK,1E      MEMB 552
      110.4)      MEMB 553
      122      FORMAT (2X,4HSTRESSES)      MEMB 554
      123      FORMAT ((11, 3(12,1X,1E10.4)))      MEMB 555
      124      FORMAT (2X,4HMEMB,15,2X,5HNODES,215,2X,6HLENGTH,1E10.4,2X,4HAREA,1      MEMB 556
      1E10.4)      MEMB 557
      125      FORMAT (2X,2HL=,15,2X,6HBASEA=,1E12,5,2X,4HL-C=,12)      MEMB 558
      126      FORMAT (11H,43,25H*****S T R E S S E S*****///)      MEMB 559
      END      MEMB 560

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1 SUBROUTINE SUGD (FX, PY, PXY, LOADS, BASEAE, LLOD, LCRI, LLYR, NZ, STHK, SSM
    1AX)
2 SUGD
3 SUGD
4 SUGD
5 C DIMENSIONS OF FX, PY, PXY, SSMAX CONSISTENT WITH THOSE IN MEMB
6 C SUBROUTINE
7 DIMENSION PX(4,10,4), PY(4,10,4), PXY(4,10,4), SSMAX(5,4,4)
8 DIMENSION STHK(1)
9 LLOD=0
10 LLYR=0
11 LCRI=0
12 AMAX=0
13 STKSS=BASEAE
14 I=1
15 DO 4 J=1,LOADS
16 DO 4 K=1,NZ
17 IF (PX(I,J,K).LT.AMAX) GO TO 1
18 AMAX=PX(I,J,K)
19 LLOD=J
20 LLYR=K
21 LCRI=1
22 CONTINUE
23 IF (PY(I,J,K).LT.AMAX) GO TO 2
24 AMAX=PY(I,J,K)
25 LLOD=J
26 LLYR=K
27 LCRI=2
28 CONTINUE
29 IF (PXY(I,J,K).LT.AMAX) GO TO 3
30 AMAX=PXY(I,J,K)
31 LLOD=J
32 LLYR=K
33 LCRI=3
34 CONTINUE
35 IF (STKSS+GE.AMAX) GO TO 5
36 RATIO=AMAX/STKSS
37 BASEAE=BASEAE+RATIO
38 CONTINUE
39 DO 10 K=1,NZ
40 AMAX=0.
41 I=1
42 DO 9 J=1,LOADS
43 IF (PX(I,J,K).LT.AMAX) GO TO 6

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1      SUBROUTINE AVECT (X0,Y0,Z0,THE,AA,THETA,NZ,PANGLE,NFIBER)
C      DIMENSIONS OF THE, THETA, CONSISTENT WITH THOSE IN NERB SUBROUTINE
      DIMENSION X0(3), Y0(3), Z0(3), THE(4,3), AA(3,3), THETA(4,3)
5      EN=AA(1,2)*AA(2,3)-AA(2,2)*AA(1,3)
      EN=AA(1,3)*AA(2,1)-AA(2,2)*AA(1,1)
      EN=AA(1,1)*AA(2,2)-AA(1,2)*AA(2,1)
      IF (NFIBER.EQ.0) GO TO 1
      F1=COS(PANGLE)
      F2=SIN(PANGLE)
      F3=0.0
      G1=EN2*F3-EN3*F2
      G2=EN3*F1-EN1*F3
      ASSIGN=1.
      GO TO 2
15     1      CONTINUE
      F1=X0(2)-X0(1)
      F2=Y0(2)-Y0(1)
      F3=Z0(2)-Z0(1)
      AL=SQRT(F1**2+F2**2+F3**2)
      F1=F1/AL
      F2=F2/AL
      F3=F3/AL
      G1=X0(3)-X0(1)
      G2=Y0(3)-Y0(1)
      G3=Z0(3)-Z0(1)
      AL=SQRT(G1**2+G2**2+G3**2)
      G1=G1/AL
      G2=G2/AL
      G3=G3/AL
      FN=F2*G3-F3*G2
      FO=F3*G1-F1*G3
      FN=F1*G2-F2*G1
      TN=EN1*F1+EN2*F2+EN3*F3
      TD=EN1*FN+EN2*FN2+EN3*FN3
      TT=TN/10
      BF1=FN+FN1*TT
      BF2=FO+FN2*TT
      BF3=FN3+FN3*TT
      AL=SQRT(BF1**2+BF2**2+BF3**2)
      BF1=BF1/AL
      BF2=BF2/AL
      BF3=BF3/AL
      F1=AA(1,1)*BF1+AA(1,2)*BF2+AA(1,3)*BF3
      F2=AA(2,1)*BF1+AA(2,2)*BF2+AA(2,3)*BF3
      F3=EN1*BF1+EN2*BF2+EN3*BF3
      BG1=EN2*BF3-EN3*BF2
      BG2=EN3*BF1-EN1*BF3
      BG3=EN1*BF2-EN2*BF1
      AL=SQRT(BG1**2+BG2**2+BG3**2)
      BG1=BG1/AL
      BG2=BG2/AL
      BG3=BG3/AL
      G1=AA(1,1)*BG1+AA(1,2)*BG2+AA(1,3)*BG3
      G2=AA(2,1)*BG1+AA(2,2)*BG2+AA(2,3)*BG3
      G3=EN1*BG1+EN2*BG2+EN3*BG3
      ASSIGN=EN1*FN1+EN2*FN2+EN3*FN3
      ASSIGN=ASSIGN/ABS(ASSIGN)

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SUBROUTINE AVECT	74/74	OPT=1	FTN 4.5*414	10/29/76	12.20.38	PAGE	2
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60	2	CONTINUE					
		G1=G1*ASIGN					59
		G2=G2*ASIGN					60
		DO 3 N=1,NZ					61
		THETA(N,1)=F1*THE(N,1)*G1*THE(N,2)					62
		THETA(N,2)=F2*THE(N,1)*G2*THE(N,2)					63
	3	THETA(N,3)=0.0					64
65		RETURN					65
		END					66
							67

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1  SUBROUTINE SURFACE (SX,BASEAE,SSTT,SSTC,LOADS,STHK,LLOD)
C  SJA,--(LOADS) NEEDS TO BE GREATER THAN OR EQUAL TO LOADS IN
C  MAIN PROGRAM
5  DIMENSION SX(1), SJX(10), STHK(1)
    AMAX=0.0
    LLOD=0
    DO 1 I=1,LOADS
      STRESS=SSTT
      IF (SX(I).LT.0.) STRESS=SSTC
10     SJX(I)=ABS(SX(I))/STRESS
      STPS=BASEAE
      DO 2 I=1,LOADS
        IF (AMAX.GT.SJX(I)) GO TO 2
        AMAX=SJX(I)
        LLOD=I
2     CONTINUE
      STHK(1)=AMAX
      IF (STPS.GE.AMAX) RETURN
      RATIO=AMAX/STPS
      BASEAE=BASEAE*RATIO
      RETURN
      END
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1  SUBROUTINE COORD (MA,MB,MC,MD,X,Y,Z,AA,XI,ETA,AL,NND,NO,AB)
   DIMENSION X(1), Y(1), Z(1), AA(3,3), AB(3), XI(5), ETA(5)
   K1=MA
   K2=MB
   K3=MC
   K4=MD
   XCOMP=X(K2)-X(K1)
   YCOMP=Y(K2)-Y(K1)
   ZCOMP=Z(K2)-Z(K1)
   AL=SQRT(XCOMP**2+YCOMP**2+ZCOMP**2)
   AA(1,1)=XCOMP/AL
   AA(1,2)=YCOMP/AL
   AA(1,3)=ZCOMP/AL
   IF (NND.LT.3) RETURN
   XCOMP=X(K3)-X(K1)
   YCOMP=Y(K3)-Y(K1)
   ZCOMP=Z(K3)-Z(K1)
   AL=SQRT(XCOMP**2+YCOMP**2+ZCOMP**2)
   AB(1)=XCOMP/AL
   AB(2)=YCOMP/AL
   AB(3)=ZCOMP/AL
   AL=SQRT((AA(1,2)*AB(3)-AA(1,3)*AB(2))**2+(AA(1,3)*AB(1)-AA(1,1)*AB(3))**2+(AA(1,1)*AB(2)-AA(1,2)*AB(1))**2)
   AB(2,1)=(AA(1,3)*AB(2)-AA(1,1)*AB(3))/AL
   AB(3,1)=(AA(1,1)*AB(2)-AA(1,2)*AB(1))/AL
   AB(2,2)=(AA(1,1)*AB(2)+AA(1,2)*AB(1))/AL
   AB(3,2)=(AA(1,2)*AB(3)-AA(1,3)*AB(2))/AL
   AB(2,3)=(AA(1,1)*AB(3)+AA(1,2)*AB(2))/AL
   IF (NO.EQ.1) RETURN
   XI(1)=0.0
   ETA(1)=0.0
   XI(2)=(X(K2)-X(K1))*AA(1,1)+(Y(K2)-Y(K1))*AA(1,2)+(Z(K2)-Z(K1))*AA(1,3)
   XI(3)=(X(K3)-X(K1))*AA(1,1)+(Y(K3)-Y(K1))*AA(1,2)+(Z(K3)-Z(K1))*AA(1,3)
   XI(4)=(X(K4)-X(K1))*AA(1,1)+(Y(K4)-Y(K1))*AA(1,2)+(Z(K4)-Z(K1))*AA(1,3)
   XI(5)=(X(K5)-X(K1))*AA(1,1)+(Y(K5)-Y(K1))*AA(1,2)+(Z(K5)-Z(K1))*AA(1,3)
   IF (NND.LE.3) GO TO 1
   XI(4)=(X(K4)-X(K1))*AA(1,1)+(Y(K4)-Y(K1))*AA(1,2)+(Z(K4)-Z(K1))*AA(1,3)
   XI(5)=(X(K5)-X(K1))*AA(1,1)+(Y(K5)-Y(K1))*AA(1,2)+(Z(K5)-Z(K1))*AA(1,3)
   ETA(4)=(X(K4)-X(K1))*AA(2,1)+(Y(K4)-Y(K1))*AA(2,2)+(Z(K4)-Z(K1))*AA(2,3)
   ETA(5)=(X(K5)-X(K1))*AA(2,1)+(Y(K5)-Y(K1))*AA(2,2)+(Z(K5)-Z(K1))*AA(2,3)
   XI(5)=(XI(2)+XI(3)+XI(4))/4.0
   ETA(5)=(ETA(3)+ETA(4))/4.0
   CONTINUE
   RETURN
   END

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1  SUBROUTINE ELAS (AA,AB,EE,EO,DA,E)
   DIMENSION AA(3,3), AB(1), EE(3,3), EO(3), DA(3,3), EA(3,3), E(3,3)
   DIMENSION EP(3,3)
   EOK=SQRT(EO(1)**2+EO(2)**2+EO(3)**2)
   DO 1 I=1,3
     EO(I)=EO(I)/EOK
   EN1=AA(1,2)*AA(2,3)-AA(2,2)*AA(1,3)
   EN2=AA(1,3)*AA(2,1)-AA(1,1)*AA(2,3)
   FN3=AA(1,1)*AA(2,2)-AA(1,2)*AA(2,1)
   AL=SQRT(EN1**2+EN2**2+FN3**2)
   EN1=EN1/AL
   EN2=EN2/AL
   EN3=EN3/AL
   A1=AA(1,1)*EO(1)+AA(1,2)*EO(2)+AA(1,3)*EO(3)
   A2=AA(2,1)*EO(1)+AA(2,2)*EO(2)+AA(2,3)*EO(3)
   A3=EN1*EO(1)+EN2*EO(2)+EN3*EO(3)
   E1=EN2**3-EN3**2
   E2=EN3**3-EN1**2
   E3=EN1**3-EN2**2
   AL=SQRT(E1**2+E2**2+E3**2)
   E1=E1/AL
   E2=E2/AL
   E3=E3/AL
   AL11=A1
   AL12=A2
   AL21=A1
   AL22=A2
   AL31=A1
   AL32=A2
   AL41=A1
   AL42=A2
   AL51=A1
   AL52=A2
   AL61=A1
   AL62=A2
   AL71=A1
   AL72=A2
   AL81=A1
   AL82=A2
   AL91=A1
   AL92=A2
   AL101=A1
   AL102=A2
   AL111=A1
   AL112=A2
   AL121=A1
   AL122=A2
   AL131=A1
   AL132=A2
   AL141=A1
   AL142=A2
   AL151=A1
   AL152=A2
   AL161=A1
   AL162=A2
   AL171=A1
   AL172=A2
   AL181=A1
   AL182=A2
   AL191=A1
   AL192=A2
   AL201=A1
   AL202=A2
   AL211=A1
   AL212=A2
   AL221=A1
   AL222=A2
   AL231=A1
   AL232=A2
   AL241=A1
   AL242=A2
   AL251=A1
   AL252=A2
   AL261=A1
   AL262=A2
   AL271=A1
   AL272=A2
   AL281=A1
   AL282=A2
   AL291=A1
   AL292=A2
   AL301=A1
   AL302=A2
   AL311=A1
   AL312=A2
   AL321=A1
   AL322=A2
   AL331=A1
   AL332=A2
   AL341=A1
   AL342=A2
   AL351=A1
   AL352=A2
   AL361=A1
   AL362=A2
   AL371=A1
   AL372=A2
   AL381=A1
   AL382=A2
   AL391=A1
   AL392=A2
   AL401=A1
   AL402=A2
   AL411=A1
   AL412=A2
   AL421=A1
   AL422=A2
   AL431=A1
   AL432=A2
   AL441=A1
   AL442=A2
   AL451=A1
   AL452=A2
   AL461=A1
   AL462=A2
   AL471=A1
   AL472=A2
   AL481=A1
   AL482=A2
   AL491=A1
   AL492=A2
   AL501=A1
   AL502=A2
   AL511=A1
   AL512=A2
   AL521=A1
   AL522=A2
   AL531=A1
   AL532=A2
   AL541=A1
   AL542=A2
   AL551=A1
   AL552=A2
   AL561=A1
   AL562=A2
   AL571=A1
   AL572=A2
   AL581=A1
   AL582=A2

```

SUBROUTINE ELAS

74/74 OPT=1

FTN 4.5+4.14

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PAGE 2

```
      DO 8 J=1,3
      DA(1,J)=EP(I,J)
      DA(1,3)=DA(1,3)*2.
      DA(2,3)=DA(2,3)*2.
      DA(3,1)=DA(3,1)/2.
      DA(3,2)=DA(3,2)/2.
      RETURN
      END
```

```
      ELAS
      ELAS
      ELAS
      ELAS
      ELAS
      ELAS
      ELAS
      ELAS
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AD-A040 257

AIR FORCE FLIGHT DYNAMICS LAB WRIGHT-PATTERSON AFB OHIO F/G 9/2
COMPUTER PROGRAM (OPTCOMP) FOR OPTIMIZATION OF COMPOSITE STRUCT--ETC(U)
FEB 77 N S KHOT

UNCLASSIFIED

AFFDL-TR-76-149

NL

2 OF 2
AD
A040257



10/29/76 12.20.38

FIN 4.5*414

SUBROUTINE STRAIN 74/74 OPT=1

```

1  SUBROUTINE STRAIN (UV,X,Y,EX,EY,XY,L,TRI)
   DIMENSION UV(2,L), X(3), Y(3), EX(1), EY(1), EXY(1), A(3,3)
   DET=X(1)*Y(2)-Y(3)+X(2)*Y(3)-Y(1)+X(3)*Y(1)-Y(2)
   TRI=DET/2.0
5  A(1,1)=Y(2)-Y(3)
   A(2,1)=X(3)-X(2)
   A(3,1)=X(2)*Y(3)-X(3)*Y(2)
   A(1,2)=Y(3)-Y(1)
   A(2,2)=X(1)-X(3)
   A(3,2)=X(3)*Y(1)-X(1)*Y(3)
10  A(1,3)=Y(1)-Y(2)
   A(2,3)=X(2)-X(1)
   A(3,3)=Y(1)*Y(2)-X(2)*Y(1)
15  DO 1 I=1,3
     DO 1 J=1,3
       A(I,J)=A(I,J)/DET
     DO 3 K=1,L
       EX(K)=0.0
       EY(K)=0.0
       EXY(K)=0.0
       KX=0.0
       DO 2 I=1,3
         IX=I+KX
         EX(K)=EX(K)+A(1,I)*UV(IX,K)
         EY(K)=EY(K)+A(2,I)*UV(IX,K)
         EXY(K)=EXY(K)+A(3,I)*UV(IX,K)+A(1,I)*UV(IX+1,K)
         KX=KX+1
2  CONTINUE
3  RETURN
   END

```

```

1  SUBROUTINE COMP (EK,THK,TRIANG,XI,ETA,E)
   DIMENSION EK(12,12), XI(5), ETA(5), E(3,3)
   X32=XI(3)-XI(2)
   X31=XI(3)-0.0
   X21=XI(2)-0.0
   Y32=ETA(3)-0.0
   Y31=ETA(3)-0.0
   Y21=0.0
   TRIANG=ABS(XI(2))*ABS(ETA(3))/2.
   DO 1 I=1,12
     DO 1 J=1,12
       EK(I,J)=0.0
     E11=E(1,1)
     E12=E(1,2)
     E13=E(1,3)
     E22=E(2,2)
     E23=E(2,3)
     E33=E(3,3)
     EK(1,1)=E11*Y32**2-2.*E13*X32*Y32+E33*X32**2
     EK(2,1)=-E12*X32*Y32+E23*X32**2+E13*Y32**2-E33*X32*Y32+EK(2,1)
     EK(3,1)=-E11*Y31*Y32+E13*X32*Y31+E13*Y31*Y32-E33*X31*Y32+EK(3,1)
     EK(4,1)=E12*Y32*X31-E23*X32*X31-E13*Y31*Y32+E33*X32*Y31+EK(4,1)
     EK(5,1)=E11*Y21*Y32-E13*X32*Y21-E13*X21*Y32+E33*X32*X21+EK(5,1)
     EK(6,1)=-E12*X21*Y32+E23*X32*X21+E13*Y32*Y21-E33*X32*Y21+EK(6,1)
     EK(2,2)=E22*X32**2-2.*E23*X32*Y32+E33*Y32**2+EK(2,2)
     EK(3,2)=-E13*Y31*Y32+E12*X32*Y31+E13*X31*Y32-E23*X32*X31+EK(3,2)
     EK(4,2)=E23*X31*Y32-E22*X32*X31-E33*Y32*Y31+E23*X32*Y31+EK(4,2)
     EK(5,2)=E13*Y32*Y21-E12*X32*Y21-E33*Y32*X21+E23*X32*X21+EK(5,2)
     EK(6,2)=-E23*Y32*Y21+E22*X32*X21+E33*Y32*Y21-E23*X32*Y21+EK(6,2)
     EK(3,3)=-E13*X31*Y31+E11*Y31**2+E33*X31**2-E13*X31*Y31+EK(3,3)
     EK(4,3)=E23*X31**2-E12*Y31*X31-E33*X31*Y31+E13*Y31**2+EK(4,3)
     EK(5,3)=E13*Y21*Y21-E11*Y31*Y21-E33*X31*X21+E13*Y31*X21+EK(5,3)
     EK(6,3)=-E23*X31*X21+E12*Y31*X21+E33*X31*Y21-E13*Y21*Y31+EK(6,3)
     EK(4,4)=E22*X31**2-2.*E23*X31*Y31+E33*Y31**2+EK(4,4)
     EK(5,4)=E12*X31*Y21-E13*Y31*Y21-E23*X31*X21+E33*Y31*X21+EK(5,4)
     EK(6,4)=-E22*X21*X31+E23*Y31*X21+E23*X31*Y21-E33*Y31*Y21+EK(6,4)
     EK(5,5)=E11*Y21**2-2.*E13*X21*Y21+E33*X21**2+EK(5,5)
     EK(6,5)=-E12*Y21*X21+E23*X21**2+E13*Y21**2-E33*X21*Y21+EK(6,5)
     EK(6,6)=-E23*Y21*X21**2+E22*X21**2+E33*Y21**2
     DO 2 I=1,6
       IP=I+1
       DO 2 J=IP,6
         EK(I,J)=EK(J,I)
       DO 3 I=1,6
         DO 3 J=I,6
           EK(I,J)=(EK(I,J)/TRIANG)*THK*0.25
         RETURN
       END

```



```

1  SUBROUTINE ASEMBL (A,B,MA,MB,MC,MD,MM,IO,NNODES,M)
   DIMENSION A(1), B(M,M), IO(1), NA(4), NAA(3)
   IX(I,J)=I*(J-1)+1
   M2=NNODES*MM
   NA(1)=IX(M,MA)
   NA(2)=IX(M,MB)
   IF (NNODES.GE.3) NA(3)=IX(M,MC)
   IF (NNODES.GE.4) NA(4)=IX(M,MD)
   IF (NNODES.LE.3) GO TO 2
   DO 1 I=1,3
     KX=I/3
     KY=I/2
     IF (NA(KX+2).LT.NA(KY+3)) GO TO 1
     KH=NA(KX+2)
     NA(KX+2)=NA(KY+3)
     NA(KY+3)=KH
   CONTINUE
   DO 3 I=2,NNODES
     NAA(I-1)=NA(I)-NA(I-1)-MM
     KH=MM
     IAA=NA(1)
     KHH=1
     DC 7 J=1,M2
     IF (J.LE.KH) GO TO 4
     KHH=KHH+1
     IAA=NA(KHH)
     KH=KH+M1
     JX=IO(IAA)-IAA*NA(1)
     KY=MM
     DO 6 I=1,J
       IF (J.LE.KY.OR.I.LE.KY) GO TO 5
       KX=I/MM
       JX=JX+NAA(KX)
       KY=KY+M1
       A(JX)=A(JX)+3(I,J)
       JX=JX+1
       IAA=IAA+1
     RETURN
   END

```



```

1  SUBROUTINE CONDONS (EK,EKK,MA,MB,MC,MD,N0)
   DIMENSION EK(12,12), EKK(12,12)
   IF (NO.EQ.2) GO TO 6
   DO 1 I=1,12
   DO 1 J=1,12
   EKK(I,J)=0.
1  DET=EK(9,9)*EK(10,10)-EK(9,10)**2
   AX=EK(9,3)
   EK(9,9)=EK(10,10)/DET
10  EK(10,10)=AX/DET
   EK(9,10)=-EK(9,10)/DET
   EK(10,9)=EK(9,10)
   KX=0
   DO 2 I=3,10
   KX=KX+1
15  DO 2 J=1,8
   DO 2 K=3,10
2  EKK(KX,J)=EKK(KX,J)+EK(I,K)*EK(K,J)
   IF (NO.EQ.1) RETURN
   KX=0
   DO 3 I=3,10
   KX=KX+1
25  DO 3 J=1,8
   EKK(I,J)=EKK(KX,J)
   EKK(KX,J)=0
   DO 4 I=1,8
   DO 4 J=1,8
   DO 4 K=3,10
4  EKK(I,J)=EKK(I,J)+EK(I,K)*EK(K,J)
   DO 5 I=1,8
   DO 5 J=1,8
5  EKK(I,J)=EK(I,J)-EKK(I,J)
   IF (NO.EQ.3) RETURN
6  CONTINUE
   NZE=0
   NTH=3
   NFO=4
   NFI=5
   NSE=7
   NTW=12
40  IF (MC.LT.MB) CALL CHANGE (EK,NTH,NFI,NFO,NTM,NTM,NZE)
   IF (MD.LT.MB) CALL CHANGE (EK,NTH,NSE,NFO,NTM,NTM,NZE)
   IF (MD.LT.MC) CALL CHANGE (EK,NFI,NSE,NFO,NTM,NTM,NZE)
   RETURN
45  END

```

SUBROUTINE	SUM	74/74	OPT=1	FTN 4.54414	10/29/76	12.20.38	PAGE 1
1		SUBROUTINE SUM (EK,EKK,MA,MB,MC)					
		DIMENSION EK(12,12), EKK(12,12), NA(3)					
		M=2					
5		NA(1)=2*(MA-1)+1					
		NA(2)=2*(M3-1)+1					
		NA(3)=2*(MC-1)+1					
		IH=0					
		DO 4 I=1,6					
		JH=0					
10		IF (I.LE.IH) GO TO 1					
		IH=IH+1					
		IHH=IH/M					
		KX=NA(IHH)					
15	1	DO 3 J=1,6					
		IF (J.LE.JH) GO TO 2					
		JH=JH+1					
		IHH=JH/M					
		KY=NA(IHH)					
20	2	EK(KX,KY)=EK(KX,KY)+EKK(I,J)					
	3	KY=KY+1					
	4	KX=KX+1					
		RETURN					
		END					

1	SUBROUTINE CHANGE (EK,IX,IY,NND,M,L,IR)	2	CHANG
	DIMENSION EK(M,L)	3	CHANG
	KX=IX	4	CHANG
	KY=IY	5	CHANG
5	M2=2*NND	6	CHANG
	IF (IR.EQ.1) M2=L	7	CHANG
	DO 2 I=1,2	8	CHANG
	DO 1 J=1,M2	9	CHANG
	AX=EK(KX,J)	10	CHANG
10	EK(KX,J)=EK(KY,J)	11	CHANG
	EK(KY,J)=AX	12	CHANG
	KX=KX+1	13	CHANG
	KY=KY+1	14	CHANG
	IF (IR.EQ.1) RETURN	15	CHANG
15	KX=KX-2	16	CHANG
	KY=KY-2	17	CHANG
	DO 4 I=1,2	18	CHANG
	DO 3 J=1,M2	19	CHANG
	AX=EK(J,KX)	20	CHANG
20	EK(J,KX)=EK(J,KY)	21	CHANG
	EK(J,KY)=AX	22	CHANG
	KX=KX+1	23	CHANG
	KY=KY+1	24	CHANG
	RETURN	25	CHANG
25	END	26	CHANG


```

1  SUBROUTINE TRNSF (K,AA,B,C,M,NNO,M)
2  DIMENSION EK(12,12), AA(3,3), B(M,M), C(M,M)
3  M=2*NN
4  M=M*NN
5  DO 3 I=1,M2
6  JA=M
7  KA=0
8  LA=0
9  DO 3 J=1,M3
10  BI,J)=0.0
11  IF (J-J1) 2,2,1
12  JA=JA+M
13  KA=KA+2
14  LA=LA+4
15  JAA=J-IA
16  DO 3 K=1,2
17  KAA=K+K2
18  RI,J)=J(I,J)+EK(I,KAA)*AA(K,JAA)
19  DO 6 J=1,M3
20  JA=M
21  KA=0
22  LA=0
23  DO 6 I=1,M3
24  G(I,J)=0.0
25  IF (I-JA) 5,5,4
26  JA=JA+M1
27  KA=KA+2
28  LA=LA+4
29  JAA=I-IA
30  DO 6 K=1,2
31  KAA=K+K1
32  G(I,J)=C(I,J)+AA(K,JAA)*B(KAA,J)
33  RETURN
34  END
35

```



```

1  SUBROUTINE POP (MMB,JN,M1,MA,MB,MC,MD,KTYPE,IC,IO,NZ)
2  DIMENSION MA(1), MB(1), MC(1), MD(1), IC(1), IO(1), KTYPE(1)
3  IX(I,J)=1*(J-1)+1
4  NZ=0
5  NN=MM+J
6  DO 1 I=1,NN
7  IC(I)=N1
8  DO 9 L=1,MMB
9  KNODE=4
10 IF (KTYPE(L).LE.4) KNODE=KTYPE(L)
11 NNODES=2
12 ITRI=0
13 KX=IX(M1,MA(L))
14 KY=IX(M1,MB(L))
15 IF (IC(KY).LT.KX) GO TO 4
16 DO 3 I=1,MM
17 IC(KY)=KX
18 KY=KY+1
19 IF (KNODE-3) 3,5,6
20 IF (ITRI.EQ.1) GO TO 8
21 KY=IX(M1,MD(L))
22 ITRI=1
23 NNODES=3
24 GO TO 2
25 IF (ITRI.EQ.2) GO TO 8
26 IF (ITRI.EQ.1) GO TO 7
27 KY=IX(M1,MC(L))
28 ITRI=ITRI+1
29 NNODES=4
30 GO TO 2
31 KY=IX(M1,MD(L))
32 ITRI=ITRI+1
33 GO TO 2
34 GO TO 2
35 CONTINUE
36 CONTINUE
37 DO 11 I=1,NN,4M
38 IF (IC(I).LT.1) GO TO 11
39 KX=I
40 DO 10 J=1,MM
41 IC(KX)=I
42 KX=KX+1
43 CONTINUE
44 DO 12 I=1,NN
45 NZ=NZ+(1-IC(I)+1)
46 IO(I)=NZ
47 KX=(NN+NN+1)/2
48 WRITE (5,13)
49 WRITE (5,14) KX,NZ
50 WRITE (5,15)
51 WRITE (5,16) (IC(I),I=1,NN)
52 WRITE (5,17)
53 WRITE (5,16) (IO(I),I=1,NN)
54 RETURN
55 C
56 FORMAT (1H1,////20X,16HGRASS POPULATION,24X,19HAPPARENT POPULATION
57 1)
58 FORMAT (//18X,I14,18X,I22//)

```

SUBROUTINE POP	74/74	OPT=1	FTN 4.5+414	10/29/76	12.20.38	PAGE	2
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15	FORMAT (//2X,36HSTARTING ROW NUMBERS FOR EACH COLUMN///)	POP	59
16	FORMAT (5X,10I12)	POP	60
17	FORMAT (//2X,61HNUMBERS OF DIAGONAL ELEMENTS IN SINGLE ARRAYSTIFFN	POP	61
	1ESS MATRIX ///)	POP	62
	END	POP	63

60			
----	--	--	--

```

1  SUBROUTINE GAUSS (A,F,D,IG,IO,L,N,MV,NOCONF)
2  DIMENSION A(1),IC(1),IO(1),FINN(1),DINN(1)
3  IF (NOCONF.EQ.1) GO TO 5
4  DO 4 I=1,M
5  I4=1
6  DO 3 J=1,N
7  IF (IC(I).GT.1) GO TO 3
8  I4=IO(I)-J+1
9  IF (I4.EQ.0) GO TO 2
10 DO 1 K=1,I1
11 IF (IC(I).GT.K.OR.IC(I).GT.K) GO TO 1
12 K4=IO(I)-J+K
13 K2=IO(K)
14 A(I)=A(I)*A(K2)+A(K)*A(K2)*A(K2)
15 CONTINUE
16 IF (I4.EQ.0) GO TO 3
17 K2=IO(I)
18 IF (A(K2).EQ.0.) GO TO 12
19 A(I)=A(I)/A(K2)
20 CONTINUE
21 DO 8 K=1,L
22 DO 7 I=1,M
23 O(I,K)=F(I,K)
24 I4=1
25 IF (I4.EQ.0) GO TO 7
26 DO 6 J=1,I1
27 IF (IC(I).GT.J) GO TO 6
28 I4=IO(I)-J+1
29 O(I,K)=O(I,K)-A(I)*O(J,K)
30 CONTINUE
31 DO 9 K=1,L
32 DO 8 I=1,M
33 K4=IO(I)
34 O(I,K)=O(I,K)/A(K4)
35 CONTINUE
36 DO 11 I=2,N
37 I4=I-1
38 DO 10 J=1,I1
39 K4=IO(J)
40 IF (IC(J).GT.IX) GO TO 10
41 K4=IO(K4)-K+IX
42 O(I,K)=O(I,K)-A(K4)*O(K,K)
43 CONTINUE
44 GO TO 13
45 WRITE (6,14)
46 RETURN
47 C
48 FORMAT (///2X,21HSTRUCTURE IS UNSTABLE///)
49
50 END

```

```

1  SUBROUTINE BOUND (A,IB,N,NB,ID,IC)
   DIMENSION A(1), IB(1), ID(1), IC(1)
   IM=NB
   NM=N
   DO 7 JA=1,NB
     IA=IB(JA)
     IF (IA.EF.NH) GO TO 6
     KH=IA+1
     IF (IA.GT.1) GO TO 1
     KX=1
     JX=1
     GO TO 2
1    JX=ID(IA)-ID(IA-1)
     KX=ID(1)-1+1
     GO 5 1=KH,NH
2    KY=1
     IF (IC(1).LE.1A) GO TO 3
     IC(1-1)=IC(1)-1
     I=1
     KY=0
     GO TO 4
3    IC(1-1)=IC(1)
     I=1-1
     KY=IC(1)
     IC(1-1)=ID(1)-KX-KY
     GO 5 1=KX,I
     IF (1.EQ.1A) JX=JX+1
     KX=KX+KX
     KY=KY+KX
     KX=KX+1
     KY=KY+1
     IM=IM-1
     IF (IM.EQ.1)
       GO TO 7
     CONTINUE
     RETURN
   END

```



```

1  SUBROUTINE PRNTOR (A,B,X,Y,Z,IB,N,M,L,NJ,NP,NB,NBB,NN)
   DIMENSION A(NN,L), B(NN,L), X(1), Y(1), Z(1), IB(1)
   IF (NBB.EQ.0) WRITE (6,12)
   IF (NBB.NE.0) WRITE (6,13)
   NP=NP+1
   KI=1
   JJN=70/L+1
   WRITE (6,15)
   DO 11 I=1,NJ
   IF (NBB.NE.0) GO TO 1
   IF (I.NE.JJN) GO TO 1
   JJN=70/L+JJN
   WRITE (6,14)
   WRITE (6,15)
   NP=NP+1
   KH=H+1
   KHH=KH-1+1
   IF (NBB.EQ.0) GO TO 5
   ICHK=0
   DO 3 IJ=K1,NB
   IF (I3(IJ).GE.KHH.AND.IB(IJ).LE.KH) GO TO 2
   IF (I3(IJ).GT.KH) GO TO 4
   GO TO 3
   K1=K1+1
   ICHK=1
   CONTINUE
   IF (ICLK.EQ.0) GO TO 11
   CONTINUE
   IF (M.LT.3) GO TO 6
   WRITE (6,16) I,X(1),Y(1),Z(1),(A(J,1),J=KHH,KH),(B(J,1),J=KHH,KH)
   GO TO 7
   WRITE (6,17) I,X(1),Y(1),(A(J,1),J=KHH,KH),(B(J,1),J=KHH,KH)
   IF (L.EQ.1) GO TO 10
   DO 9 K=2,L
   IF (M.LT.3) GO TO 8
   WRITE (6,18) (A(J,K),J=KHH,KH),(B(J,K),J=KHH,KH)
   GO TO 9
   WRITE (6,19) (A(J,K),J=KHH,KH),(B(J,K),J=KHH,KH)
   GO TO 9
   CONTINUE
   CONTINUE
   CONTINUE
   RETURN
   C
   FORMAT (1H1,48X,23H*****DISPLACEMENTS*****IX//)
   FORMAT (1H1,48X,19H*****REACTIONS*****IX//)
   FORMAT (1H1,////)
   12  FORMAT (//1X,5HJOINT,8X,2H-X,8X,2H-Y,8X,2H-Z,8X,7HFORCE-X,7X,7HFOR
   13  1CE-Y,7X,7HFORCE-Z,8X,7HDISPL-X,10X,7HDISPL-Y,10X,7HDISPL-Z//)
   14  FORMAT (15,F14.3,F10.3,F10.3,F12.3,F14.3,F14.3,1PE17.8,1PE17.8,1PE
   15  17.8)
   16  FORMAT (15,F14.3,F10.3,F10.3,F12.3,F14.3,F14.3,1PE17.8,1PE17.8)
   17  FORMAT (39X,F12.3,F14.3,F14.3,1PE18.6,1PE17.8)
   18  FORMAT (39X,F12.3,F14.3,14X,1PE18.6,1PE17.8)
   19  END

```

SUBROUTINE PRINTQ 74/74 OPT=1 FTW 4.54414 10/29/76 12.20.38 PAGE 2

CARD NO. SEVERITY DETAILS DIAGNOSIS OF PROBLEM

24 I K1 THIS STATEMENT REDEFINES A CURRENT LOOP CONTROL VARIABLE OR PARAMETER.

10/29/76 12:20.38

FTN 4.54414

SUBROUTINE REDUCE 74474 OPT=1

```

1  SUBROUTINE REDUCE (F,IS,N,NB,L,NN)
    DIMENSION F(NN,L), IS(1)
    DO 5 J=1,L
        IM=NB
        WHEN
            1  I=IB(IH)
            2  IF (I-NH) 2,4,4
            3  NH=NH-1
            4  DO 3 K=1,NH
                KI=K+1
                F(K,J)=F(KI,J)
                IM=IM-1
                NH=NH-1
            5  IF (IM.EQ.0) GO TO 5
            GO TO 1
        CONTINUE
        RETURN
    END

```

```

1  SUBROUTINE RESTOR (D,IB,N,NB,L,NN)
   DIMENSION D(NN,L), IB(1), TOR1(10), TOR2(10)
   NH=NB
   IH=1
   1  I=IB(IH)
   IF (I.GE.NH) GO TO 7
   DO 2 K=1,L
   TOR1(K)=D(I,K)
   D(I,K)=0.
   2  J=I+1
   IF (J.GE.NH) GO TO 5
   DO 4 K=1,L
   TOR2(K)=D(J,K)
   D(J,K)=TOR1(K)
   TOR1(K)=TOR2(K)
   4  IF (I.GE.NH) GO TO 9
   I=I+1
   GO TO 3
   DO 8 K=1,L
   D(I,K)=0.
   IF (IH.NE.5) GO TO 10
   IH=IH+1
   NH=NH+1
   GO TO 1
   CONTINUE
   RETURN
   END

```



```

1  SUBROUTINE ELSTRS (EDR,A,AL,SA,E,L,ENG)
   DIMENSION EDR(12,L), A(3,3), SX(1), ENG(1)
   DO 1 K=1,L
     SX(K)=(EDR(1,K)-EDR(2,K))/AL
     SX(K)=-SX(K)
     ENG(K)=SX(K)*SX(K)*E
     SX(K)=E*SX(K)
     1 CONTINUE
   RETURN
   END
10

```

ELSTRS 2
 ELSTRS 3
 ELSTRS 4
 ELSTRS 5
 ELSTRS 6
 ELSTRS 7
 ELSTRS 8
 ELSTRS 9
 ELSTRS 10
 ELSTRS 11

```

1  SUBROUTINE SCOMP (EK,THK,QTR,X,Y,G,EC)
   DIMENSION EK(12,12), X(5), Y(5)
   QTP=(X(2)+Y(3)+X(3)+Y(4)-X(3)+Y(2)-X(4)+Y(3))*0.5
   XI=X(4)-X(2)
   EK(1,1)=XI**2/4.
   EK(2,1)=-Y(4)*XI/4.
   EK(3,1)=-X(3)*XI/4.
   EK(4,1)=XI*Y(3)/4.
   EK(2,2)=Y(4)**2/4.
   EK(3,2)=Y(4)*X(3)/4.
   EK(4,2)=-Y(4)*Y(3)/4.
   EK(3,3)=X(3)**2/4.
   EK(4,3)=-X(3)*Y(3)/4.
   EK(4,4)=Y(3)**2/4.
   DO 1 I=1,3
     K=I+1
     DO 1 J=K,4
       EK(I,J)=EK(J,I)
     DO 2 I=1,4
       L=I+4
       DO 2 J=1,4
         EK(L,J)=-EK(I,J)
       DO 3 I=1,8
         DO 3 J=1,4
           K=J+4
           EK(I,K)=-EK(I,J)
         DO 4 I=1,8
           DO 4 J=1,8
             EK(I,J)=EK(I,J)*G*THK/QTR
           EK(I,J)=EK(I,J)/EC
         CONTINUE
       RETURN
     END

```

10/29/76 12.20.38

FTN 4.5*14

```

SUBROUTINE SSRS
  74/74  OPT=1

  SUBROUTINE SSRS (UV,X,Y,THK,AREA,TA,G,L,ENG)
  DIMENSION UV(12,L), X(5), Y(5), TA(1), A(8), ENG(1)
  AREA=(X(2)*Y(3)+X(3)*Y(4)-X(4)*Y(2)-X(4)*Y(3))*0.5
  A(1)=X(4)-X(2)
  A(2)=-Y(4)
  A(3)=-X(3)
  A(4)=Y(3)
  DO 1 I=1,4
    J=I+4
    A(J)=-A(I)
  DO 2 I=1,8
    A(I)=A(1)*THK/2.
  DO 3 K=1,L
    TA(K)=0.0
  DO 3 I=1,8
    TA(K)=TA(K)+A(I)*UV(I,K)*G/(AREA*THK)
  DO 4 I=1,L
    ENG(I)=TA(I)**2/G
  CONTINUE
  RETURN
  END

```

SSRS 2
 SSRS 3
 SSRS 4
 SSRS 5
 SSRS 6
 SSRS 7
 SSRS 8
 SSRS 9
 SSRS 10
 SSRS 11
 SSRS 12
 SSRS 13
 SSRS 14
 SSRS 15
 SSRS 16
 SSRS 17
 SSRS 18
 SSRS 19
 SSRS 20
 SSRS 21
 SSRS 22

```

1      SUBROUTINE BFORCE (PRCT,AX,AY,ASX,ASY,ASXY,NZ,LOADS)
C      DIMENSIONS OF AX,AY,ASX,ASY,ASXY, CONSISTENT WITH THOSE IN
C      MEMB SUBROUTINE
5      DIMENSION PRCT(1), AX(1,10), AY(1,10), ASX(4,10,4), ASY
1(4,10,4), ASXY(4,10,4)
      I=1
      DO 1 J=1,LOADS
        AX(I,J)=1.0
        AY(I,J)=0.0
        ASX(I,J)=0.0
        ASY(I,J)=0.0
        DO 1 K=1,NZ
          AX(I,J)=PRCT(K)*ASX(I,J,K)+AX(I,J)
          AY(I,J)=PRCT(K)*ASY(I,J,K)+AY(I,J)
          ASX(I,J)=PRCT(K)*ASXY(I,J,K)+ASX(I,J)
15         CONTINUE
          RETURN
        END
2      BFORCE
3      BFORCE
4      BFORCE
5      BFORCE
6      BFORCE
7      BFORCE
8      BFORCE
9      BFORCE
10     BFORCE
11     BFORCE
12     BFORCE
13     BFORCE
14     BFORCE
15     BFORCE
16     BFORCE
17     BFORCE
18     BFORCE

```



```

1  SUBROUTINE SBUCKL (A,AX,AY,AXY,THK,LOADS,BWIDE,THK1,THK2)
   DIMENSION A(3,3), AX(1,10), AY(1,10), AXY(1,10)
   C ALL DIMENSIONS CONSISTENT WITH THOSE IN MEMB SUBROUTINE
   I=1
5  DO 1 J=1,LOADS
   AX(I,J)=AX(I,J)*(-1.*THK)
   AY(I,J)=AY(I,J)*(-1.*THK)
   AXY(I,J)=AXY(I,J)*(-1.*THK)
   TMAX=THK
10  APY=3.1415927**2
   I=1
   DO 7 J=1,LOADS
   AB=0.0
   AC=0.0
   ANX=AXY(I,J)
   BETA=A(2,2)/A(1,1)
   ALPHA=A(2,2)*A(1,2)+2.*A(3,3)/A(1,1)
   ANX=0.0
   IF (AX(I,J).LT.0..AND..AY(I,J).LT.0.) GO TO 5
   IY=0
   GAMA=0.
   IF (AY(I,J).LT.0.) GO TO 2
   IF (AX(I,J).LT.0.) GO TO 3
   GAMA=AY(I,J)/AX(I,J)
   IF (GAMA.GT.1.) IY=1
2  CONTINUE
   ANX=AX(I,J)
   IF (IY.EQ.0) GO TO 4
3  CONTINUE
   BETA=1./BETA
   ALPHA=ALPHA*A(1,1)/A(2,2)
   IF (GAMA.NE.0.) GAMA=1./GAMA
   ANX=AY(I,J)
   IY=1
35  CONTINUE
   AMU=GAMA**2-ALPHA*GAMA+BETA
   IF (AMU.LT.0.) ANX=0.0
   IF (AMU.LT.0.) GO TO 5
   AMU=-GAMA*AMU**0.5
   AKX=(AMU**2+ALPHA*AMU*BETA)/(AMU+GAMA)
   ANXCR=AKX*APY*A(1,1)/(12.*BWIDE**2)
   IF (IY.EQ.1) ANXCR=AKX*APY*A(2,2)/(12.*BWIDE**2)
   GO TO 6
5  CONTINUE
6  THETA=C.*BETA**0.5/ALPHA
   AKXY=(0.742+0.216*THETA+0.38*THETA**2)*(ALPHA*BETA/2.)**0.5
   IF (THETA.GE.1.) AKXY=(3.293+2.047/THETA)*BETA**0.75
   ANXCR=AKXY*APY*A(1,1)/(BWIDE**2.*12.)
   IF (IY.EQ.1) ANXCR=AKXY*APY*A(2,2)/(BWIDE**2.*12.)
   IF (ANX.NE.0.) AS=ANX/ANXCR
   AC=ANX/ANXCR**2
   AA=AB**2+4.*AC
   AA=SQRT(AA)
   THK1=(AA+AA)/2.
   THK2=(AA-AA)/2.
   IF (THK1.LT.THK2) THK1=THK2

```

60	7	TH3=THK1**0.33333	59
		IF (TH3.LT.TH3) TH3=TH3	60
		CONTINUE	61
	C	RETURN	62
		END	63
			64

```

1  SUBROUTINE ANORM (A,AL,MEMBS,AMAX,NC,NZ,NCC)
   DIMENSION A(1), AL(NCC,NZ)
   AMAX=0.0
   DO 7 K=1,2
   IF (NC.EQ.0) GO TO 3
   DO 2 I=1,NC
   A(I)=0.0
   DO 1 J=1,NZ
   A(I)=A(I)+AL(I,J)
   CONTINUE
   CONTINUE
   IF (K.EQ.2) RETURN
   DO 4 I=1,MEMBS
   IF (AMAX.LT.A(I)) AMAX=A(I)
   K1=NC+1
   DO 5 I=K1,MEMBS
   A(I)=A(I)/AMAX
   IF (NC.EQ.0) GO TO 7
   DO 6 I=1,NC
   DO 6 J=1,NZ
   AL(I,J)=AL(I,J)/AMAX
   CONTINUE
   CONTINUE
   RETURN
   END

```



```

1 SUBROUTINE ENGS (SSMAX,BSX,BSY,BSXY,FX,PY,PXY,EXFI,EYFI,EXYFI,ENG, ENGS
  1 EFFSTR,GADS,NZ,KH,NORTIA,NENG,NEF)
  2 COMMON /CC/ E1,E22,ANU1,ANU2,GS
  3
  4 C ALL DIMENSIONS TO BE CONSISTENT WITH THOSE IN MEMB SUBROUTINE
  5
  6 DIMENSION BSX(4,10,4), BSY(4,10,4), BSXY(4,10,4), SSMAX(5,4,4), E1
  7 11(4,4), E22(4,4), ANU1(4,4), ANU2(4,4), GSH(4,4), EXFI(4,10,4), EY
  8 2FI(4,10,4), EXYFI(4,10,4), ENG(10,4), TRI(1), EFFSTR(4,10,4), FX(4
  9 3,10,4), PY(4,10,4), PXY(4,10,4)
  10 DO 3 J=1,LO4S
  11 DO 3 K=1,NZ
  12 ENG(J,K)=0.0
  13 I=1
  14 X1=SSMAX(1,K,KH)
  15 IF (BSX(I,J,K).LT.0.) X1=SSMAX(2,K,KH)
  16 X2=SSMAX(3,K,KH)
  17 IF (BSY(I,J,K).LT.0.) X2=SSMAX(4,K,KH)
  18 X3=SSMAX(5,K,KH)
  19 PX(I,J,K)=ABS(BSX(I,J,K)/X1)
  20 PY(I,J,K)=ABS(BSY(I,J,K)/X2)
  21 PXY(I,J,K)=ABS(BSXY(I,J,K)/X3)
  22 EF1=PX(I,J,K)*2*PY(I,J,K)
  23 EF2=PX(I,J,K)*PY(I,J,K)*2*PXY(I,J,K)**2
  24 IF (NEF.EQ.1) EFFSTR(I,J,K)=EF1-EF2
  25 IF (NEF.EQ.2) EFFSTR(I,J,K)=EF1-EF2*X2/X1
  26 EF4=EF1-EF2
  27 EFN=EF1-EF2*X2/X1
  28 IF (NORTIA.EQ.2) GO TO 1
  29 Y1=(X1-ANU1(K,KH)*X2)/E11(K,KH)
  30 Y2=(X2-ANU2(K,KH)*X1)/E22(K,KH)
  31 Y3=X3/GSH(K,KH)
  32 GO TO 2
  33 CONTINUE
  34 CNU=1.-ANU1(K,KH)*ANU2(K,KH)
  35 C11=E11(K,KH)/CNU
  36 C22=E22(K,KH)/CNU
  37 Y1=X1*C11+X2*C12
  38 Y2=X1*C12+X2*C22
  39 Y3=X3*GSH(K,KH)
  40 CONTINUE
  41 EC1=X1*Y1
  42 EC2=X2*Y2
  43 EC3=X3*Y3
  44 EC2=EC1
  45 EC3=EC1
  46 E1=BSX(I,J,K)*EXFI(I,J,K)
  47 E2=BSY(I,J,K)*EYFI(I,J,K)
  48 E3=BSXY(I,J,K)*EXYFI(I,J,K)
  49 IF (NENG.EQ.1) ENG(J,K)=ENG(J,K)*(E1+E2+E3)
  50 IF (NENG.EQ.2) ENG(J,K)=ENG(J,K)*E1/EC1+E2/EC2+E3/EC3
  51 IF (NENG.EQ.3) ENG(J,K)=ENG(J,K)+EFH
  52 IF (NENG.EQ.4) ENG(J,K)=ENG(J,K)+EFN
  53 EFFSTR(I,J,K)=EFFSTR(I,J,K)+0.5
  54 CONTINUE
  55 RETURN
  56 END
  57

```

```

1  SUBROUTINE STRES (E,E1,EX,EY,EXY,SYLO,SXLO,SYLO,SXZO,SYZO,SXZY,E STRESS
    1XFI,EYFI,SXFI,SYFI,DA,ADLOS,I,IL) STRESS
2  C ALL DIMENSIONS TO BE CONSISTENT WITH THOSE IN MEMB SUBROUTINE STRESS
3  DIMENSION E(3,3), EX(10), EY(10), SXLO(4,10,4), SYLO(4,10 STRESS
4  1,4), SXZO(4,10,4), SXZO(4,10,4), SYZO(4,10,4), SXZY(4,10 STRESS
5  2I(4,10,4), SYFI(4,10,4), SXFI(4,10,4), DA(3,3), AD(3,3), EXFI(4,1 STRESS
6  30,4), EYFI(4,10,4), EXYFI(4,10,4) STRESS
7  C COMMENT I=TRIANGLE IL= LOADING CONDITION K= LAYER STRESS
8  DO 1 K=1,LDS STRESS
9  DO 1 K=1,LDS STRESS
10 C STRESS
11 SYLO(I,K,IL)=(E(1,1)*EX(K)+E(1,2)*EY(K)+E(1,3)*EXY(K))*E1 STRESS
12 SYLO(I,K,IL)=(E(2,1)*EX(K)+E(2,2)*EY(K)+E(2,3)*EXY(K))*E1 STRESS
13 SXLO(I,K,IL)=(E(3,1)*EX(K)+E(3,2)*EY(K)+E(3,3)*EXY(K))*E1 STRESS
14 SXFI(I,K,IL)=DA(1,1)*SXLO(I,K,IL)+DA(1,2)*SYLO(I,K,IL)+DA(1,3)*SXY STRESS
15 1LO(I,K,IL) STRESS
16 SYFI(I,K,IL)=DA(2,1)*SXLO(I,K,IL)+DA(2,2)*SYLO(I,K,IL)+DA(2,3)*SXY STRESS
17 1LO(I,K,IL) STRESS
18 SXFI(I,K,IL)=DA(3,1)*SXLO(I,K,IL)+DA(3,2)*SYLO(I,K,IL)+DA(3,3)*SX STRESS
19 1YLO(I,K,IL) STRESS
20 SXZO(I,K,IL)=AD(1,1)*SXLO(I,K,IL)+AD(1,2)*SYLO(I,K,IL)+AD(1,3)*SXY STRESS
21 1LO(I,K,IL) STRESS
22 SYZO(I,K,IL)=AD(2,1)*SXLO(I,K,IL)+AD(2,2)*SYLO(I,K,IL)+AD(2,3)*SXY STRESS
23 1LO(I,K,IL) STRESS
24 SXZY(I,K,IL)=AD(3,1)*SXLO(I,K,IL)+AD(3,2)*SYLO(I,K,IL)+AD(3,3)*SX STRESS
25 1YLO(I,K,IL) STRESS
26 EXFI(I,K,IL)=DA(1,1)*EX(K)+DA(1,2)*EY(K)+DA(1,3)*EXY(K)/2.0 STRESS
27 EYFI(I,K,IL)=DA(2,1)*EX(K)+DA(2,2)*EY(K)+DA(2,3)*EXY(K)/2.0 STRESS
28 EXYFI(I,K,IL)=DA(3,1)*EX(K)+2.*DA(3,2)*EY(K)+2.*DA(3,3)*EXY(K) STRESS
29 CONTINUE STRESS
30 RETURN STRESS
31 END STRESS
32

```

```

1      SUBROUTINE SHILL (EFFSTR,LOADS,BASEAE,LLUD,LCRI,LLYR,NZ,STHK)
      C EFFSTR DIMENSION CONSISTENT WITH THAT IN MEMB SUBROUTINE
      DIMENSION STHK(1)
      DIMENSION EFFSTR(4,10,4)
      LCRI=1
      AMAX=0.0
      STRSS=BASEAE
      I=1
      DO 2 J=1,LOADS
      DO 2 K=1,NZ
      IF (AMAX.GT.EFFSTR(I,J,K)) GO TO 1
      AMAX=EFFSTR(I,J,K)
      LLUD=J
      LLYR=K
      1      CONTINUE
      2      CONTINUE
      IF (STRSS.GE.AMAX) GO TO 3
      RATIO=AMAX/STRSS
      BASEAE=BASEAE*RATIO
      3      CONTINUE
      DO 6 K=1,NZ
      I=1
      DO 5 J=1,LOADS
      IF (EFFSTR(I,J,K).LT.AMAX) GO TO 4
      AMAX=EFFSTR(I,J,K)
      4      CONTINUE
      5      CONTINUE
      STHK(K)=AMAX
      6      CONTINUE
      RETURN
      END

```

```

1  SUBROUTINE AVSTRS (ABC,TRANG,QUAD,LDS,NZ)
   DIMENSION ABC(4,10,4), TRANG(4)
   DO 2 J=1,LDS
     DO 2 K=1,NZ
       AMAX=0.0
       DO 1 I=1,4
         AMAX=AMAX+ABC(I,J,K)*TRANG(I)
       CONTINUE
       AMAX=AMAX/QUAD
       ABC(1,J,K)=AMAX
     CONTINUE
   RETURN
   END
2
10
5
2
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5
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7
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9
10
11
12
13
14
AVG
AVG
AVG
AVG
AVG
AVG
AVG
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AVG
AVG

```



```

1  SUBROUTINE AEO (X,Y,Z,FR,IGNO,NN,LD,JOINTS,NB,MM)
C  DIMENSIONS OF AML,AMR,AFL,AFR,AL,AR, NEEDS CHANGE ONLY IF LOADS IN
C  MAIN IS GREATER THAN 10
5  DIMENSION X(1), Y(1), Z(1), FR(NN,LD), IGNO(1)
      DIMENSION AML(3,10), AMR(3,10), AFL(3,10), AFR(3,10), AL(3,10), AR
      (3,10)
      WRITE (6,13)
      DO 1 I=1,LD
      DO 1 J=1,MM
      AML(J,I)=0.0
      AMR(J,I)=0.0
      AFL(J,I)=0.0
      AFR(J,I)=0.0
      CONTINUE
      KI=1
      IP=0
      DO 10 I=1,JOINTS
      DO 2 I1=1,MM
      DO 2 J1=1,LD
      AL(I1,J1)=0.0
      AR(I1,J1)=0.0
      DO 8 J=1,MM
      IP=IP+1
      DO 6 K=KI,NB
      IF (IGNO(K).EQ.IP) GO TO 4
      DO 3 L=1,LD
      AL(J,L)=FR(IP,L)
      GO TO 6
      DO 5 L=1,LD
      AR(J,L)=FR(IP,L)
      KI=KI+1
      GO TO 7
      CONTINUE
      CONTINUE
      DO 9 J=1,LD
      AMR(1,J)=AMR(1,J)+Y(I)*AR(3,J)-Z(I)*AR(2,J)
      AML(1,J)=AML(1,J)+Y(I)*AL(3,J)-Z(I)*AL(2,J)
      AMR(2,J)=AMR(2,J)+Z(I)*AR(1,J)-X(I)*AR(3,J)
      AML(2,J)=AML(2,J)+Z(I)*AL(1,J)-X(I)*AL(3,J)
      AMR(3,J)=AMR(3,J)+X(I)*AR(2,J)-Y(I)*AR(1,J)
      AML(3,J)=AML(3,J)+X(I)*AL(2,J)-Y(I)*AL(1,J)
      AFR(1,J)=AFR(1,J)+AL(1,J)
      AFL(1,J)=AFL(1,J)+AR(1,J)
      AFR(2,J)=AFR(2,J)+AL(2,J)
      AFL(2,J)=AFL(2,J)+AR(2,J)
      AFR(3,J)=AFR(3,J)+AL(3,J)
      AFL(3,J)=AFL(3,J)+AR(3,J)
      CONTINUE
      CONTINUE
      WRITE (6,14)
      WRITE (6,15)
      DO 11 J=1,LD
      WRITE (6,16) J, (AML(I,J),I=1,3), (AFL(I,J),I=1,3)
      WRITE (6,17)
      WRITE (6,15)
      DO 12 J=1,LD

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```

1 SUBROUTINE ALINK (A,AL,STRENG,STNG,ELENTN,NELEM,NLINK,NZ,NG,NSKIN, ALINK
  ININT,NSTLTY,NCC)
2
3 C DIMENSION OF NELEM CONSISTENT WITH THAT IN MAIN PROGRAM
4 C ASUM,SAREA,--(NZ)
5
6 DIMENSION A(1), AL(NCC,NZ), STRENG(NCC,NZ), STNG(1), ELENTN(1), NL
7 LINK(1)
8
9 DIMENSION NELEM(5,10), ASUM(4), SAREA(4)
10 IF (NSKIN.EQ.0) GO TO 9
11 DO 8 I=1,NSKIN
12 IK=NLINK(I)
13 DO 1 IL=1,NZ
14 ASUM(IL)=0.0
15 SAREA(IL)=0.0
16 DO 3 J=1,IK
17 MEM=NELEM(I,J)
18 DO 2 IL=1,NZ
19 IF (NSTLTY.GT.0) STRENG(MEM,IL)=1.0
20 ASUM(IL)=ASUM(IL)+ELENTN(MEM)*STRENG(MEM,IL)*AL(MEM,IL)
21 IF (NSTLTY.GT.0) AL(MEM,IL)=1.0
22 SAREA(IL)=SAREA(IL)+ELENTN(MEM)*AL(MEM,IL)
23 CONTINUE
24 DO 4 IL=1,NZ
25 ASUM(IL)=ASUM(IL)/SAREA(IL)
26 DO 7 J=1,IK
27 MEM=NELEM(I,J)
28 DO 6 IL=1,NZ
29 IF (NSTLTY.GT.0) GO TO 5
30 STRENG(MEM,IL)=ASUM(IL)
31 GO TO 6
32 AL(MEM,IL)=ASUM(IL)
33 CONTINUE
34 CONTINUE
35 CONTINUE
36 CONTINUE
37 IF (NINT.EQ.0) GO TO 14
38 NP=NSKIN+1
39 DO 13 I=NP,NINT
40 IK=NLINK(I)
41 ASUM(I)=0.0
42 SAREA(I)=0.0
43 DO 10 J=1,IK
44 MEM=NELEM(I,J)
45 IP=MEM-NC
46 IF (NSTLTY.GT.0) STNG(IP)=1.0
47 ASUM(I)=ASUM(I)+ELENTN(MEM)*STNG(IP)*A(MEM)
48 IF (NSTLTY.GT.0) A(MEM)=1.0
49 SAREA(I)=SAREA(I)+ELENTN(MEM)*A(MEM)
50 CONTINUE
51 ASUM(I)=ASUM(I)/SAREA(I)
52 DO 12 J=1,IK
53 MEM=NELEM(I,J)
54 IP=MEM-NC
55 IF (NSTLTY.GT.0) GO TO 11
56 STNG(IP)=ASUM(I)
57 GO TO 12
58 A(IP)=ASUM(I)
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SUBROUTINE ALINK	74/74	OPT=1	FTN 4.5+414	10/29/76	12.20.38	PAGE	2
60	12	CONTINUE		ALINK	59		
	13	CONTINUE		ALINK	60		
	14	CONTINUE		ALINK	61		
		RETURN		ALINK	62		
		END		ALINK	63		

LOAD MAP - OPTCOMP

FMA OF THE LOAD 101
LMA+1 OF THE LOAD 75060

TRANSFER ADDRESS -- OPTCOMP 6662

ENTRY POINTS.

ENTRY	ADDRESS	PROGRAM
INPUT#	2551	OPTCOMP
TAPE#	2551	
OUTPUT#	4612	
TAPE#	4612	
OPTCOMP	6662	
MEMB	36231	
SUGD	50604	
AVECT	50773	
SURFACE	51241	
COORD	51325	
ELAS	51601	
STRAIN	52166	
COMP	52357	
ELSTIF	53026	
ASSEMBL	53202	
CONDNS	53416	
SUM	53725	
CHANGE	54044	
TRNSFM	54173	
POP	54365	
GAUSS	54710	
BOUND	55233	
PRNTDR	55373	
REDUCE	56020	
RESTOR	56121	
ELSTRS	56260	
SCOMP	56306	
SSRS	56511	
BFORCE	56663	
SBUCKL	56763	
ANORM	57271	
MULT	57435	
ENGS	57601	
STRES	60157	
SMILL	60400	
AVSTRS	60522	
AEQ	60606	
ALINK	61467	
RANDOM	62011	
LINLIM	62015	
QENTRY	62055	
FECCHR	62166	
FECPRT	62206	
FECMSK	62245	
FECFAL	62306	
QENTRY=		
COMIO=		
FECMSK=		
FLYOUT=		

LOAD MAP - OPTCOMP

ENTRY	ADDRESS	PROGRAM
FEEOV.	62330	
FEEXP.	62332	
FEORND.	62367	
FEOSCA.	62424	
FEZRO.	62510	
END.	62646	
EXIT	62673	
STOP.	62675	
ANORM.	62704	
SYSARG.	62734	
IOERR.	62753	
SYSEND.	62777	
SVP=5	63001	
CLSLNK.	63014	
SYSER.	63043	
SYSERR.	63057	
SYST1A.	63121	
SVP=1	63125	
SVP=3	63201	
SVP=4	63207	
SYS=6	63234	
SYS2=	63260	
FEORE.	63327	
FEIGND.	63421	
FEISL.	63435	
FEINUM.	63457	
FEISG.	63470	
FEPLK.	63533	
FEIBLK.	63543	
FEIST.	63557	
ERRSET	63562	
FEERR.	63572	
INPCI.	63740	
INPCR.	64021	
KOJPT.	64077	
KODMT=	64537	
KOREP.	64564	
FEOL.	64594	
FEI.	64557	
FEEXEL.	64626	
FEAPH.	64634	
FEPLS.	64641	
FECONV.	64654	
FEORIF.	64705	
FEORIO.	64711	
FEONTL.	64716	
CLOCK	64730	
TIME	64730	
DATE	64735	
JOATE	64742	
SECOND	64747	
GOTDER.	64762	
ALOG	64775	
ALOG10	65002	
ALOG.	65007	

ENTRY	ADDRESS	PROGRAM
ALOG10.	65027	
EXP	65071	EXP
EXP.	65107	
COS	65166	SINCUS=
SIN	65171	
COS.	65174	
SIN.	65204	
SYSAID=	65253	SYSAID=
FEIEXP.	65254	FLTIM=
FEIFSC.	65276	
FEONAP.	65440	FMTAP=
FECAP.	65446	
FECPMT.	65465	
FECPMU.	65470	
FECPJE.	65573	
FECLP.	65574	
FEGRP.	65617	
FECEE.	65634	
FEV.	65702	
FEBUG.	65711	
CBO.	66005	FORUTL=
BFN.	66012	
GETFIT.	66023	GETFIT=
NAME.	66060	
KRINIT.	66415	KRAKER=
OUTCI.	66505	OUTC=
OUTCR.	66603	
SORT	66663	SORT
SORT.	66702	
SYSIST.	66730	SYS=IST
MORGUE.	66732	
XTOI.	67012	XTOI=
XTOY.	67022	XTOY=
RM.CIO	67035	CIO.RM
RM.RCLA	67045	
RM.RCLP	67052	
RM.SYS	67065	
MOVE.RM	67104	MOVE.RM
MCT.RM	67171	MCT.RM
OSUB.RM	67417	OSUB.RM
OPEN.SQ	67524	OPEN.SQ
OPXX.SQ	67641	
OPEX.SQ	70006	OPEX.SQ
RLEG.RM	70033	RLEG.RM
CLSF.SQ	70104	CLSF.SQ
RPT.SQ	70172	
CLSV.SQ	70246	CLSV.SQ
REM.SQ	70376	REM.SQ
GET.SQ	70456	GET.SQ
SKGT.SQ	70527	
GONT.SQ	70556	
GXIT.SQ	70636	
GRTJ.SQ	70717	
AINI.SQ	70751	
AMBL.SQ	70756	

LOAD MAP - OPTCOMP

ENTRY	ADDRESS	PROGRAM
AMAC.SQ	70762	
OXIT.SQ	71463	
GET.Z	71510	Z.SQ
GET.S	71611	FSU.SQ
GET.U	71615	
GET.F	71617	
RMU.SQ	71623	
RMU0.SQ	71630	
RMU2.SQ	71650	
RMU1.SQ	71651	
ERR.RM	71775	ERR.RM
ERR1.RM	72001	
ERR2.RM	72157	
CHWR.SQ	72323	CHWR.SQ
PUT.SQ	72343	PUT.SQ
FLSH.SQ	73415	
WAR.SQ	73721	WAR.SQ
REPO.SQ	74120	
CLSF.RM	74201	CLSF.RM
PUT.I	74224	BRT.SQ
PUT.K	74224	
PUT.E	74224	
BTAT.SQ	74224	
PUT.C	74224	
WEOP.SQ	74340	WEOP.SQ
WEOS.SQ	74345	
SKFL.SQ	74511	SKFL.SQ
SYS	74562	SYS.RM
RCL	74575	
WNB	74601	
MSG	74611	
OPEN.RM	74623	OPEN.RM

CYBER LOADER 1.0-414

10/29/76 12.39.25.

PAGE

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[illegible]

[illegible]

TIME USED IN SECONDS = 56.1763

項目	単位	数量	金額	備考
1. 雑費	円	100	100	
2. 雑費	円	100	100	
3. 雑費	円	100	100	
4. 雑費	円	100	100	
5. 雑費	円	100	100	
6. 雑費	円	100	100	
7. 雑費	円	100	100	
8. 雑費	円	100	100	
9. 雑費	円	100	100	
10. 雑費	円	100	100	
11. 雑費	円	100	100	
12. 雑費	円	100	100	
13. 雑費	円	100	100	
14. 雑費	円	100	100	
15. 雑費	円	100	100	
16. 雑費	円	100	100	
17. 雑費	円	100	100	
18. 雑費	円	100	100	
19. 雑費	円	100	100	
20. 雑費	円	100	100	
21. 雑費	円	100	100	
22. 雑費	円	100	100	
23. 雑費	円	100	100	
24. 雑費	円	100	100	
25. 雑費	円	100	100	
26. 雑費	円	100	100	
27. 雑費	円	100	100	
28. 雑費	円	100	100	
29. 雑費	円	100	100	
30. 雑費	円	100	100	
31. 雑費	円	100	100	
32. 雑費	円	100	100	
33. 雑費	円	100	100	
34. 雑費	円	100	100	
35. 雑費	円	100	100	
36. 雑費	円	100	100	
37. 雑費	円	100	100	
38. 雑費	円	100	100	
39. 雑費	円	100	100	
40. 雑費	円	100	100	
41. 雑費	円	100	100	
42. 雑費	円	100	100	
43. 雑費	円	100	100	
44. 雑費	円	100	100	
45. 雑費	円	100	100	
46. 雑費	円	100	100	
47. 雑費	円	100	100	
48. 雑費	円	100	100	
49. 雑費	円	100	100	
50. 雑費	円	100	100	
51. 雑費	円	100	100	
52. 雑費	円	100	100	
53. 雑費	円	100	100	
54. 雑費	円	100	100	
55. 雑費	円	100	100	
56. 雑費	円	100	100	
57. 雑費	円	100	100	
58. 雑費	円	100	100	
59. 雑費	円	100	100	
60. 雑費	円	100	100	
61. 雑費	円	100	100	
62. 雑費	円	100	100	
63. 雑費	円	100	100	
64. 雑費	円	100	100	
65. 雑費	円	100	100	
66. 雑費	円	100	100	
67. 雑費	円	100	100	
68. 雑費	円	100	100	
69. 雑費	円	100	100	
70. 雑費	円	100	100	
71. 雑費	円	100	100	
72. 雑費	円	100	100	
73. 雑費	円	100	100	
74. 雑費	円	100	100	
75. 雑費	円	100	100	
76. 雑費	円	100	100	
77. 雑費	円	100	100	
78. 雑費	円	100	100	
79. 雑費	円	100	100	
80. 雑費	円	100	100	
81. 雑費	円	100	100	
82. 雑費	円	100	100	
83. 雑費	円	100	100	
84. 雑費	円	100	100	
85. 雑費	円	100	100	
86. 雑費	円	100	100	
87. 雑費	円	100	100	
88. 雑費	円	100	100	
89. 雑費	円	100	100	
90. 雑費	円	100	100	
91. 雑費	円	100	100	
92. 雑費	円	100	100	
93. 雑費	円	100	100	
94. 雑費	円	100	100	
95. 雑費				

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HEIGHT-SUM=6.740E+01    HEIGHT-BRACES=4.52000E-01    HEIGHT-POSTS=5.08243E-02    TOTAL-HEIGHT=8.253458E+01
STUOUBLE-H=1.94E-01    STUOUBLE-L=1.94E-02    STUOUBLE-R=2.1E-01    STUOUBLE-CYCLE=2.1E-01    STUOUBLE-NUM=10    EQ CYCLE NO=0    NSTABILITY=0

```

TIME USED IN SECONDS = 51.8110

[illegible]

```
WEIGHT=STN= 5.61572E+01 WEIGHT=STN= 5.78291E-02 TOTAL=WEIGHT= 7.13762E+01  
EQUATION= 1 EQUATION= 33 EQUATION= 1  
STRUCTURE NO= 1 STRUCTURE NO= 1 EQ CYCLE NO= 1 NSTBLTY= 0  
USED IN SECTIONS = 55,444;
```


[illegible][illegible][illegible]

*****TRESES*****

[illegible]

1	190E-06	2	-.2487E-07	1	1863E-08	0.	0.	1863E-08	0.
1	1	0.	0.	1	2328E-09	0.	0.	2328E-09	0.
2	MEMB	47	NODES	21	22	LENGTH	3000E+01	AREA	1000E-01
1	1492E-06	2	.6217E-08	1	1630E-08	0.	0.	1630E-08	0.
1	1	0.	0.	1	5821E-10	0.	0.	5821E-10	0.
2	MEMB	48	NODES	23	24	LENGTH	3000E+01	AREA	1000E-01
1	7461E-07	2	-.1865E-07	1	6985E-09	0.	0.	6985E-09	0.
1	1	0.	0.	1	1746E-09	0.	0.	1746E-09	0.
2	MEMB	49	NODES	25	26	LENGTH	3000E+01	AREA	1000E-01
1	2487E-07	2	.1554E-07	1	2328E-09	0.	0.	2328E-09	0.
1	1	0.	0.	1	1455E-09	0.	0.	1455E-09	0.
2	MEMB	50	NODES	27	28	LENGTH	3000E+01	AREA	1000E-01
1	4974E-07	2	-.1554E-07	1	4657E-09	0.	0.	4657E-09	0.
1	1	0.	0.	1	1455E-09	0.	0.	1455E-09	0.
2	MEMB	51	NODES	29	30	LENGTH	3000E+01	AREA	1000E-01
1	3109E-07	2	.6217E-08	1	3492E-09	0.	0.	3492E-09	0.
1	1	0.	0.	1	5821E-10	0.	0.	5821E-10	0.
2	MEMB	52	NODES	31	32	LENGTH	3000E+01	AREA	1000E-01
1	7772E-08	2	-.7772E-08	1	3492E-09	0.	0.	3492E-09	0.
1	1	0.	0.	1	8731E-10	0.	0.	8731E-10	0.
2	MEMB	53	NODES	33	34	LENGTH	3000E+01	AREA	1000E-01
1	4663E-08	2	.1166E-08	1	2910E-10	0.	0.	2910E-10	0.
1	1	0.	0.	1	1091E-10	0.	0.	1091E-10	0.
2	MEMB	54	NODES	35	36	LENGTH	3000E+01	AREA	1000E-01
1	6217E-08	2	-.1166E-08	1	8731E-10	0.	0.	8731E-10	0.
1	1	0.	0.	1	1091E-10	0.	0.	1091E-10	0.
2	MEMB	55	NODES	37	38	LENGTH	3000E+01	AREA	1000E-01

JOINT	-X	-Y	-Z	FORCE-X	FORCE-Y	FORCE-Z	DISPL-X	DISPL-Y	DISPL-Z
1	0.000	0.000	3.000						
2	0.000	0.000	6.000						
3	0.000	12.000	9.000						
4	0.000	12.000	6.000						
5	12.000	0.000	3.000						
6	12.000	0.000	6.000						
7	12.000	12.000	3.000						
8	12.000	12.000	6.000						
9	24.000	0.000	3.000						
10	24.000	0.000	6.000						
11	24.000	12.000	3.000						
12	24.000	12.000	6.000						
13	36.000	0.000	3.000						
14	36.000	0.000	6.000						
15	36.000	12.000	3.000						
16	36.000	12.000	6.000						
17	48.000	0.000	9.000						
18	48.000	0.000	6.000						
19	48.000	12.000	3.000						
20	48.000	12.000	6.000						
21	60.000	0.000	9.000						
22	60.000	0.000	6.000						
23	60.000	12.000	3.000						
24	60.000	12.000	6.000						
25	72.000	0.000	9.000						
26	72.000	0.000	6.000						
27	72.000	12.000	3.000						
28	72.000	12.000	6.000						
29	84.000	0.000	9.000						
30	84.000	0.000	6.000						
31	84.000	12.000	3.000						
32	84.000	12.000	6.000						
33	96.000	0.000	9.000						
34	96.000	0.000	6.000						
35	96.000	12.000	3.000						

[illegible]

TIME USED IN SECONDS = 110.1960

[illegible][illegible]

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[illegible][illegible]

TIME USED IN SEARCHES = 120.2433

[illegible]

[illegible]

	-X	-Y	-Z	FORCE-X	FORCE-Y	FORCE-Z	DISP-X	DISP-Y	DISP-Z
JOINT	1	0.000	0.000	9.000	0.000	0.000	0.000	0.000	0.000
	2	0.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	3	0.000	12.000	9.000	0.000	0.000	0.000	0.000	0.000
	4	0.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	5	12.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	6	12.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	7	12.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	8	12.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	9	24.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	10	24.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	11	24.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	12	24.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	13	36.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	14	36.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	15	36.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	16	36.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	17	48.000	0.000	9.000	0.000	0.000	0.000	0.000	0.000
	18	48.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	19	48.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	20	48.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	21	60.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	22	60.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	23	60.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	24	60.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	25	72.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	26	72.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	27	72.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	28	72.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	29	84.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	30	84.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	31	84.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000
	32	84.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000
	33	96.000	0.000	3.000	0.000	0.000	0.000	0.000	0.000
	34	96.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000
	35	96.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000

JOINT	-X	-Y	-Z	FORCE-X	FORCE-Y	FORCE-Z	DISPL-X	DISPL-Y	DISPL-Z
36	96.000	12.000	6.000	0.000	0.000	0.000	2.728957E-03	9.730315E-03	-1.056751E-01
37	108.000	0.000	9.000	0.000	0.000	0.000	0.000	0.000	-1.446153E-02
38	108.000	0.000	6.000	0.000	0.000	0.000	0.000	0.000	0.000
39	108.000	12.000	3.000	0.000	0.000	0.000	0.000	0.000	0.000
40	108.000	12.000	6.000	0.000	0.000	0.000	0.000	0.000	0.000
NUMBER OF LAINAE IN COMPOSITE ELEMENTS									
(1)	3	3	3	10	(3)	3	(4)	15	(5)
(6)	3	3	3	10	(3)	3	(4)	15	(5)
(11)	3	3	3	10	(3)	3	(4)	15	(5)
(16)	3	3	3	10	(3)	3	(4)	15	(5)
TOTAL THICKNESS OF ELEMENTS									
1	1	2	3	4	5	6	7	8	9
0	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
1	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
2	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
3	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
4	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
5	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
TIME USED IN SECONDS = 131.6260									
RELATIVE AREAS OF MEMBERS									
1	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
2	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
3	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
4	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
5	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
6	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
7	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
8	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
9	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
10	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
11	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
12	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
13	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
14	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
15	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
16	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
17	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
18	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
19	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
20	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
21	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
22	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
23	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
24	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
25	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
26	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
27	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
28	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
29	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
30	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
31	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
32	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
33	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
34	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
35	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
36	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
37	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
38	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
39	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000
40	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000	0.000

*****REACTIONS*****

JOINT	-X	-Y	-Z	FORCE-X	FORCE-Y	FORCE-Z	DISPL-X	DISPL-Y	DISPL-Z
37	103.000	0.000	9.000	72000.000	-2925.000	1000.000	0.000	0.000	0.000
38	108.000	0.000	5.000	-72000.000	14793.000	1000.000	0.000	0.000	0.000
39	108.000	12.000	9.000	72000.000	2925.000	1000.000	0.000	0.000	0.000
40	103.000	12.000	5.000	-72000.000	-14793.000	1000.000	0.000	0.000	0.000

SUMMATION OF FORCES

APPLIED LOADS

LOAD COND	4X	MY	MZ	FX	FY	FZ
1	-.240000E+05	.113062E-04	-.144342E-05	.226279E-06	.486921E-08	-.400000E+04
2	.600000E+04	-.199336E-07	-.251407E-06	-.250387E-07	-.100000E+04	-.339991E-08

REACTIONS

LOAD COND	4X	MY	MZ	FX	FY	FZ
1	.240000E+05	-.134669E-04	.131130E-05	-.137836E-06	-.383709E-08	.400000E+04
2	-.600000E+04	-.451631E-07	-.200467E-06	.267755E-07	.100000E+04	.598811E-08

TIME USED IN SECONDS = 133.1600

 X1NN0A4 /// END OF LIST ///
 X1NN0A4 /// END OF LIST ///

